

(12) **United States Patent**
Yehle

(10) **Patent No.:** **US 9,494,379 B2**
(45) **Date of Patent:** ***Nov. 15, 2016**

(54) **CROSSBOW**

(56) **References Cited**

(71) Applicant: **Ravin Crossbows, LLC**, Superior, WI (US)

U.S. PATENT DOCUMENTS

(72) Inventor: **Craig Thomas Yehle**, Holmen, WI (US)

4,030,473 A	6/1977	Puryear
4,192,281 A	3/1980	King
4,287,867 A	9/1981	Islas
4,479,480 A	10/1984	Holt
4,545,358 A	10/1985	Collins
4,565,182 A	1/1986	Barnett
4,587,944 A	5/1986	Barnett
4,693,228 A	9/1987	Simonds et al.
4,719,897 A	1/1988	Gaudreau

(73) Assignee: **Ravin Crossbows, LLC**, Superior, WI (US)

(*) Notice: Subject to any disclaimer, the term of this patent is extended or adjusted under 35 U.S.C. 154(b) by 0 days.

(Continued)

This patent is subject to a terminal disclaimer.

OTHER PUBLICATIONS

(21) Appl. No.: **15/098,537**

U.S. Appl. No. 61/820,792, filed May 8, 2013, Cocking Mechanism for a Bow.

(22) Filed: **Apr. 14, 2016**

(Continued)

(65) **Prior Publication Data**

Primary Examiner — Melba Bumgarner

Assistant Examiner — Amir Klayman

US 2016/0223285 A1 Aug. 4, 2016

Related U.S. Application Data

(57) **ABSTRACT**

(63) Continuation-in-part of application No. 14/107,058, filed on Dec. 16, 2013, now Pat. No. 9,354,015.

A crossbow including first and second cams are mounted to first and second bow limbs, respectively. Each of the cams includes a string guide journal, an upper power cable take-up journal, and a lower power cable take-up journal. The draw string is arranged in a reverse draw configuration. The upper and lower power cables are received in each of the respective upper and lower power cable take-up journals and are displaced away from the respective draw string journals as the bow is drawn from the released configuration to the drawn configuration. In one embodiment, the upper and lower power cable take-up journals are helical in configuration. As a result of this configuration the cams can rotate more than 270 degrees, and preferably more than 300 degrees, as the crossbow is drawn from the released configuration to the drawn configuration.

(60) Provisional application No. 62/244,932, filed on Oct. 22, 2015.

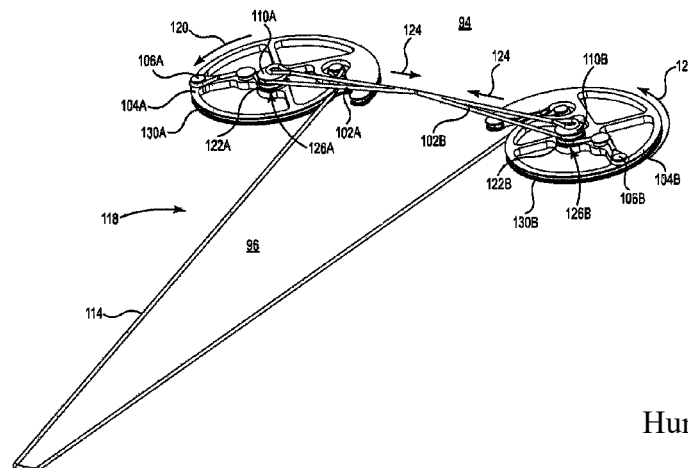
(51) **Int. Cl.**
F41B 5/10 (2006.01)
F41B 5/12 (2006.01)

(52) **U.S. Cl.**
CPC **F41B 5/105** (2013.01); **F41B 5/10** (2013.01); **F41B 5/12** (2013.01)

(58) **Field of Classification Search**
CPC F41B 5/10; F41B 5/12; F41B 5/00; F41B 5/1484; F41B 5/123; F41B 5/1403; F41B 5/105

See application file for complete search history.

19 Claims, 50 Drawing Sheets



Hunter's Manufacturing Company, Inc.
Ex. 1006

(56)

References Cited

U.S. PATENT DOCUMENTS

4,766,874 A 8/1988 Nishioka
4,877,008 A 10/1989 Troubridge
4,942,861 A 7/1990 Bozek
5,085,200 A 2/1992 Horton-Corcoran et al.
5,220,906 A 6/1993 Choma
D337,145 S 7/1993 Horton-Corcoran
5,243,956 A 9/1993 Luehring
5,265,584 A 11/1993 Judson et al.
5,368,006 A 11/1994 McPherson
5,388,564 A 2/1995 Islas
5,445,139 A 8/1995 Bybee
5,522,373 A 6/1996 Barnett
5,596,976 A 1/1997 Waiser
5,598,829 A 2/1997 Bednar
5,630,405 A 5/1997 Nizov
5,649,520 A 7/1997 Bednar
5,671,723 A 9/1997 Goff et al.
5,884,614 A 3/1999 Darlington et al.
5,934,265 A 8/1999 Darlington
6,073,351 A 6/2000 Barnett
6,095,128 A 8/2000 Bednar
6,155,243 A 12/2000 Gallops
6,205,990 B1 3/2001 Adkins
6,286,496 B1 9/2001 Bednar
6,415,780 B1 7/2002 Proctor
6,425,386 B1 7/2002 Adkins
6,460,528 B1 10/2002 Gallops, Jr.
6,571,785 B1 6/2003 Choma
6,651,641 B1 11/2003 Bower et al.
6,705,304 B1 3/2004 Pauluhn
6,712,057 B2 3/2004 Andrews
6,736,123 B1 5/2004 Summers et al.
6,776,148 B1 8/2004 Islas
6,786,214 B2 9/2004 Andrews
6,799,566 B1 10/2004 Malucelli
6,901,921 B1 6/2005 Barnett
6,913,007 B2 7/2005 Bednar
7,174,884 B2 2/2007 Kempf et al.
7,204,242 B2 4/2007 Dziekan
7,305,979 B1 12/2007 Yehle
7,328,693 B2 2/2008 Kempf
7,363,921 B2 4/2008 Kempf
D589,578 S 3/2009 Choma
D590,907 S 4/2009 Barnett
7,624,724 B2 12/2009 Bednar et al.
7,624,725 B1 12/2009 Choma
7,677,233 B2 3/2010 Bednar
7,708,001 B2 5/2010 Kempf
7,743,760 B2 6/2010 Woodland
7,748,370 B1 7/2010 Choma
7,770,567 B1 8/2010 Yehle
7,784,453 B1 8/2010 Yehle

7,810,480 B2 10/2010 Shepley et al.
7,832,386 B2 11/2010 Bednar et al.
7,836,871 B2 11/2010 Kempf
7,997,258 B2 8/2011 Shepley et al.
8,016,703 B1 9/2011 Kronengold et al.
8,020,543 B2 9/2011 Maleski et al.
8,033,275 B2 10/2011 Bednar et al.
8,042,530 B2 10/2011 Barnett
8,091,540 B2 1/2012 Matasic et al.
8,104,461 B2 1/2012 Kempf
8,191,541 B2 6/2012 Shaffer et al.
8,240,299 B2 8/2012 Kronengold et al.
8,439,025 B2 5/2013 Shaffer et al.
8,469,012 B2 6/2013 Bednar et al.
8,479,719 B2 7/2013 Bednar et al.
8,651,095 B2 2/2014 Islas
8,671,923 B2 3/2014 Goff et al.
8,763,595 B1 7/2014 Bednar et al.
8,794,225 B2 8/2014 Bednar et al.
8,899,217 B2 12/2014 Islas
9,255,753 B2 2/2016 Pulkrabek et al.
9,354,015 B2 5/2016 Yehle
2005/0022799 A1 2/2005 Bednar
2007/0028907 A1 2/2007 Bednar et al.
2009/0178657 A1 7/2009 Shaffer
2009/0194086 A1 8/2009 Kempf
2010/0012108 A1 1/2010 Bednar et al.
2010/0031945 A1 2/2010 Shaffer et al.
2010/0154768 A1 6/2010 Bednar et al.
2010/0170487 A1 7/2010 Kronengold et al.
2010/0170488 A1 7/2010 Rasor et al.
2010/0186728 A1 7/2010 Bednar et al.
2010/0269807 A1 10/2010 Kempf
2011/0203561 A1 8/2011 Shaffer et al.
2011/0232619 A1 9/2011 Bednar et al.
2011/0253118 A1 10/2011 Kempf
2011/0308508 A1 12/2011 Islas
2012/0006311 A1 1/2012 Bednar et al.
2012/0080021 A1 4/2012 Shaffer et al.

OTHER PUBLICATIONS

U.S. Appl. No. 14/071,723, filed Nov. 5, 2013, De-Cocking Mechanism for a Bow.
U.S. Appl. No. 15/171,391, filed Jun. 2, 2016, Cocking Mechanism for a Crossbow.
U.S. Appl. No. 62/224,932, filed Oct. 22, 2015, String Guide for a Bow.
U.S. Appl. No. 15/098,537, filed Apr. 14, 2016, Crossbow.
U.S. Appl. No. 15/098,557, filed Apr. 14, 2016, String Control System for a Crossbow.
U.S. Appl. No. 15/098,568, filed Apr. 14, 2016, Reduced Friction Trigger for a Crossbow.
U.S. Appl. No. 15/098,577, filed Apr. 14, 2016, Anti-Dry Fire System for a Crossbow.

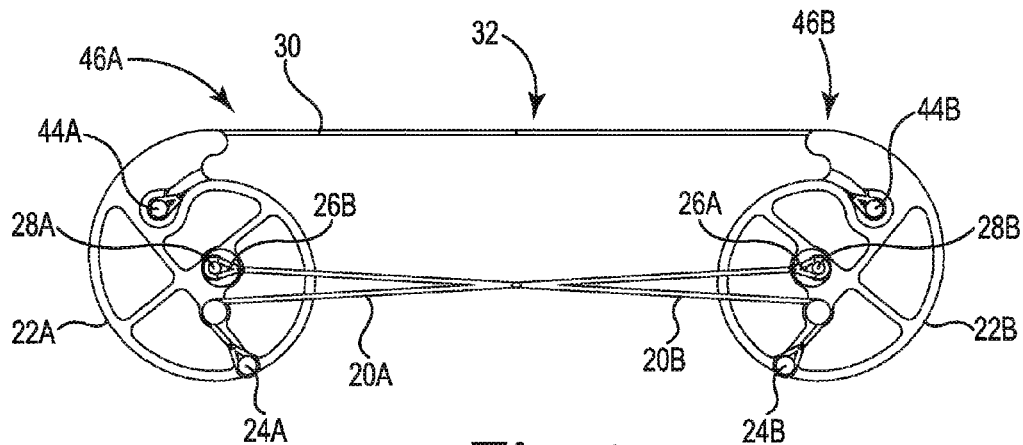


Fig. 1
PRIOR ART

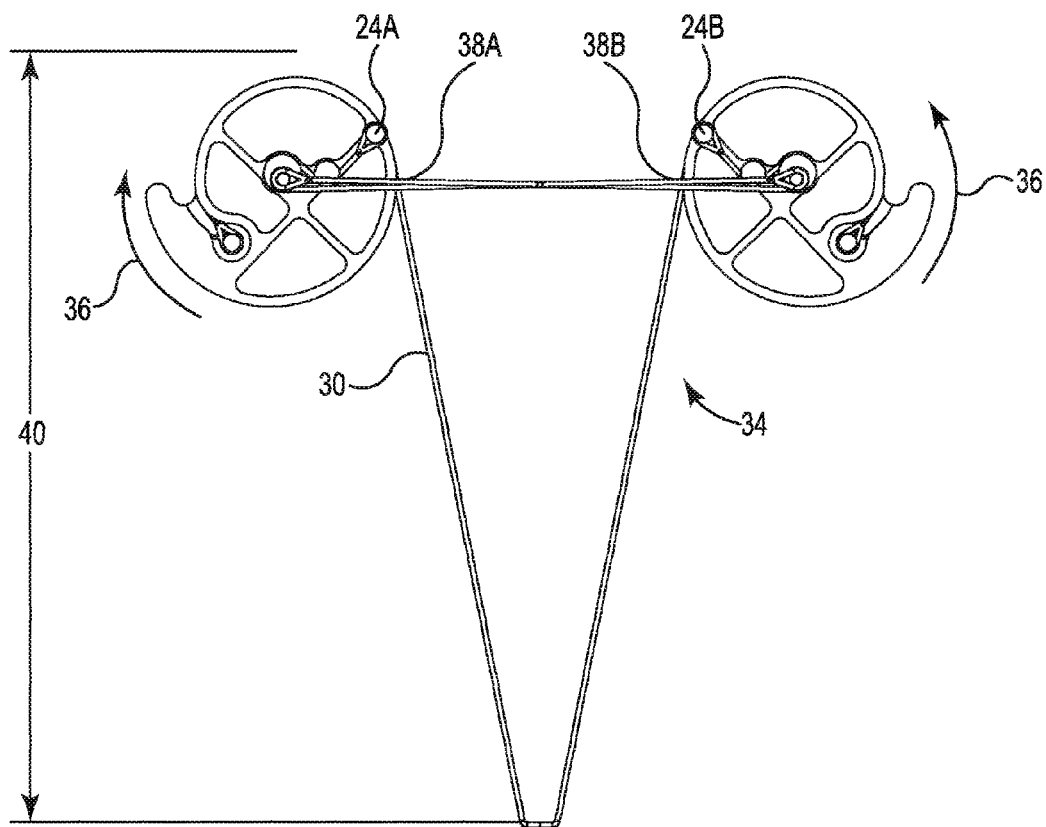


Fig. 2
PRIOR ART

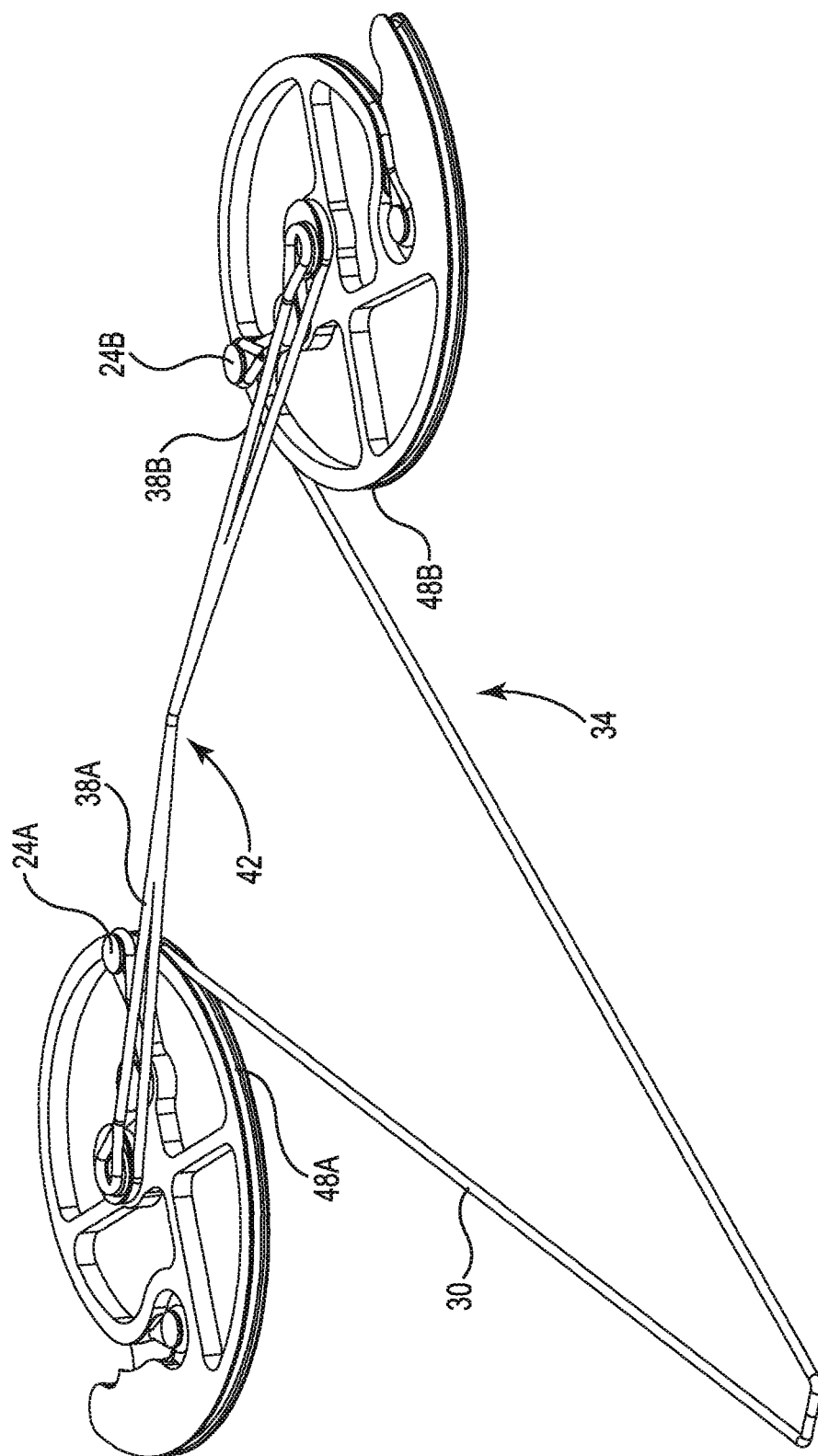
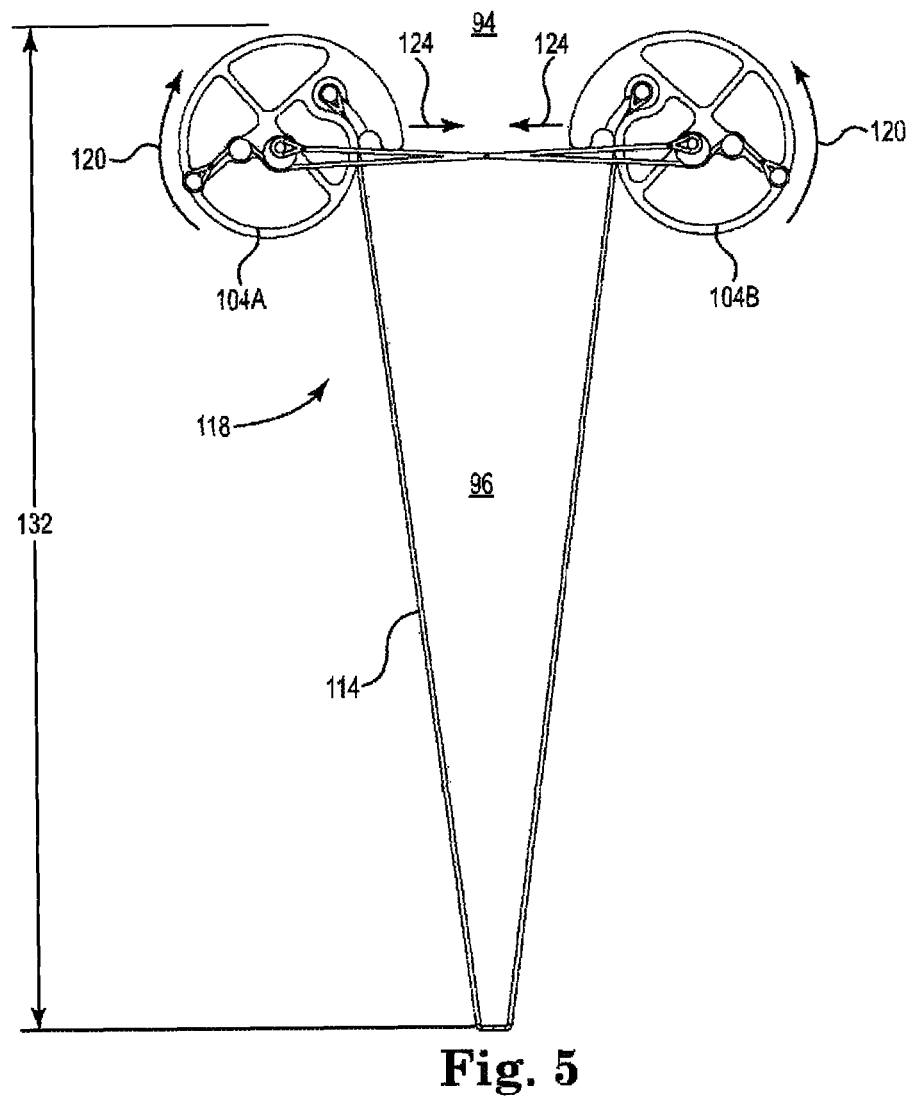
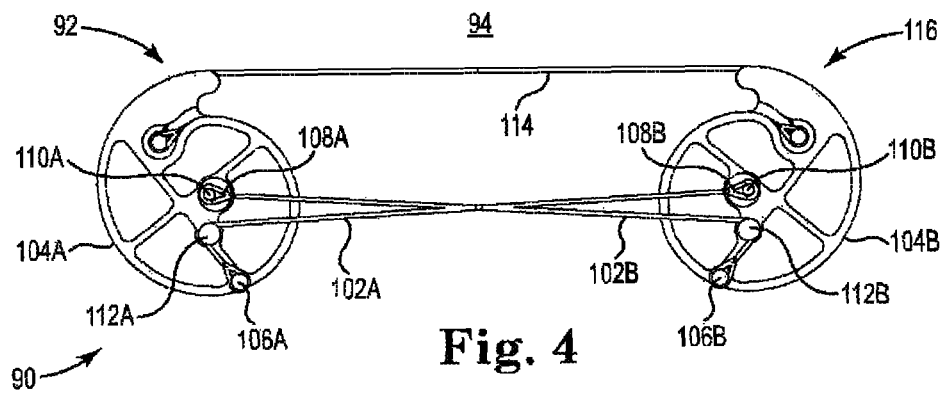


Fig. 3
PRIOR ART



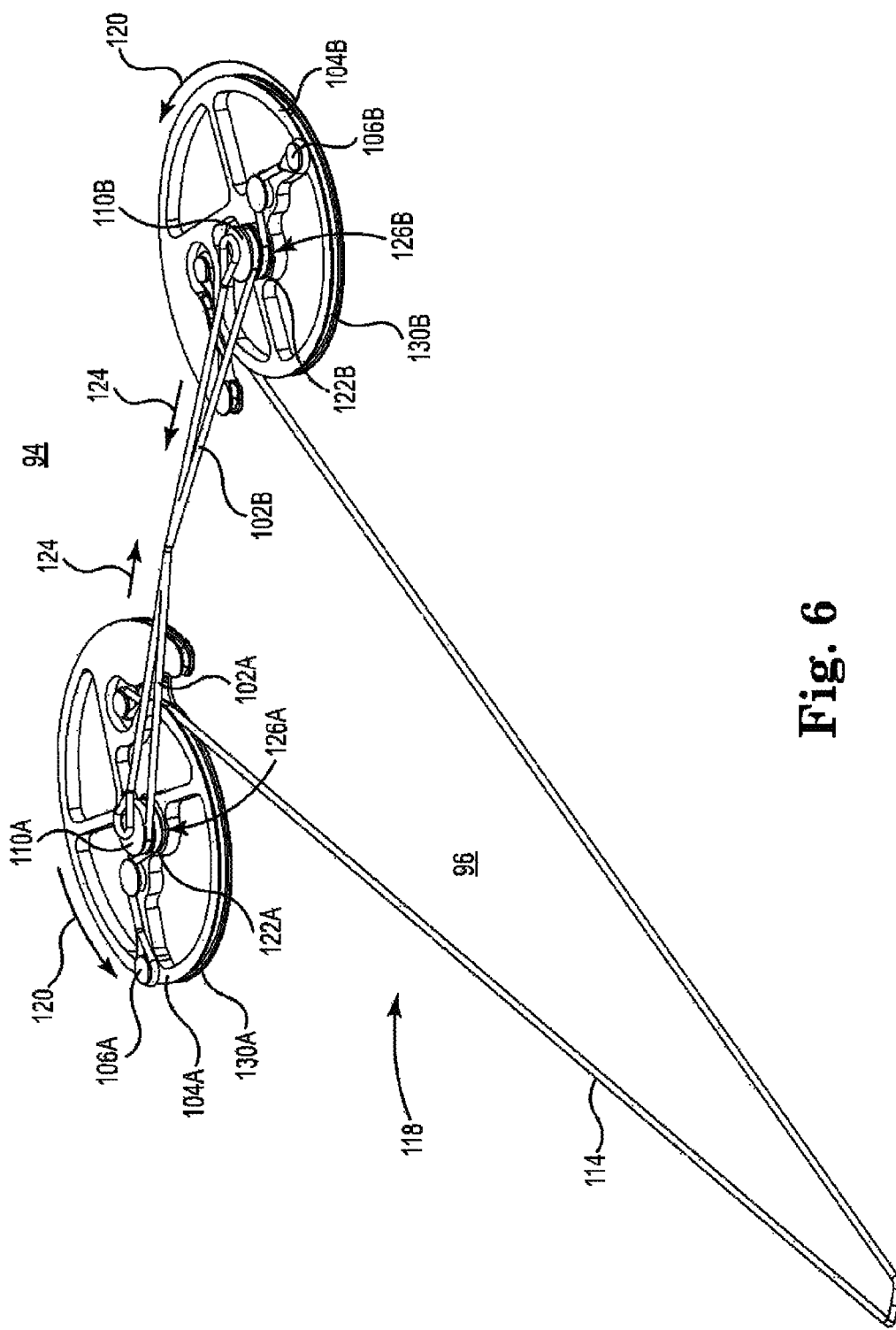


Fig. 6

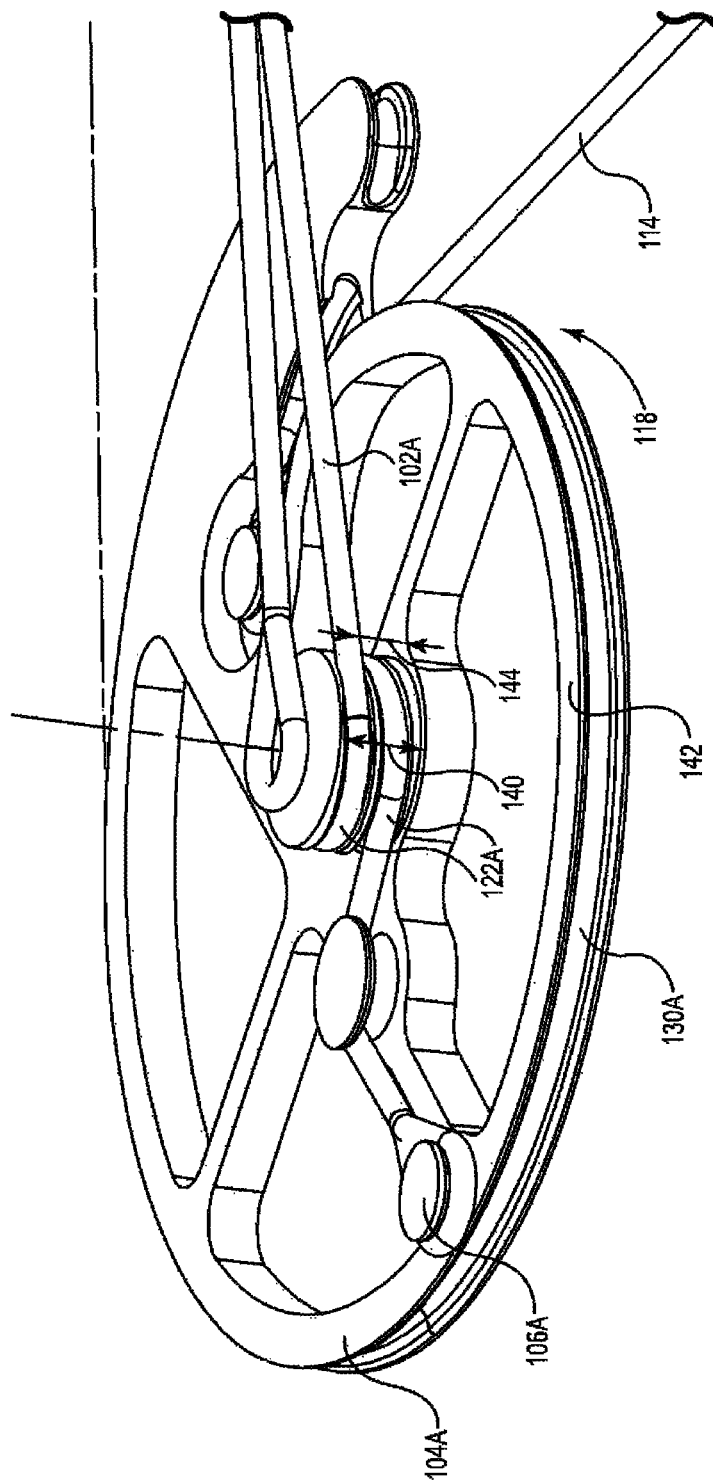


Fig. 7

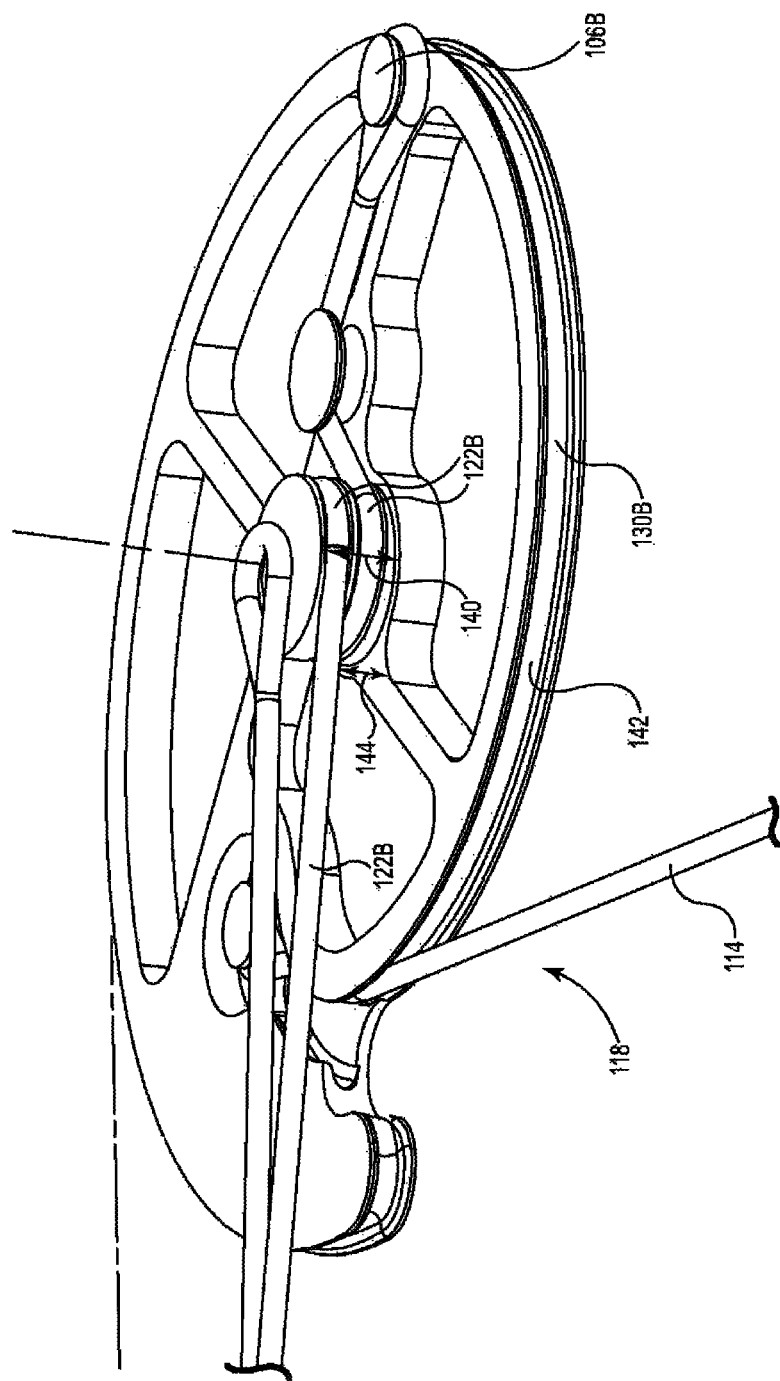


Fig. 8

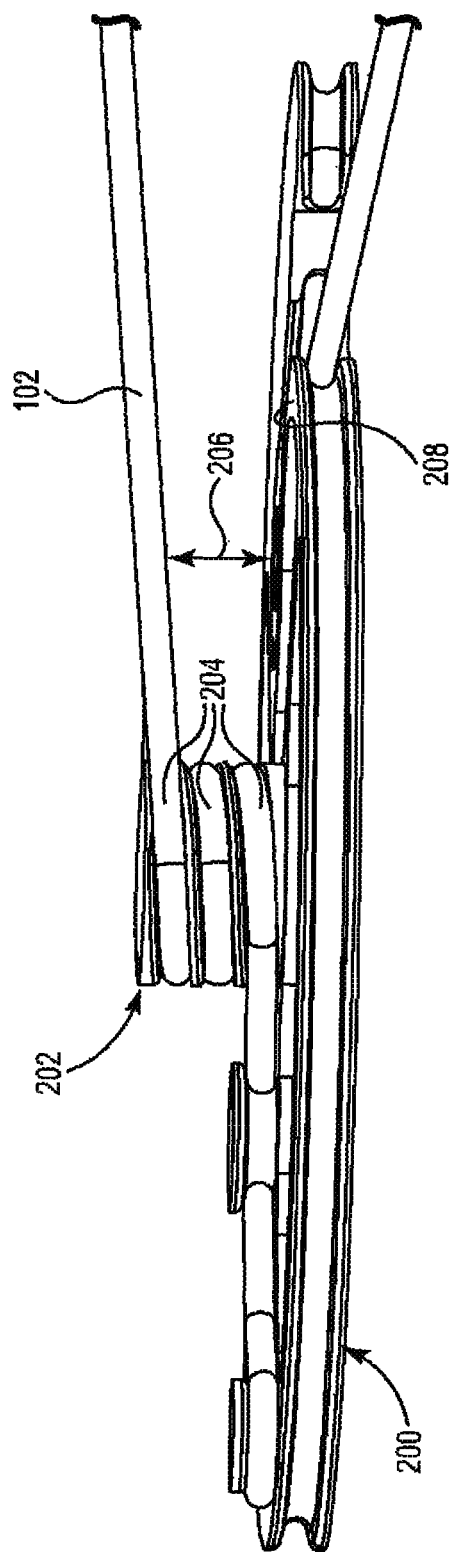


Fig. 9A

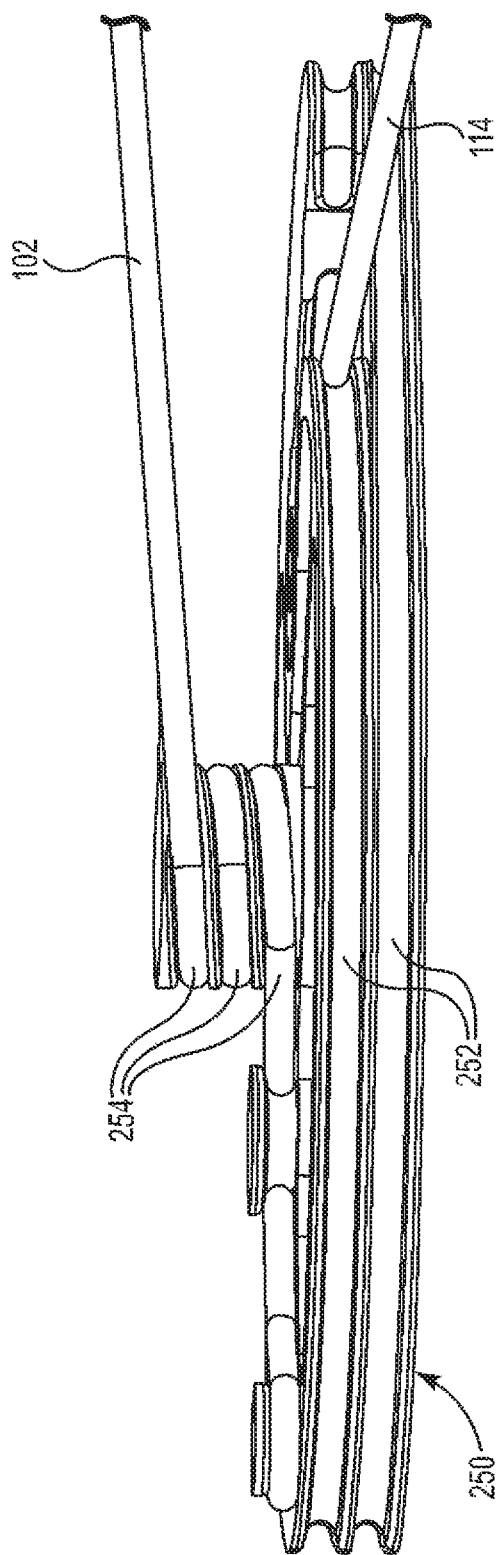


Fig. 9B

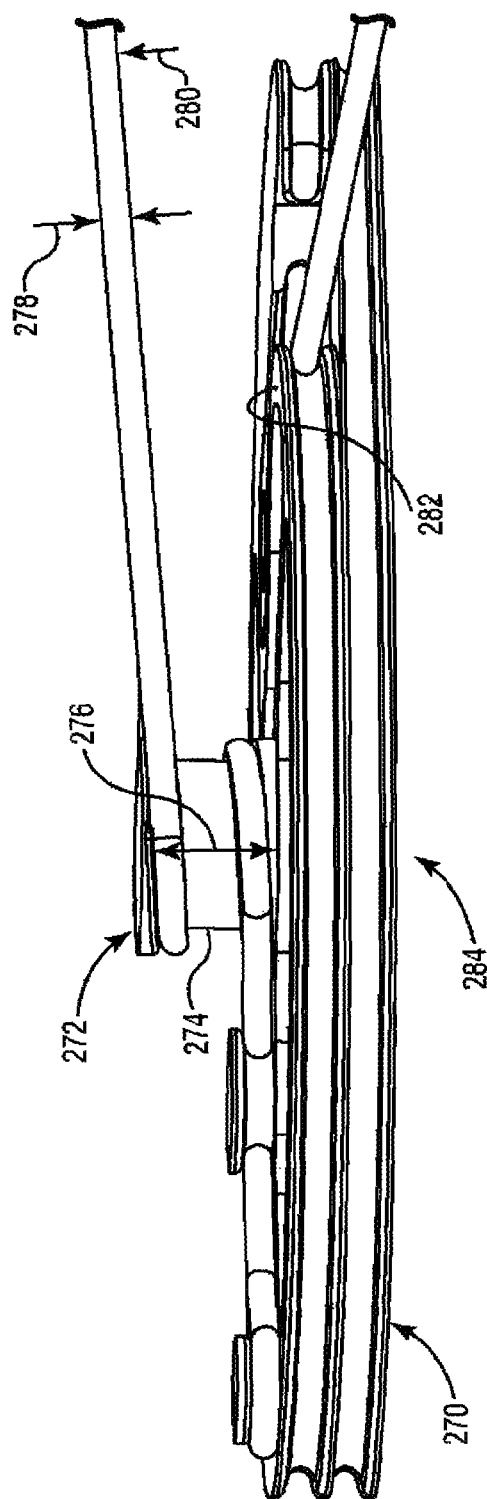
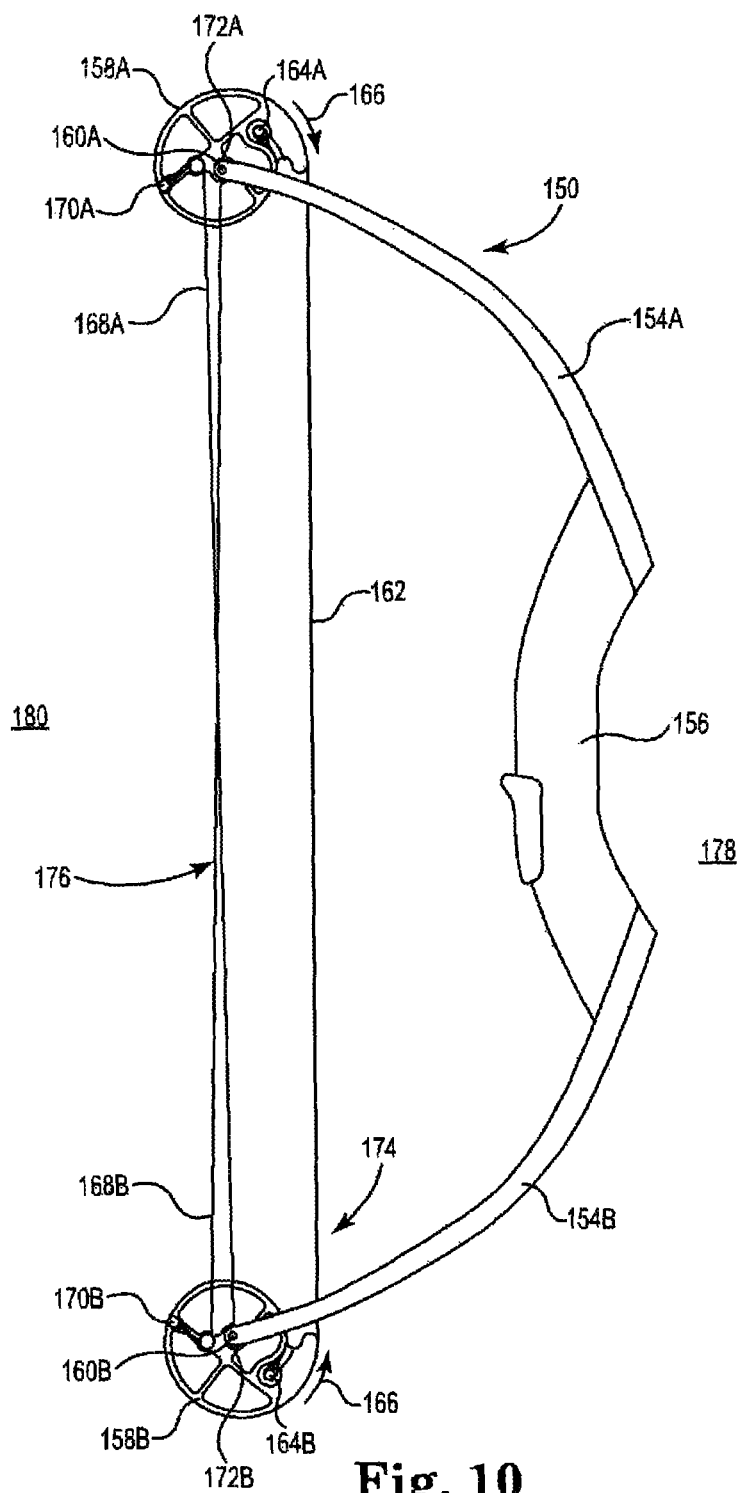


Fig. 9C



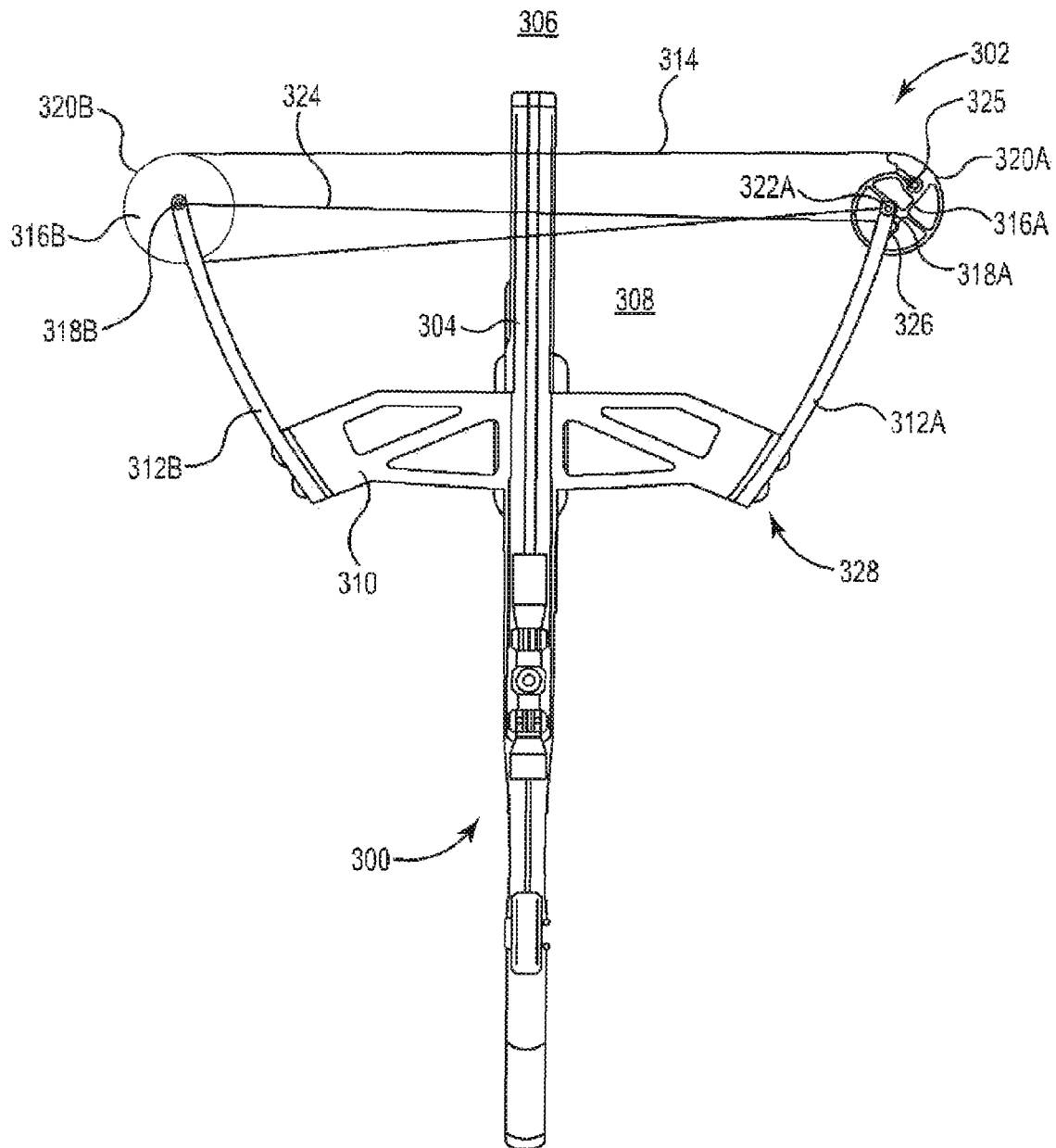


Fig. 11

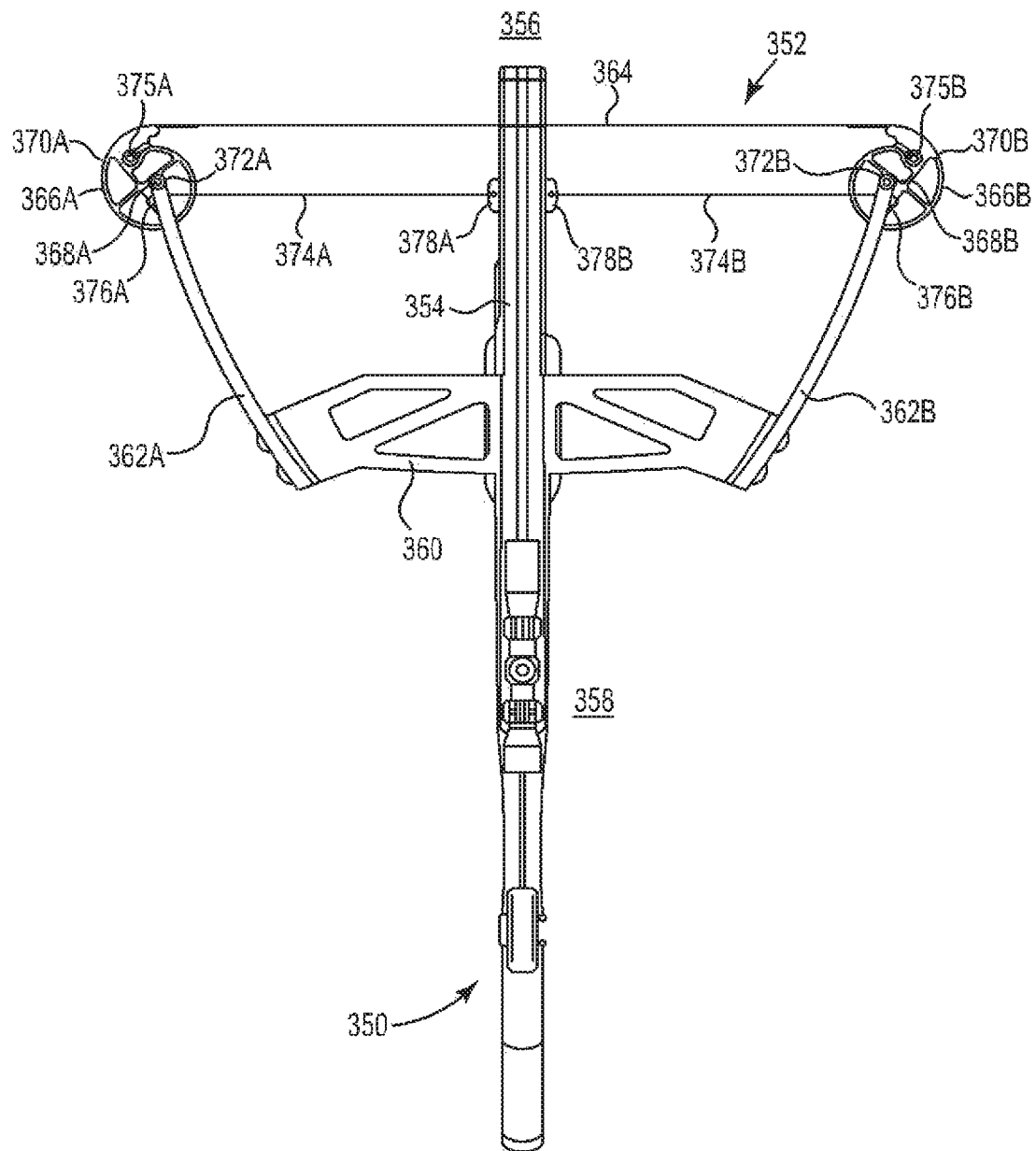


Fig. 12

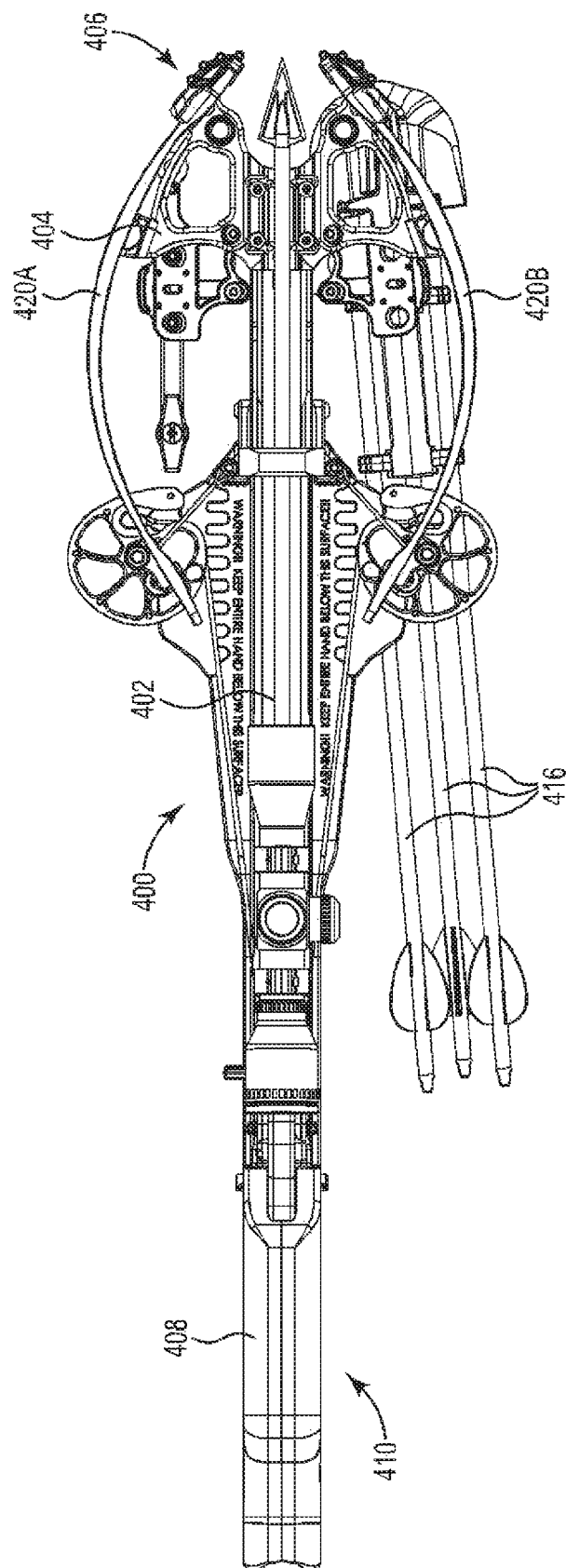
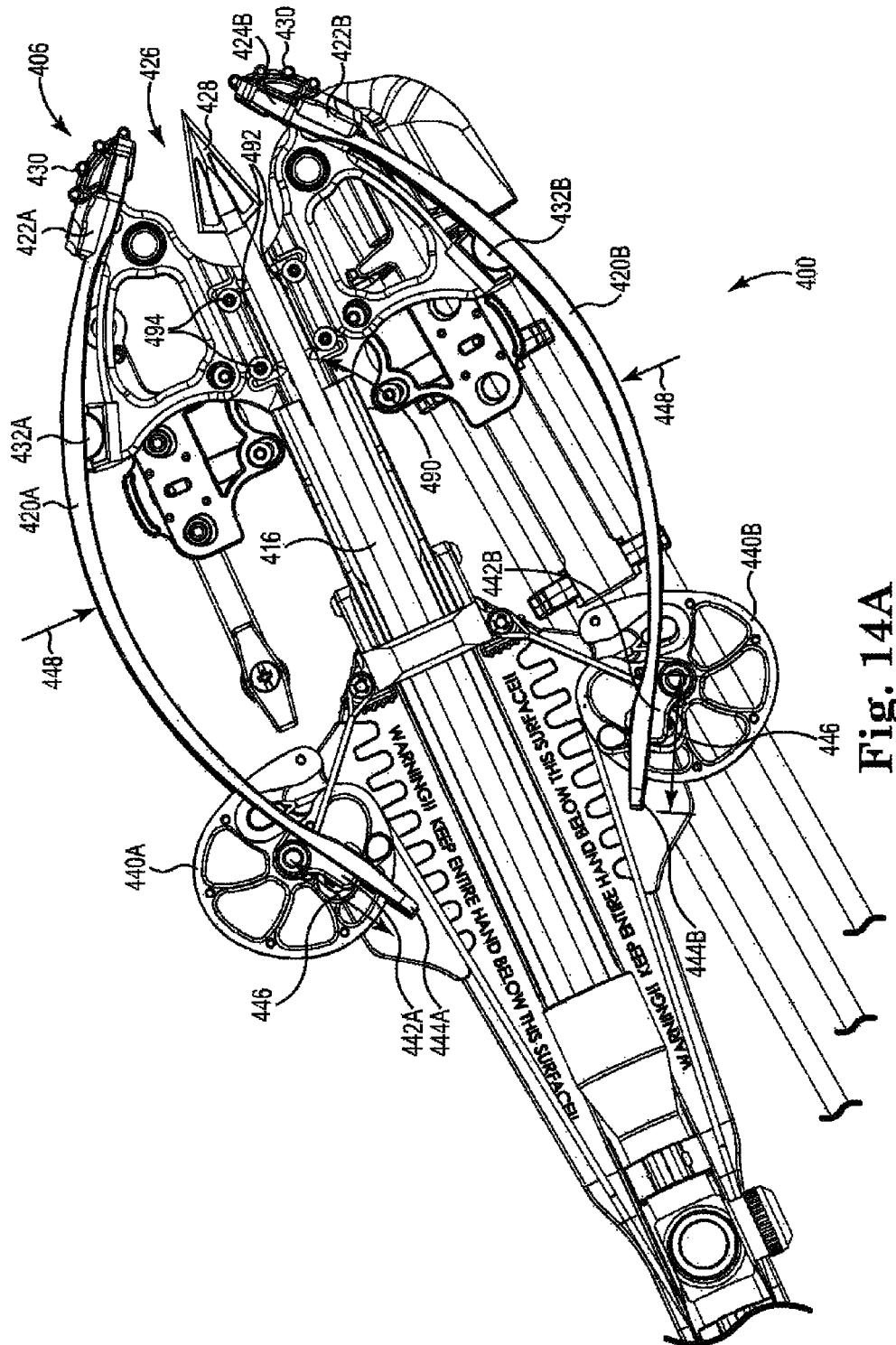


Fig. 13A



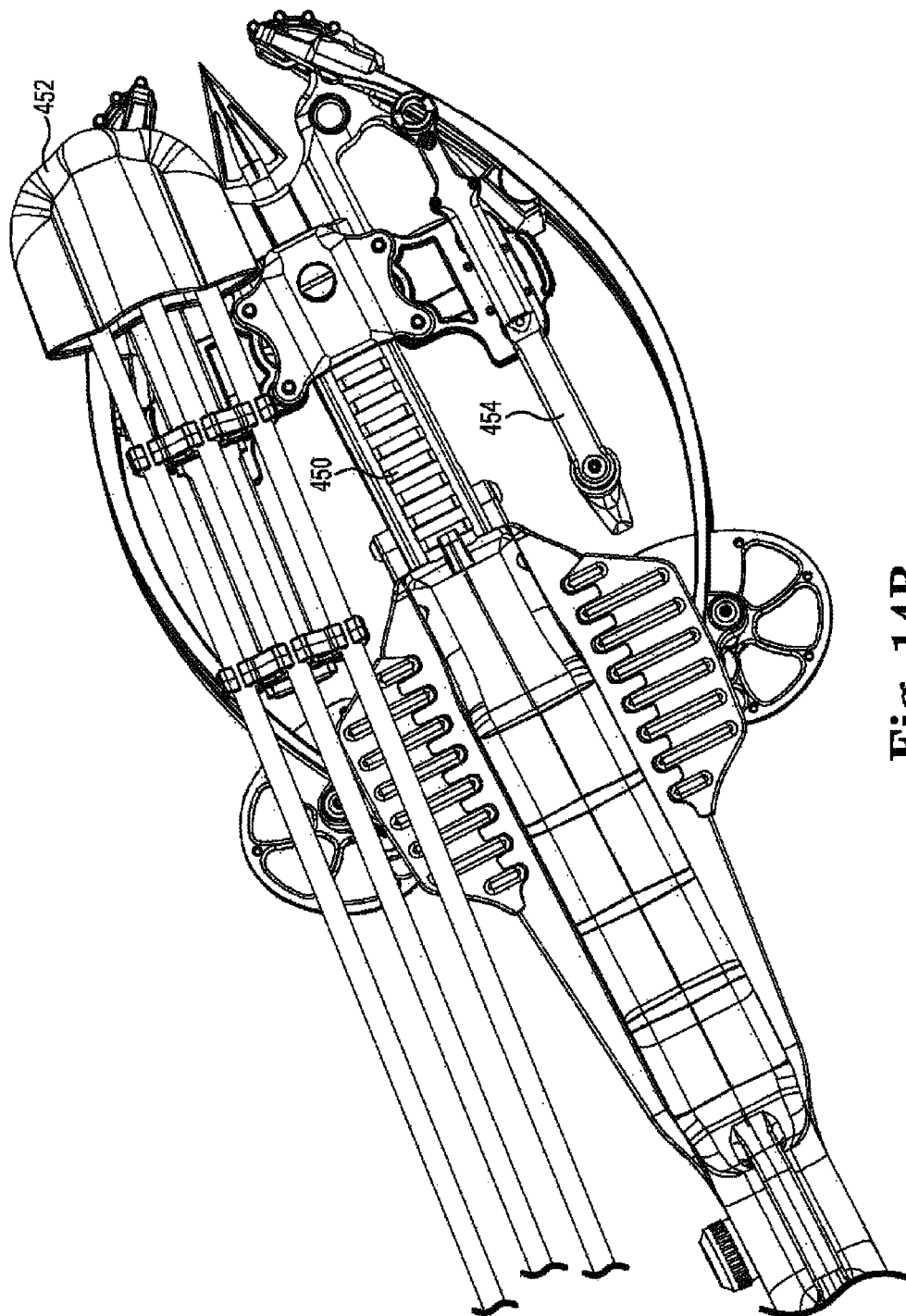


Fig. 14B

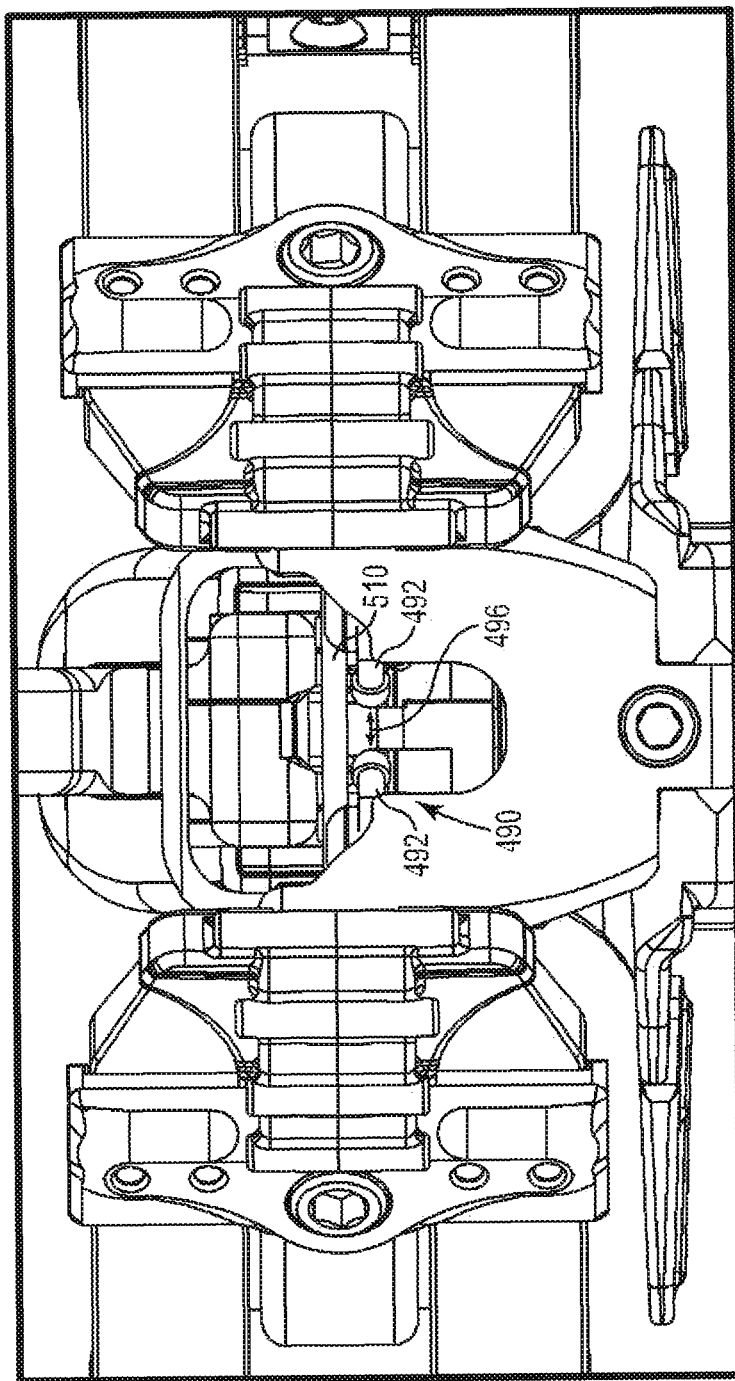


Fig. 14C

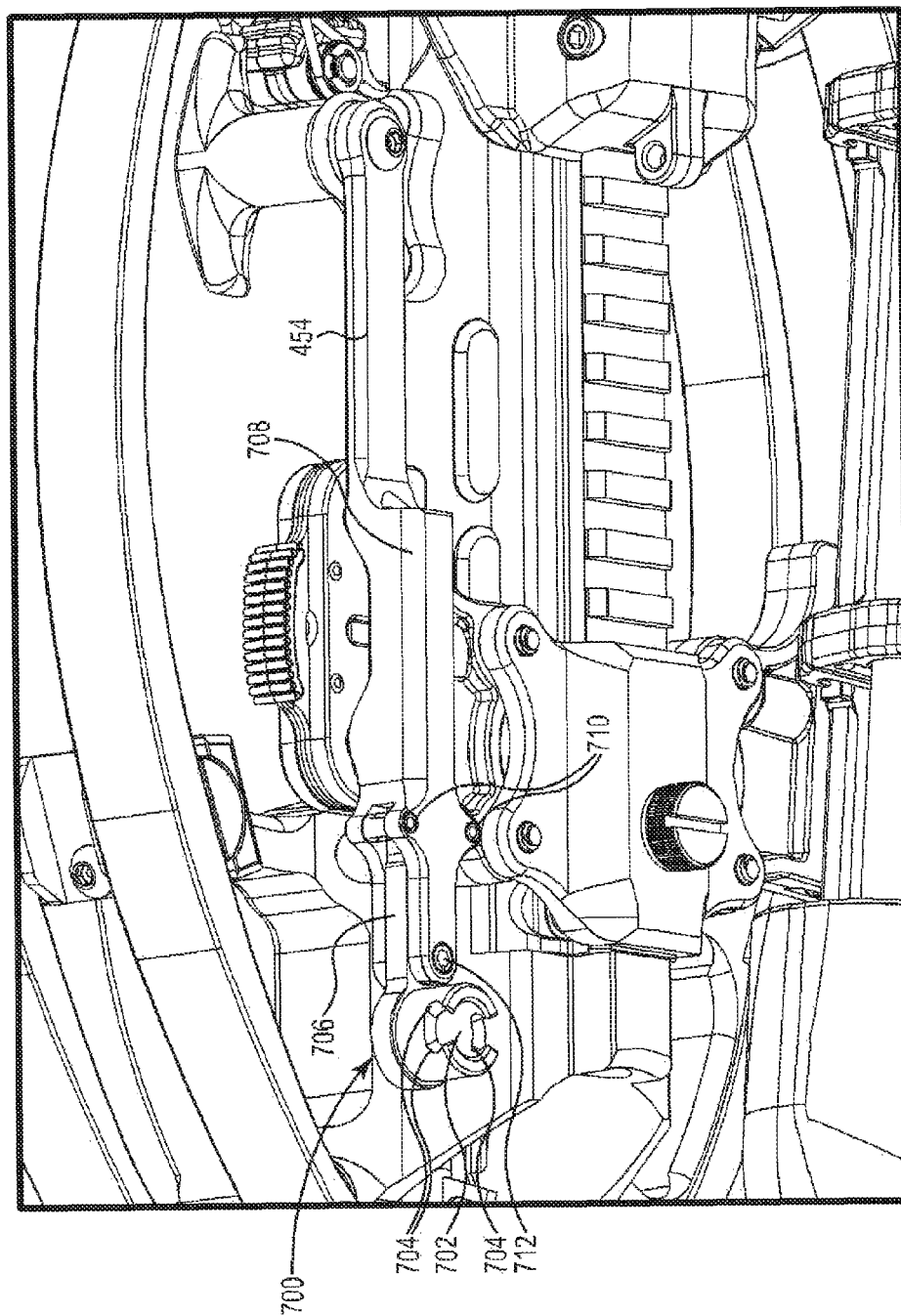


Fig. 14D

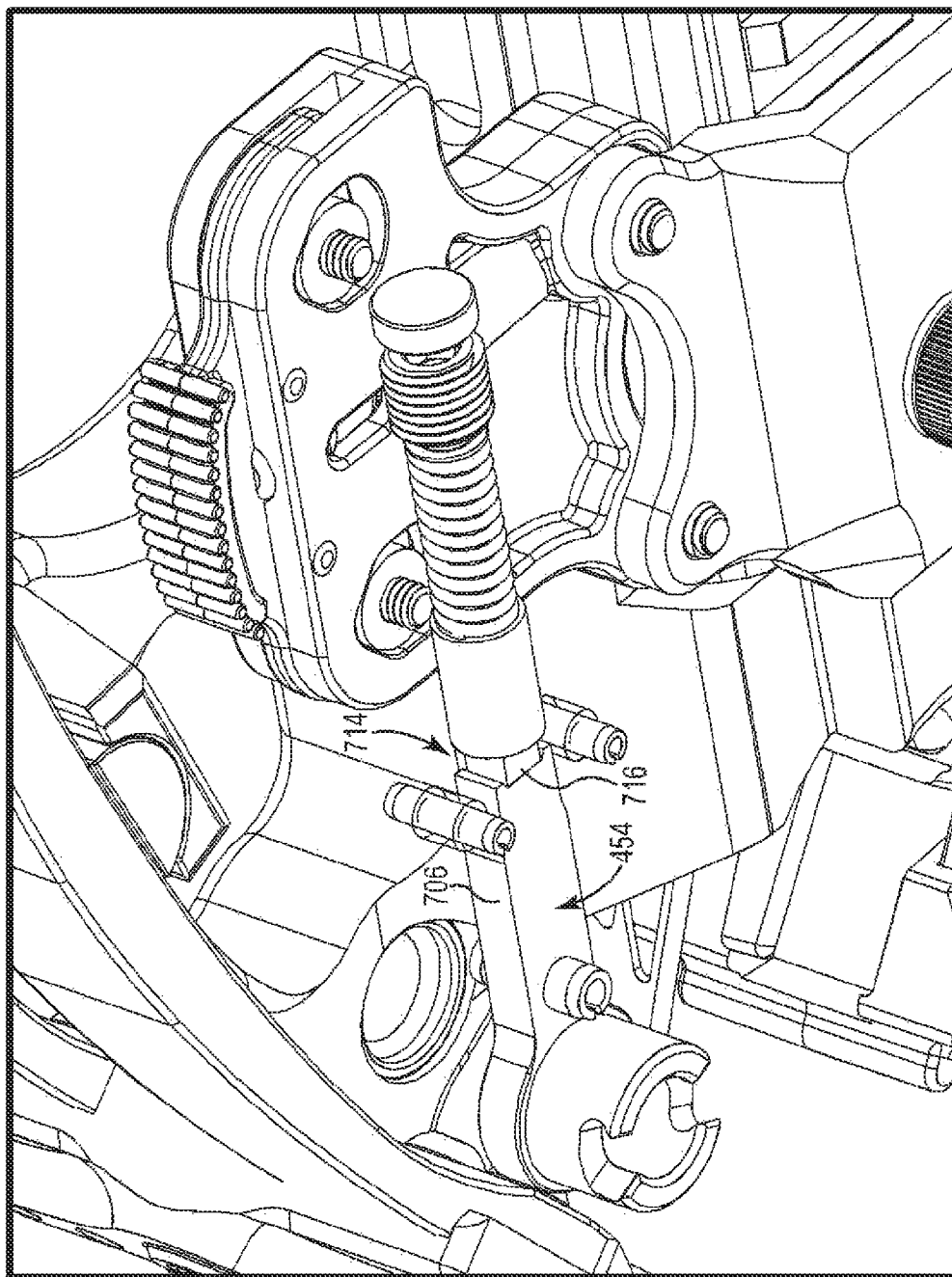


Fig. 14E

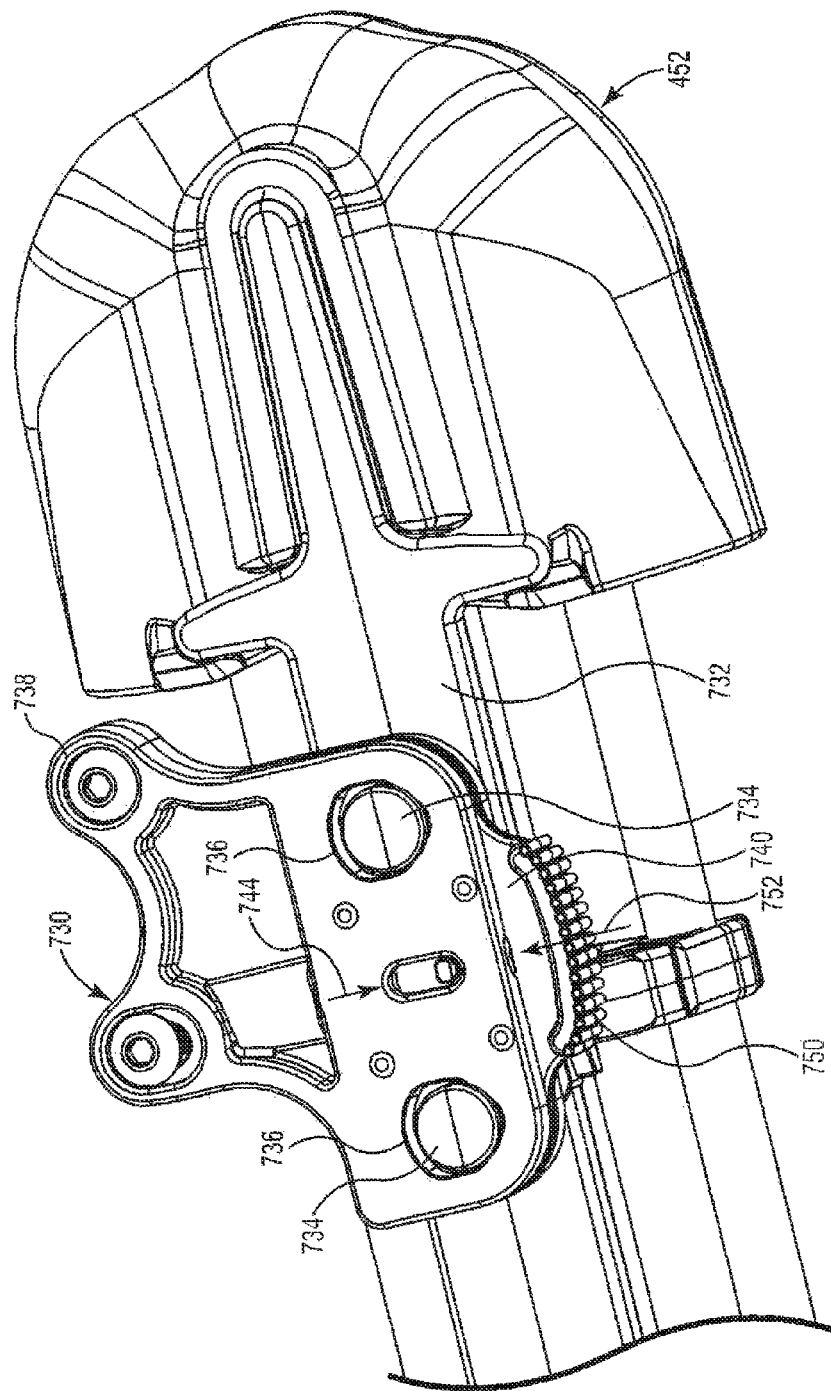


Fig. 14F

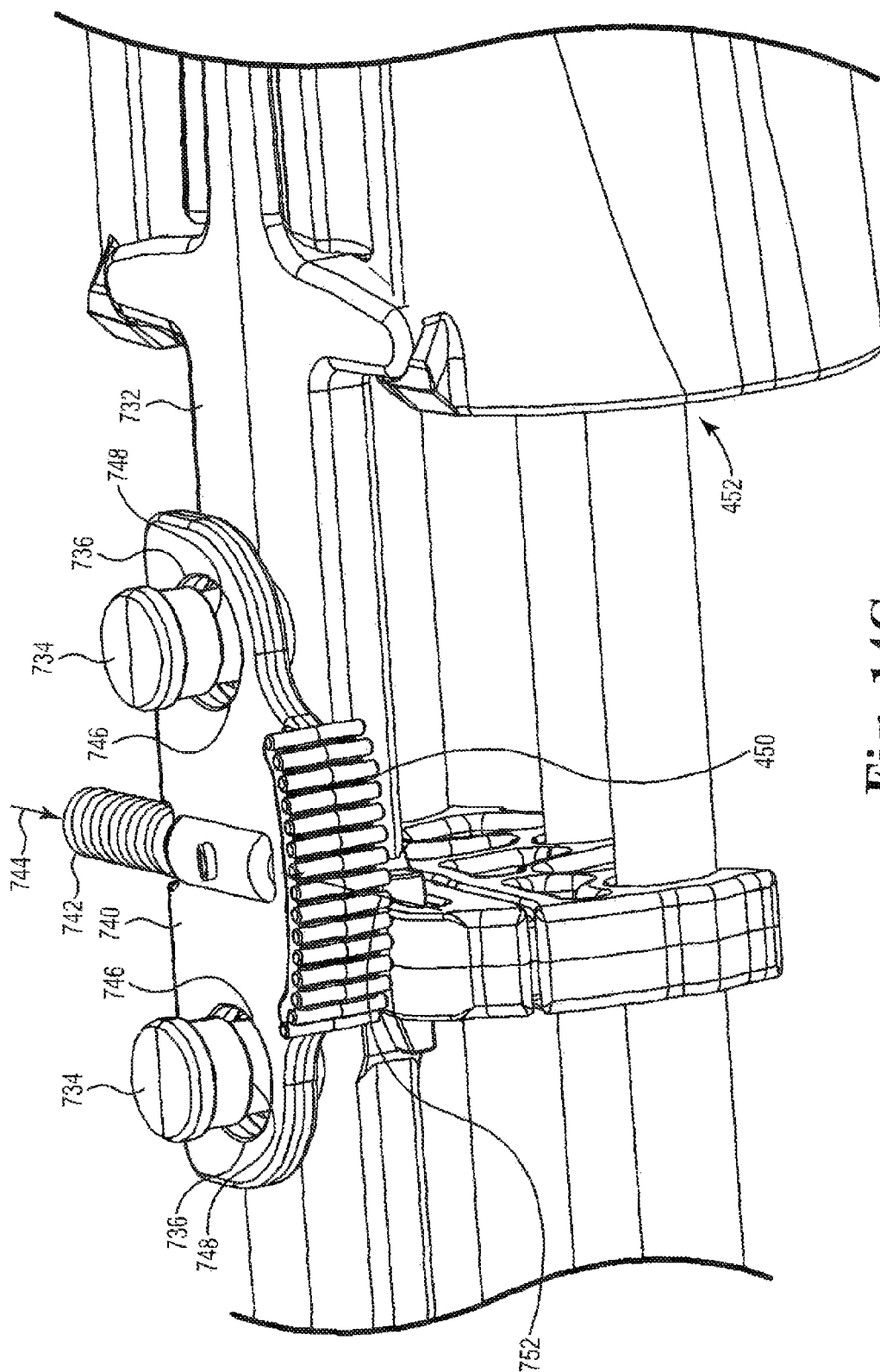


Fig. 14G

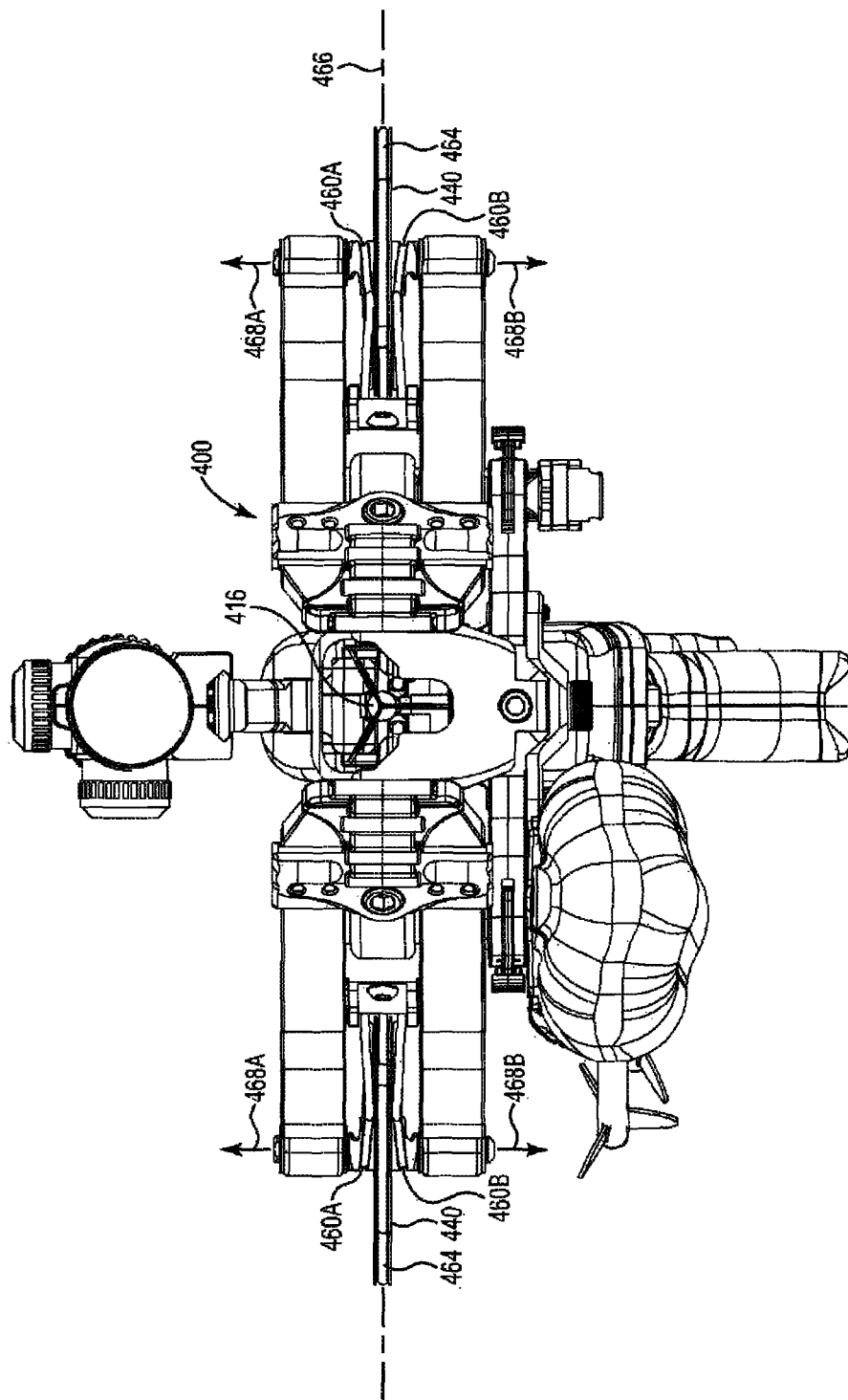


Fig. 15

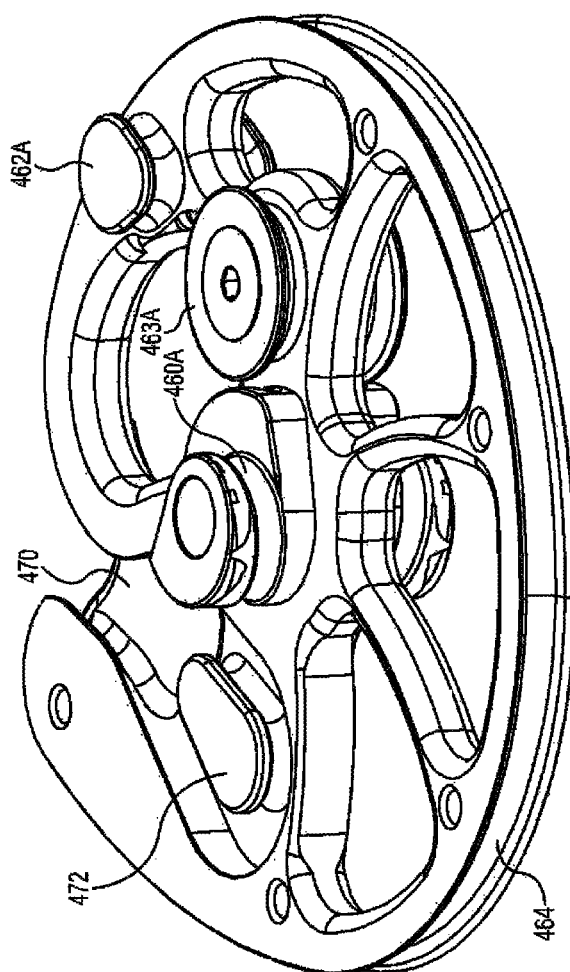


Fig. 16A

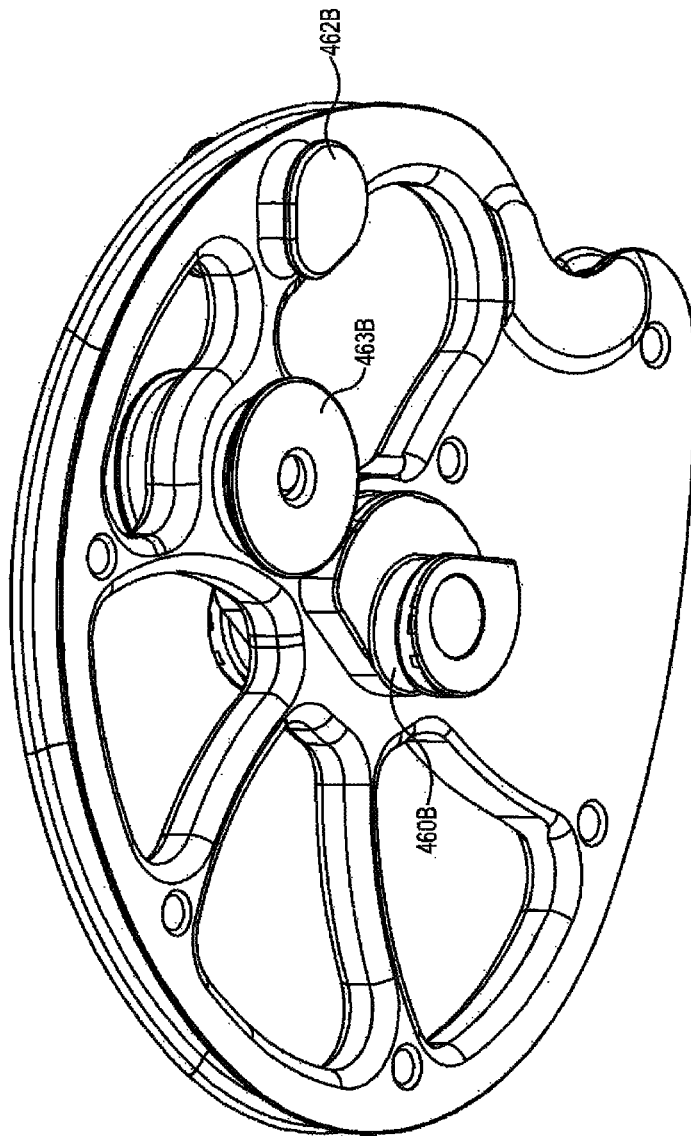


Fig. 16B

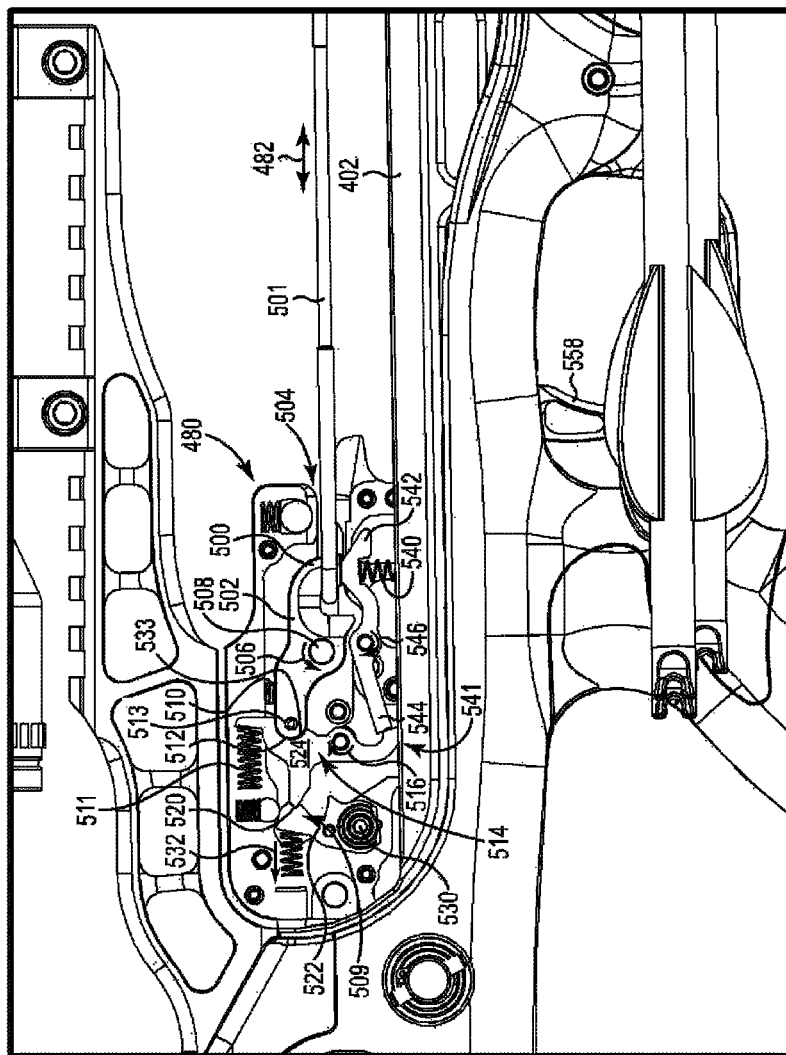


Fig. 17A

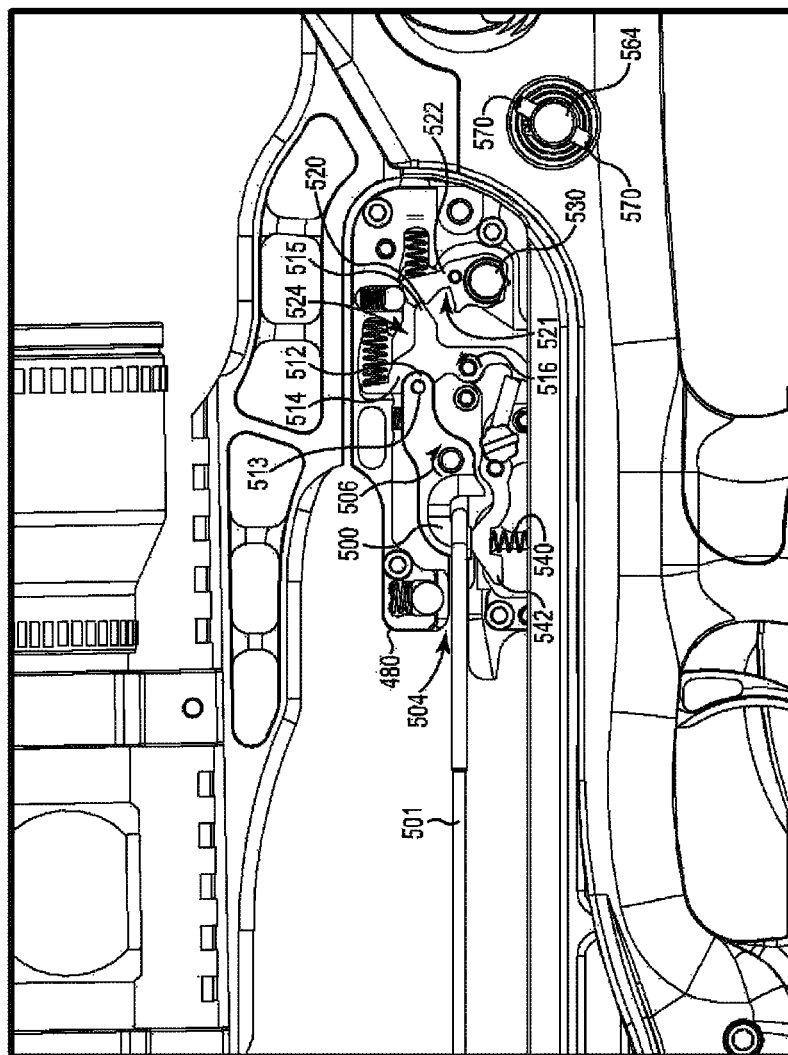


Fig. 17B

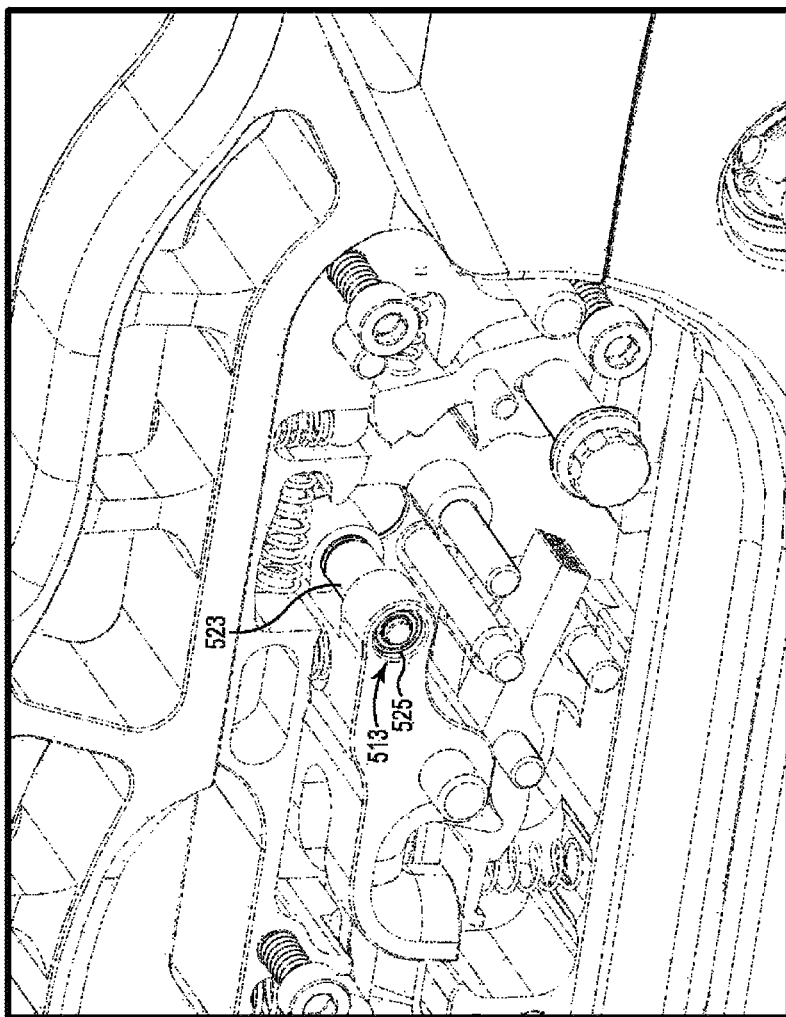


Fig. 17D

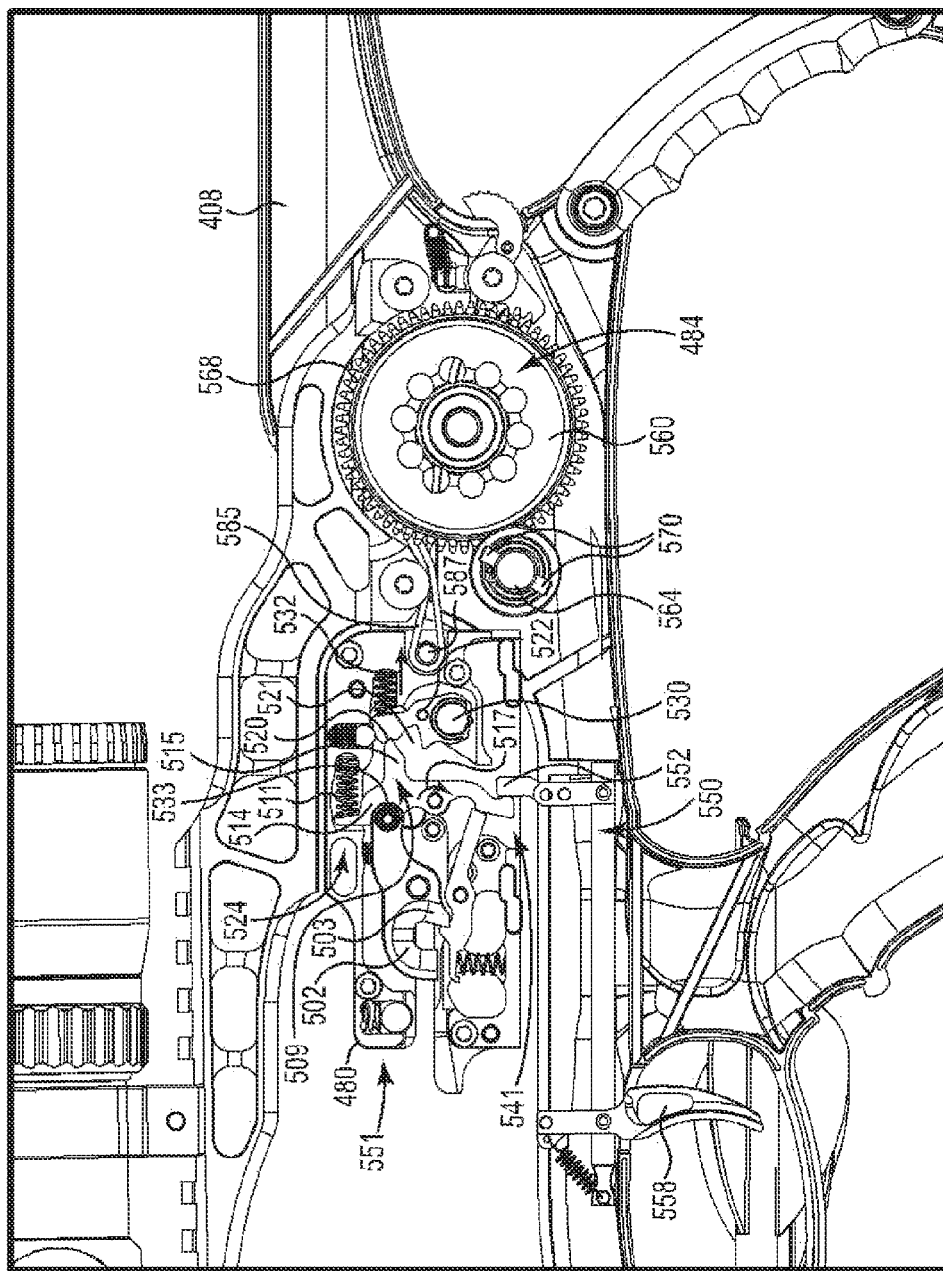


Fig. 18A

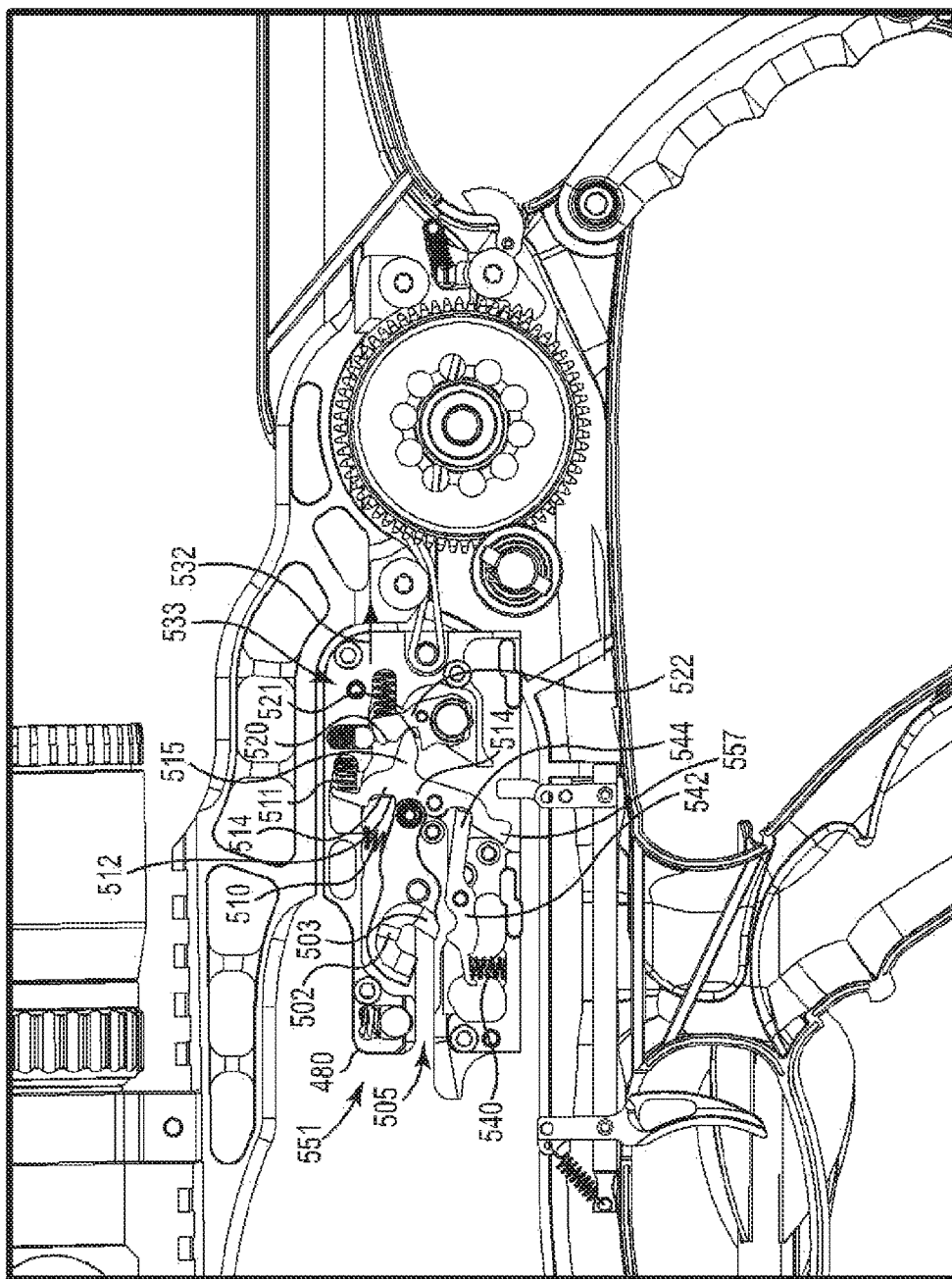


Fig. 18B

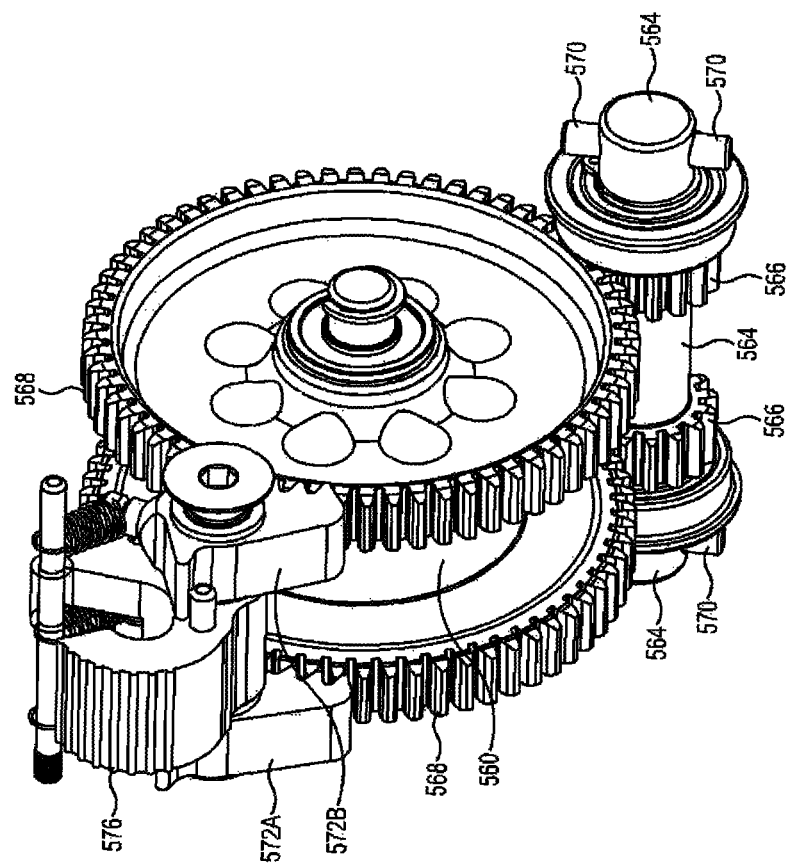


Fig. 19

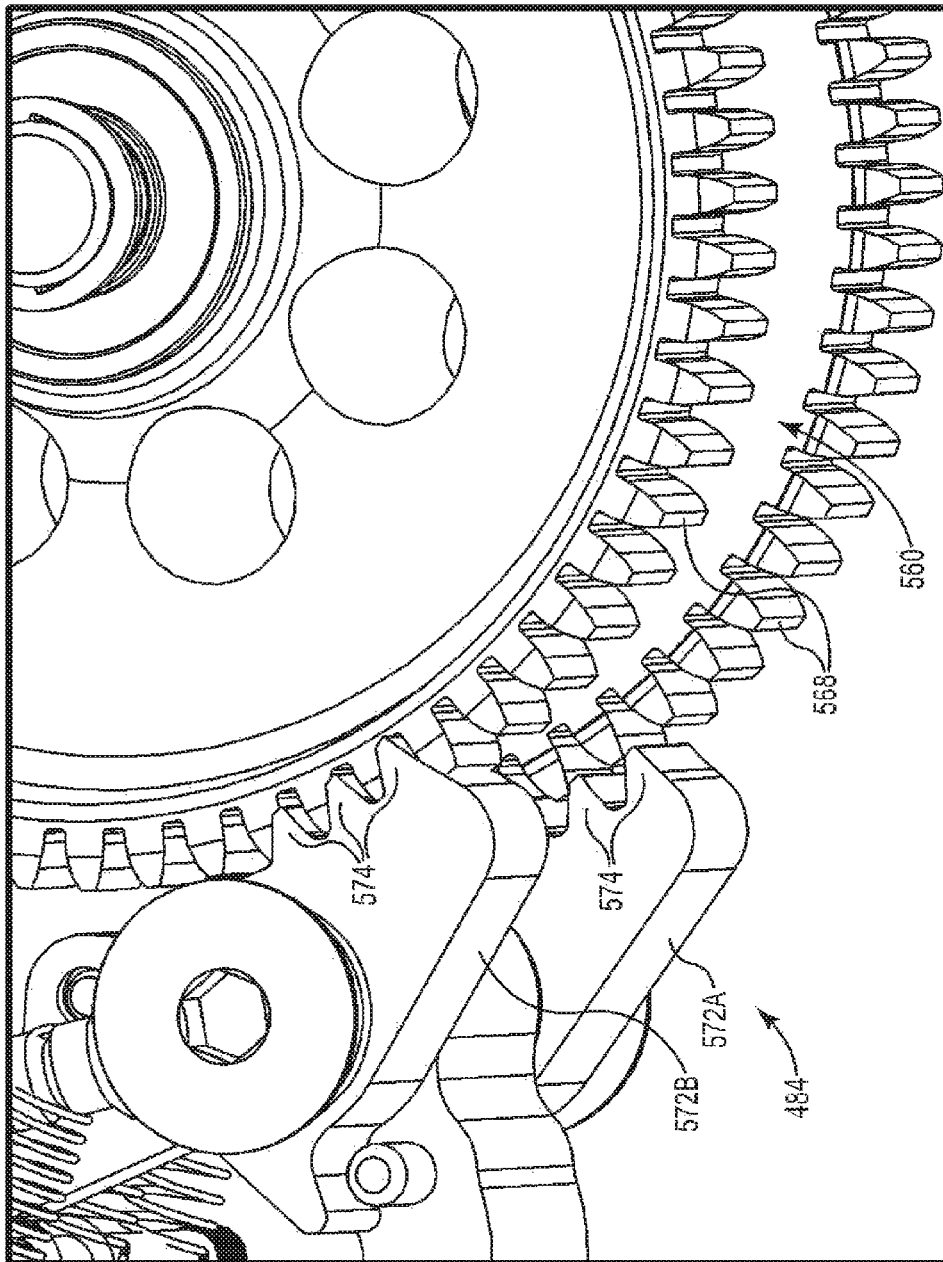


Fig. 20

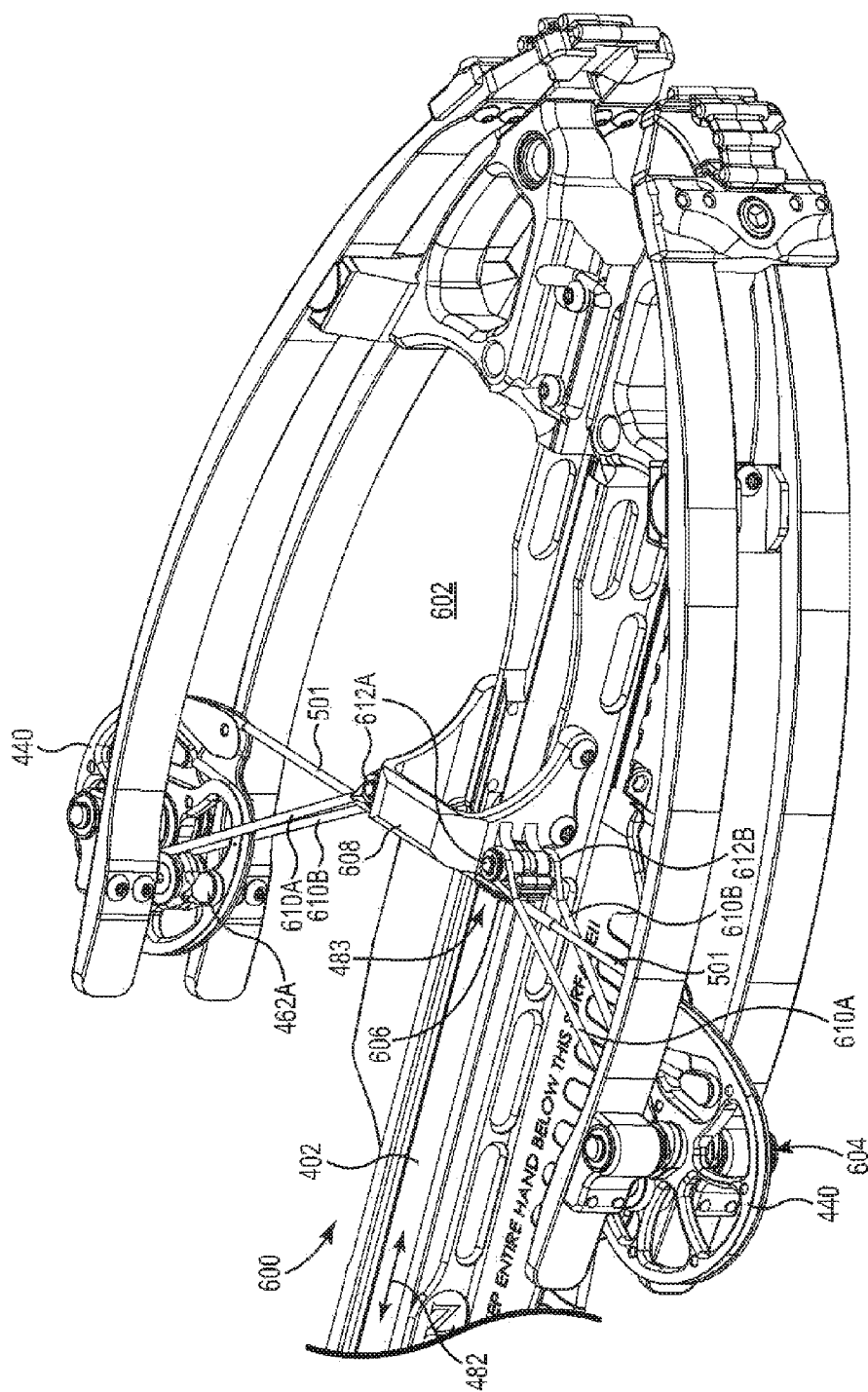


Fig. 21A

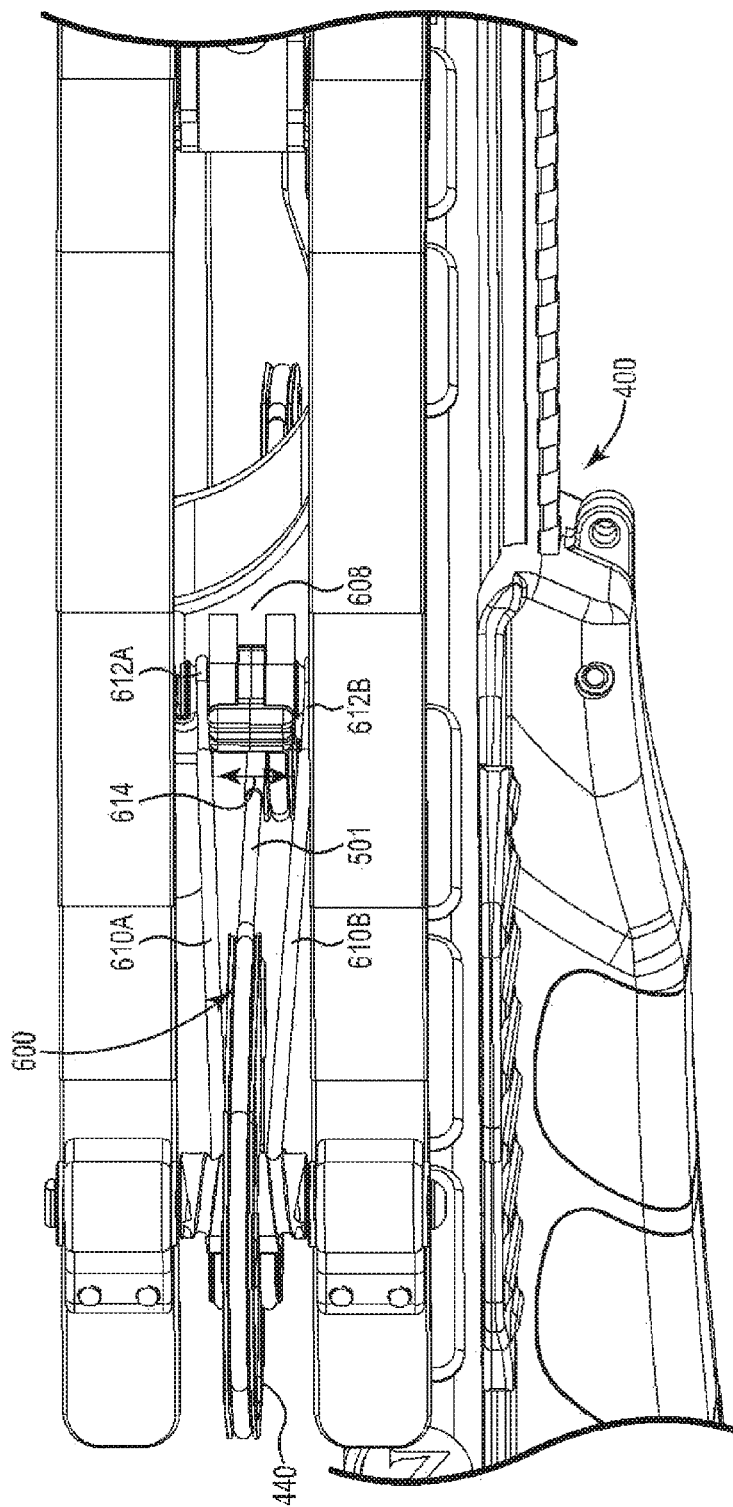
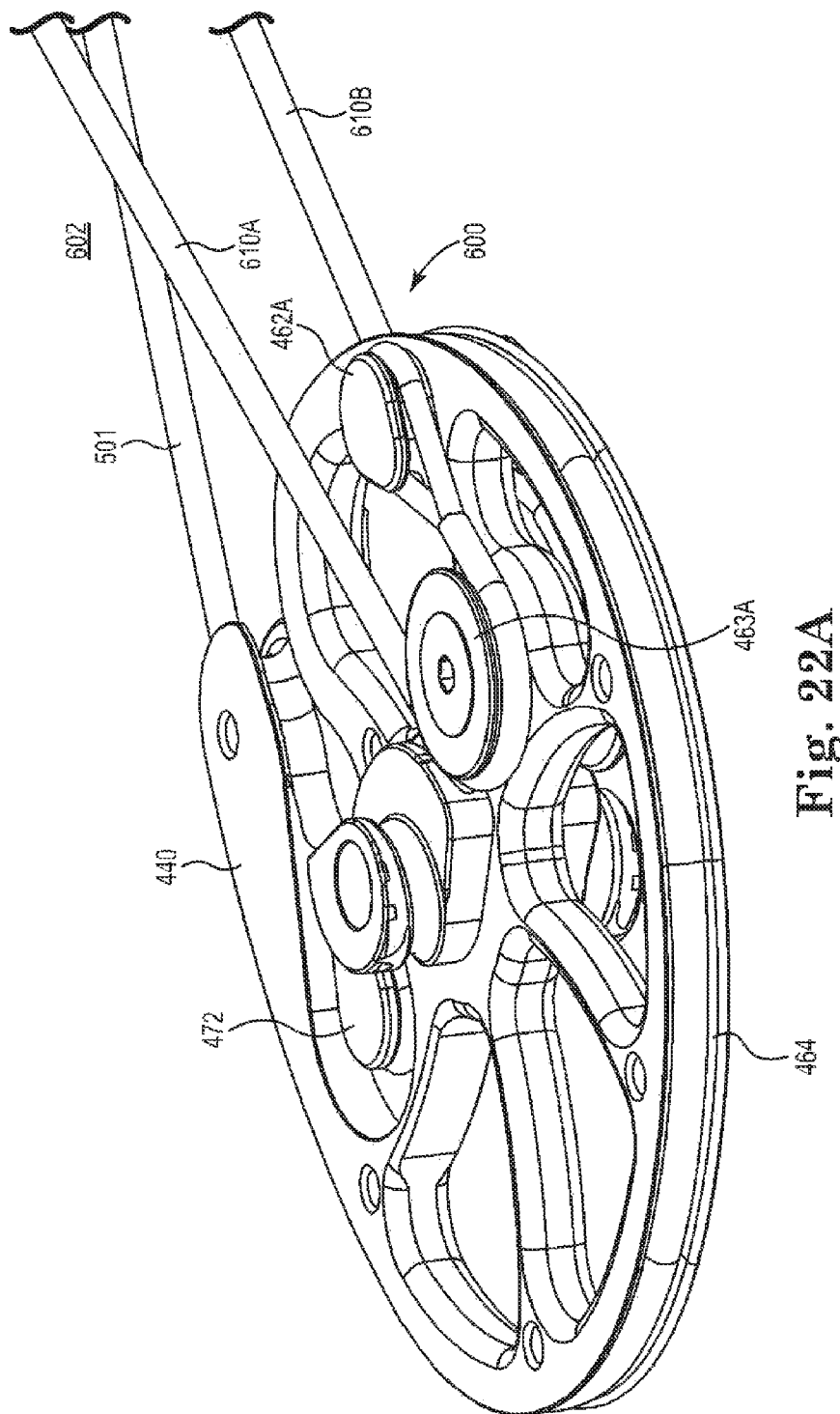


Fig. 21B



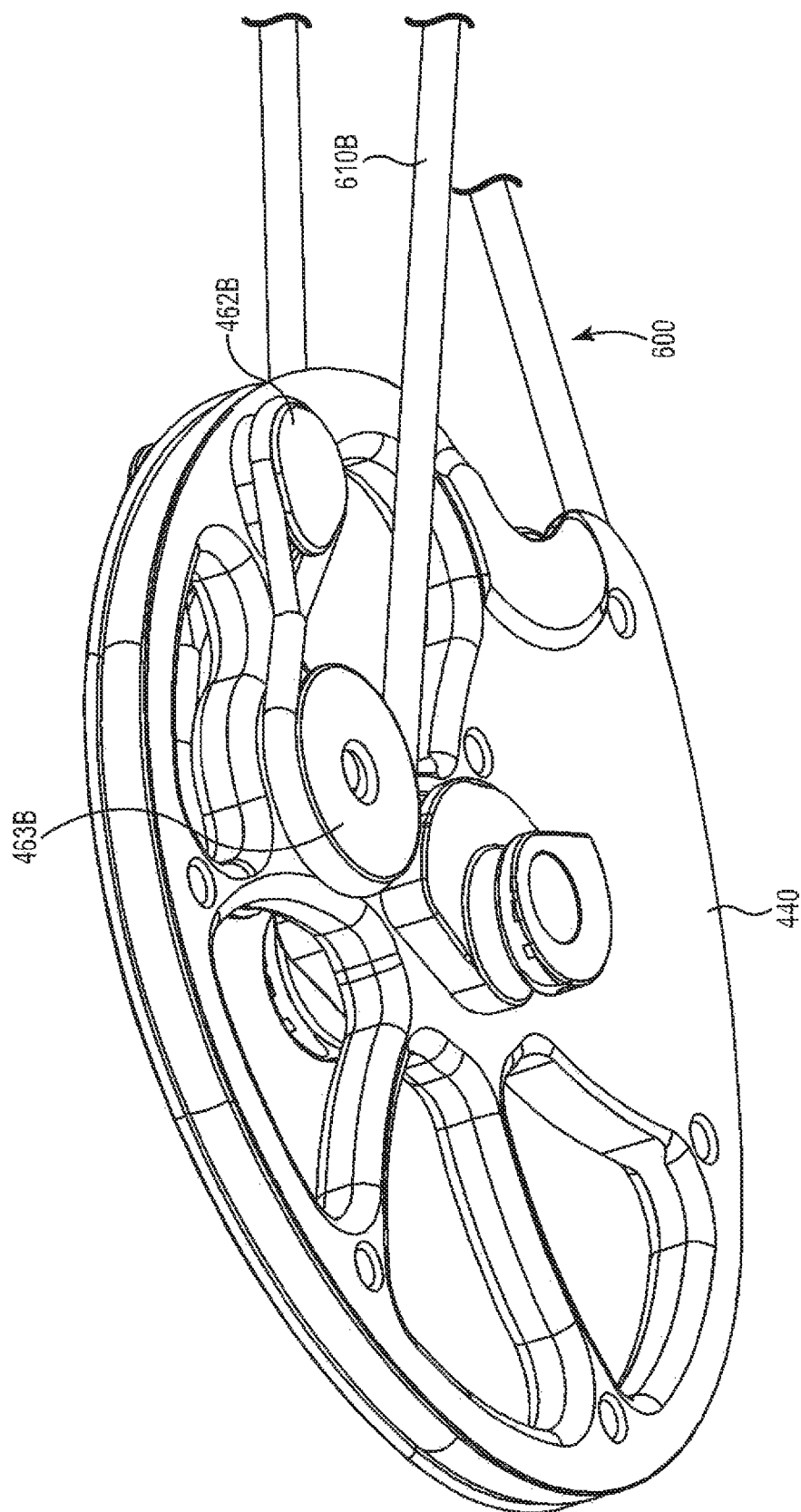


Fig. 22B

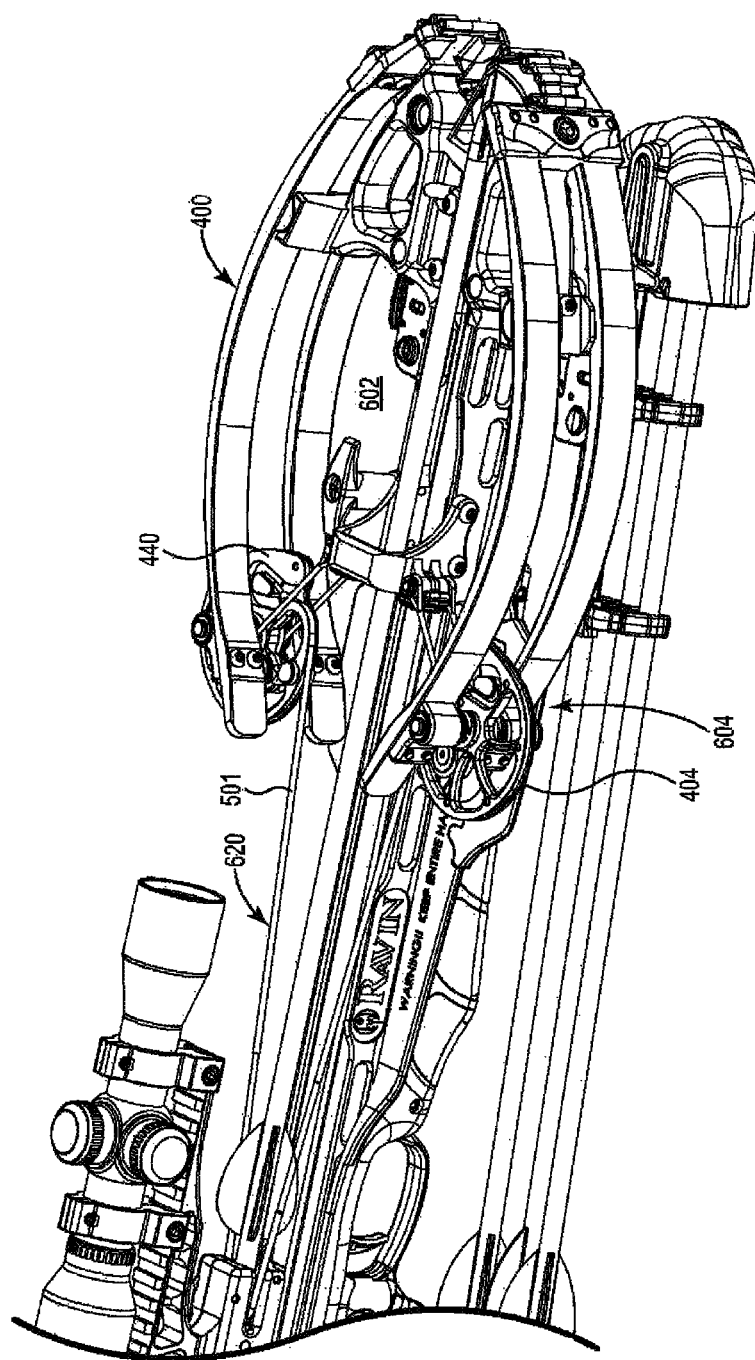


Fig. 23A

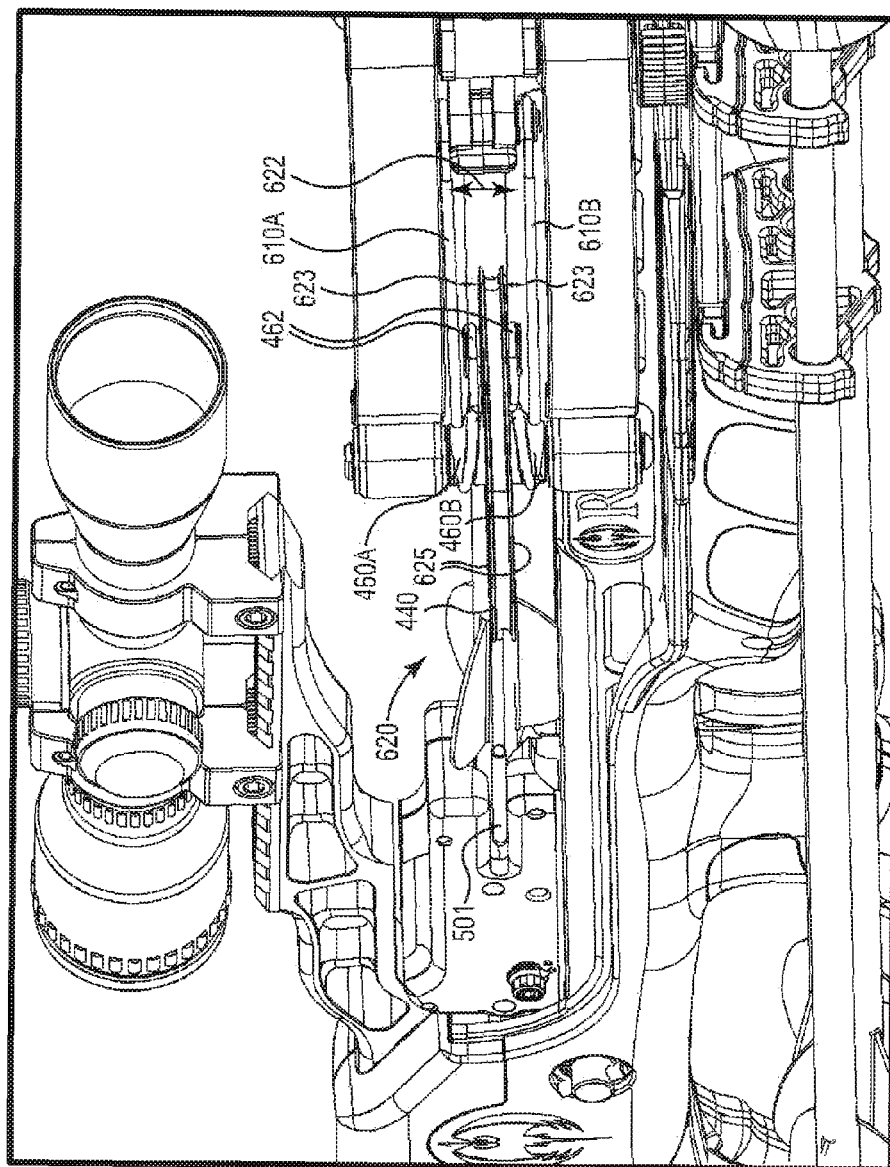


Fig. 23B

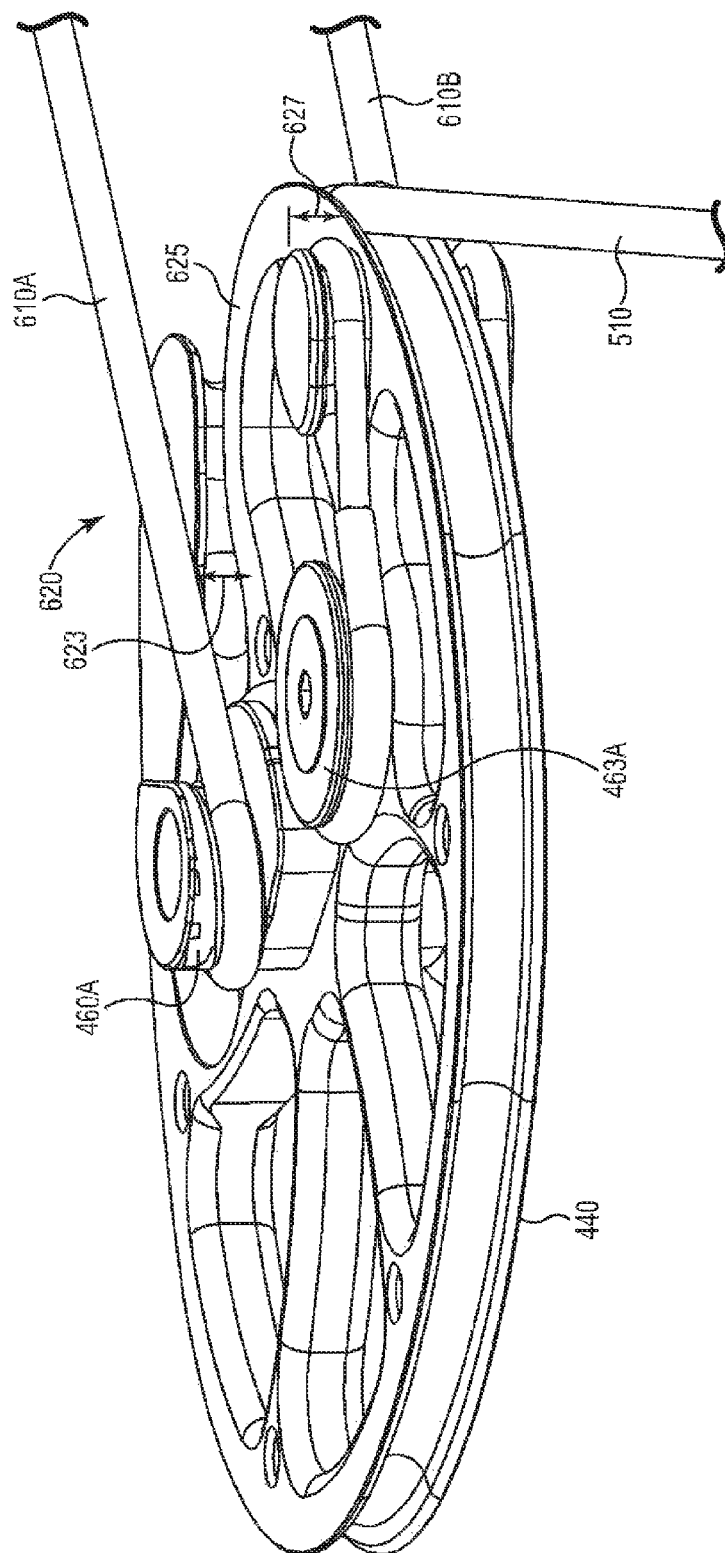


Fig. 24A

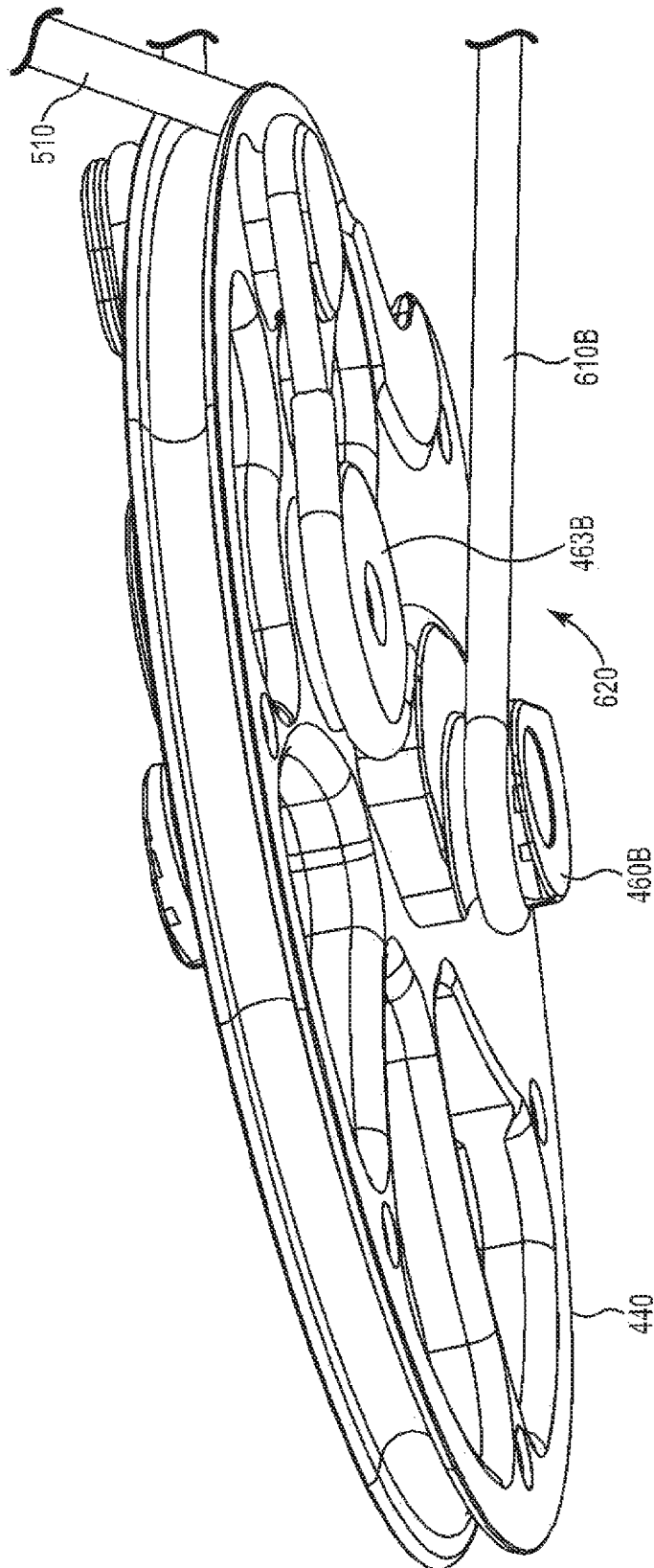


Fig. 24B

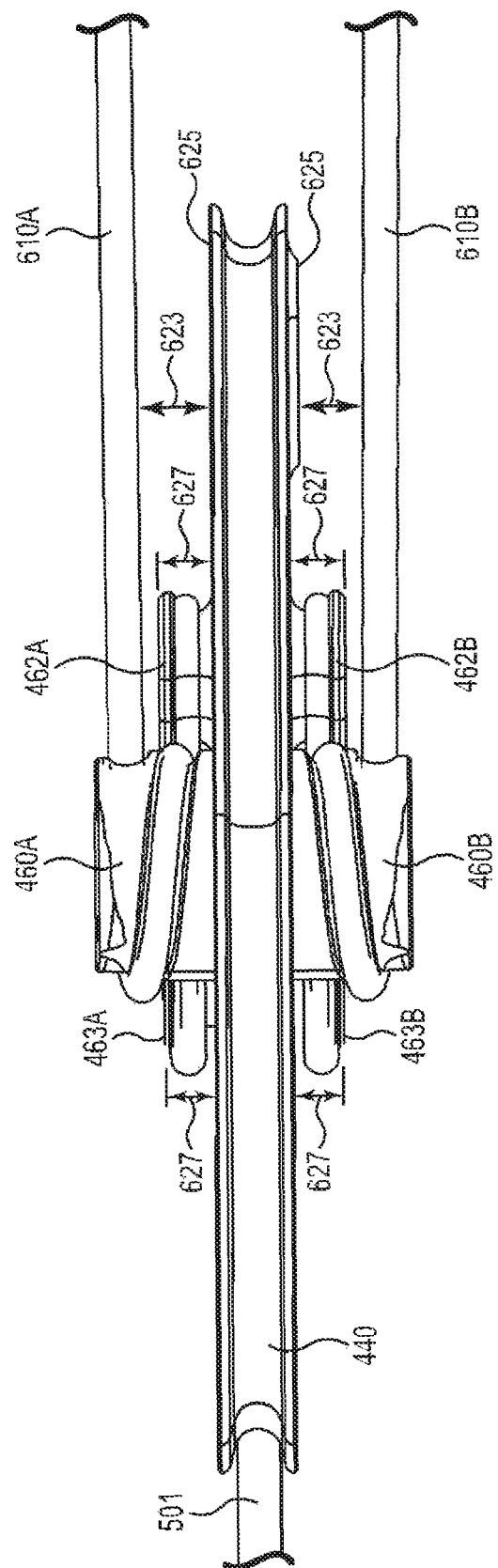


Fig. 24C

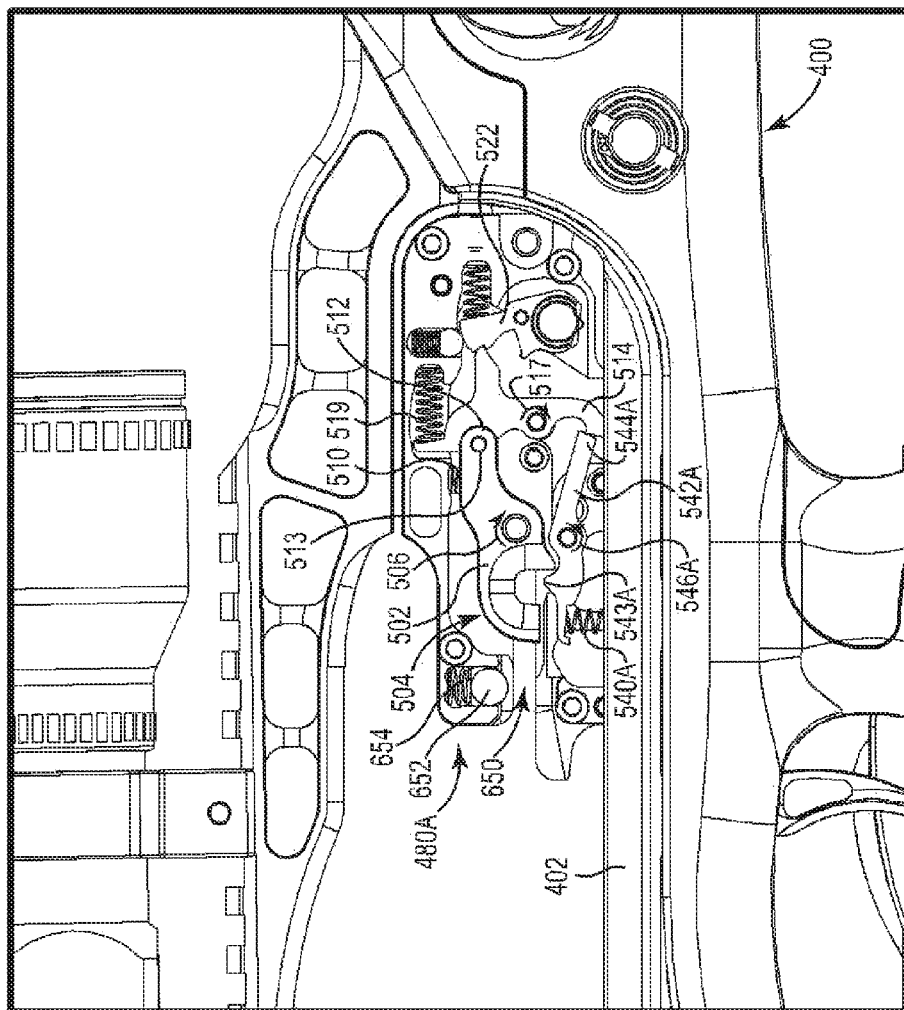


Fig. 25A

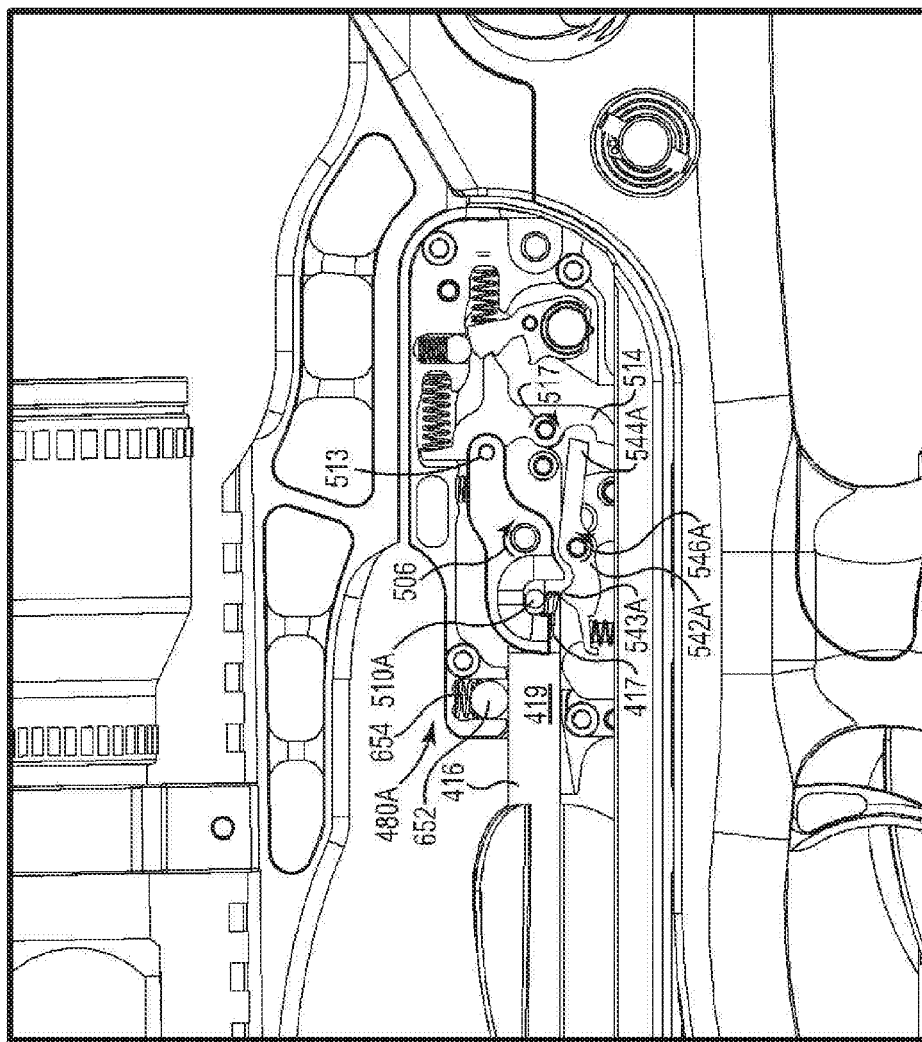


Fig. 25B

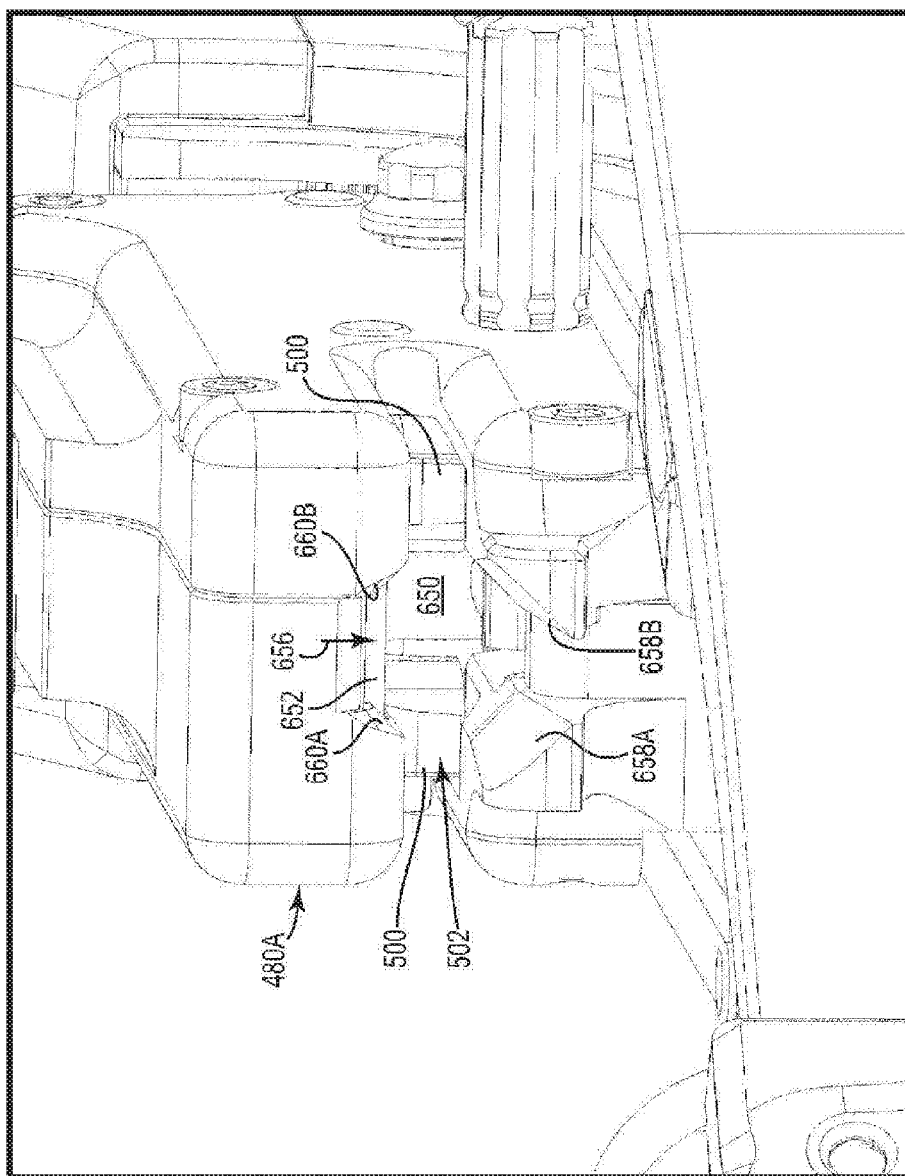


Fig. 25C

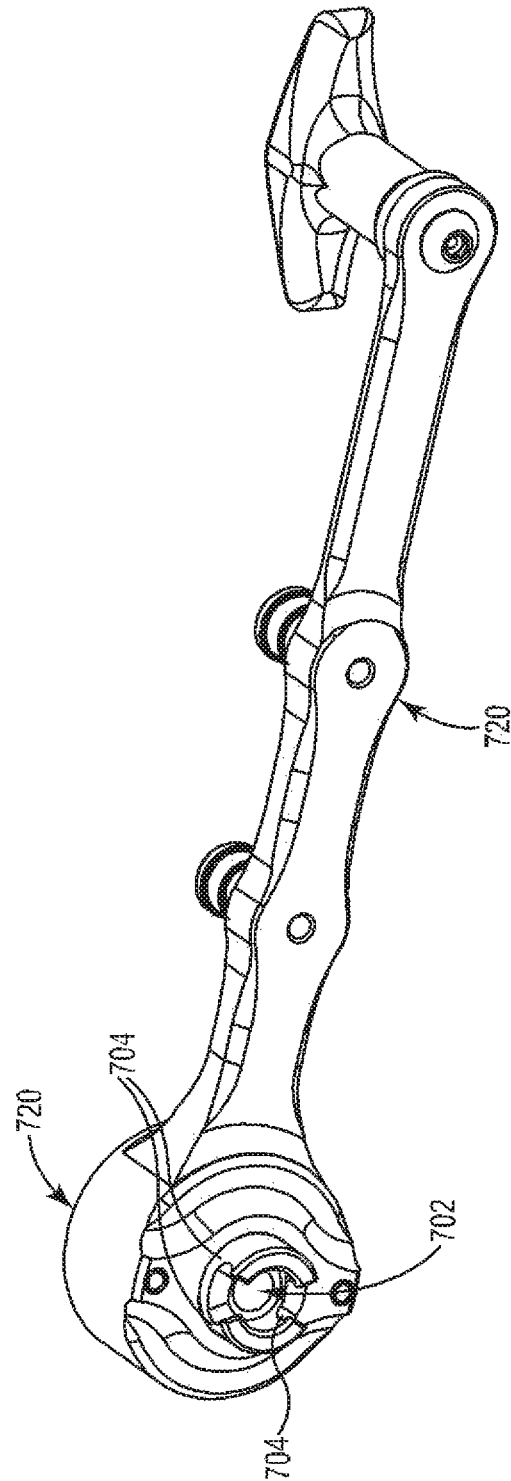


Fig. 26A

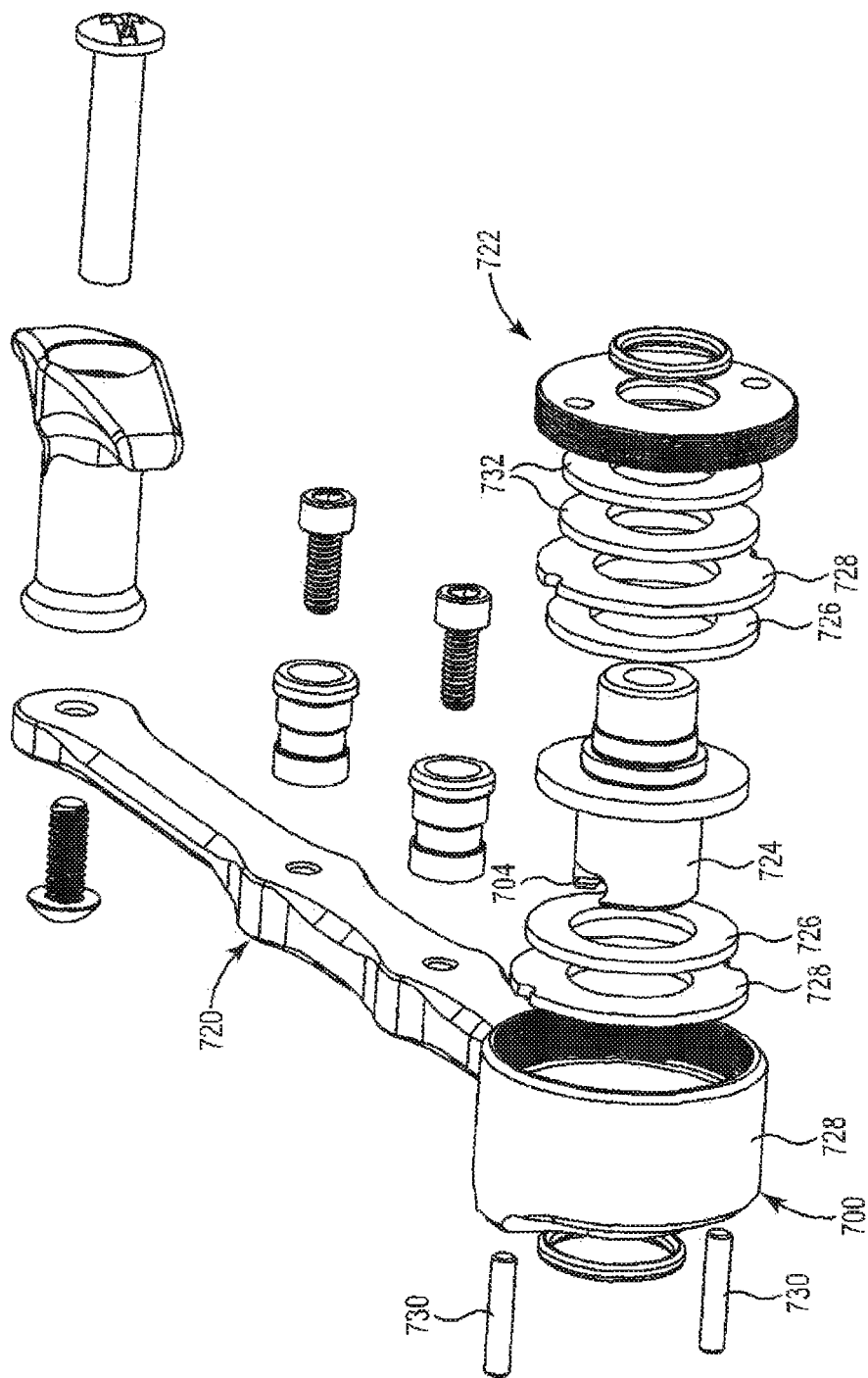


Fig. 26B

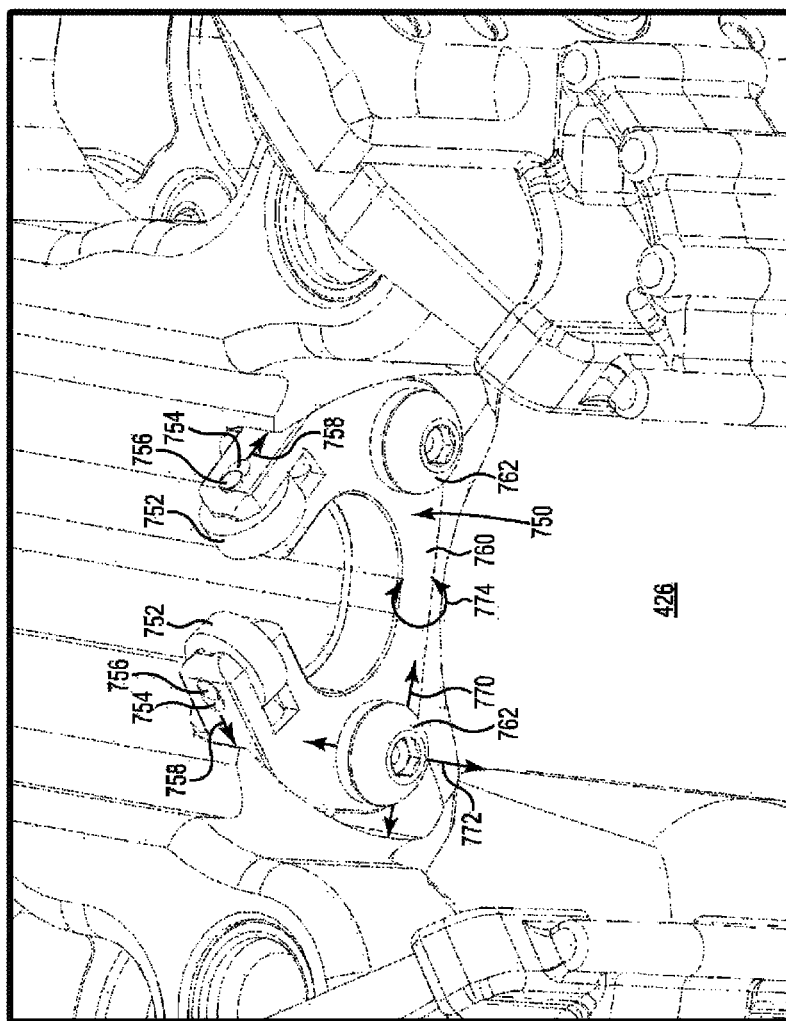


Fig. 27A

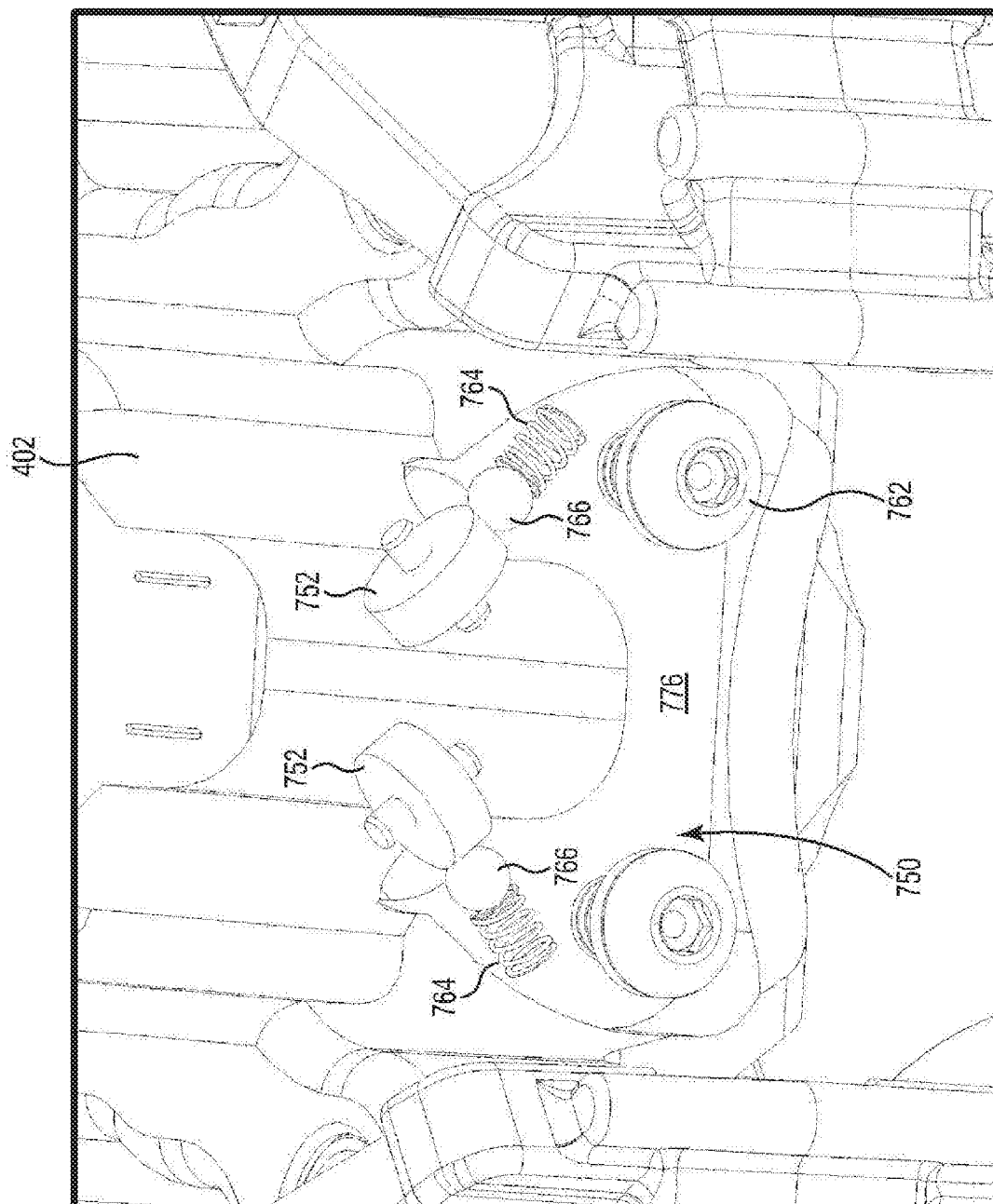


Fig. 27B

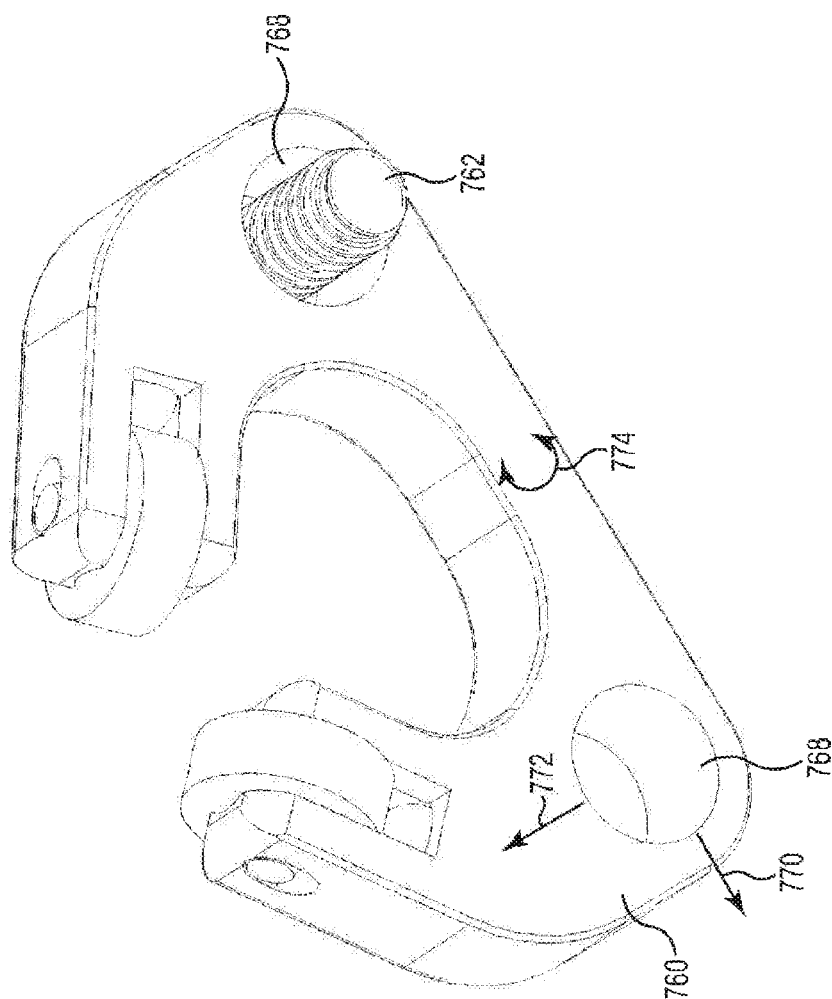


Fig. 27C

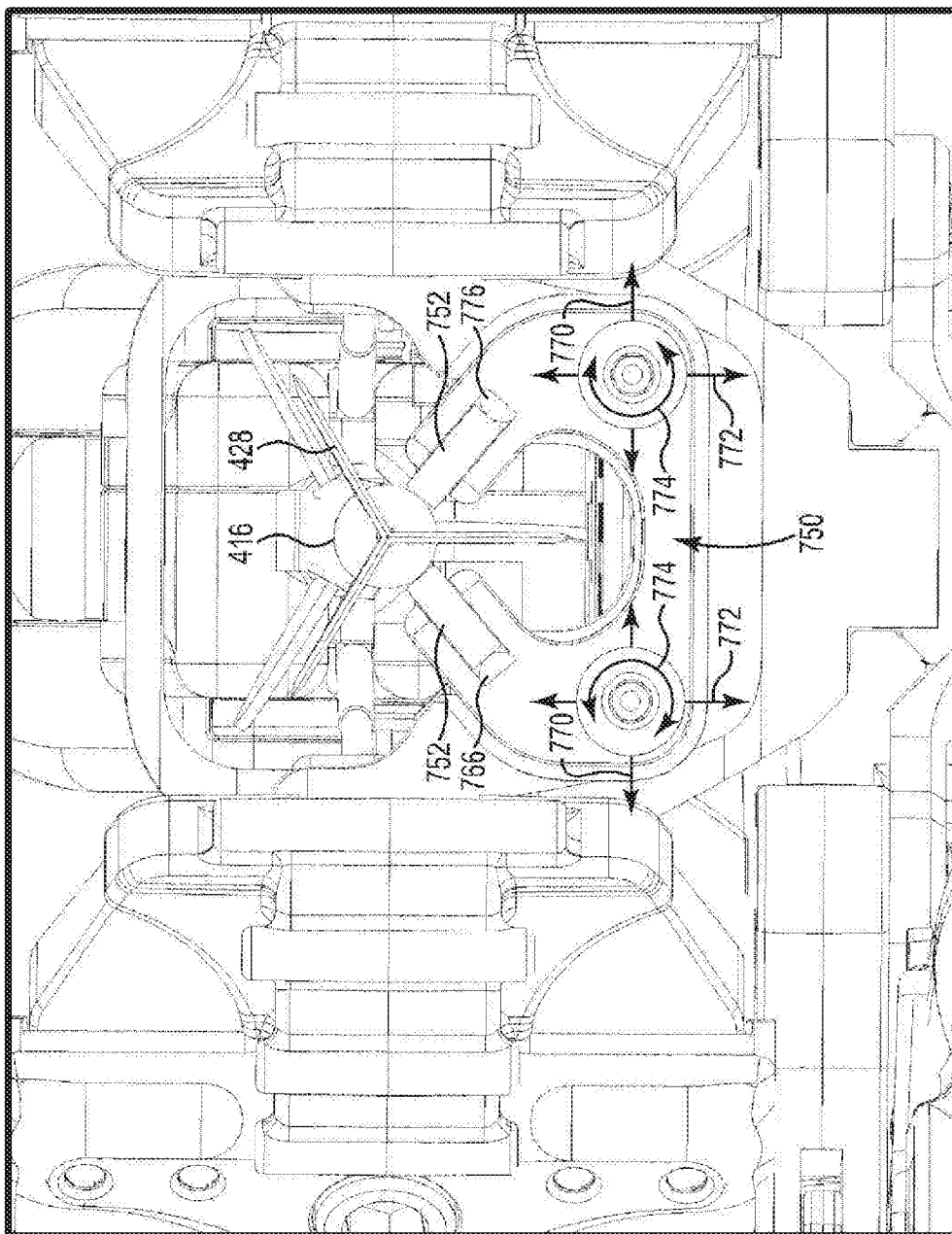


Fig. 27D

1

CROSSBOW

REFERENCE TO RELATED APPLICATIONS

The present application is a continuation-in-part of U.S. patent Ser. No. 14/107,058 entitled String Guide System for a Bow, filed Dec. 16, 2013, and claims the benefit of U.S. Prov. Application Ser. No. 62/244,932, filed Oct. 22, 2015, the entire disclosures of which are hereby incorporated by reference.

FIELD OF THE INVENTION

The present disclosure is directed to a crossbow with a string guide system that permits greater rotation of the cams and pulleys and a longer power stroke.

BACKGROUND OF THE INVENTION

Bows have been used for many years as a weapon for hunting and target shooting. More advanced bows include cams that increase the mechanical advantage associated with the draw of the bowstring. The cams are configured to yield a decrease in draw force near full draw. Such cams preferably use power cables that load the bow limbs. Power cables can also be used to synchronize rotation of the cams, such as disclosed in U.S. Pat. No. 7,305,979 (Yehle).

With conventional bows and crossbows the draw string is typically pulled away from the generally concave area between the limbs and away from the riser and limbs. This design limits the power stroke for bows and crossbows.

In order to increase the power stroke, the draw string can be positioned on the down-range side of the string guides so that the draw string unrolls between the string guides toward the user as the bow is drawn, such as illustrated in U.S. Pat. No. 7,836,871 (Kempf) and U.S. Pat. No. 7,328,693 (Kempf). One drawback of this configuration is that the power cables can limit the rotation of the cams to about 270 degrees. In order to increase the length of the power stroke, the diameter of the pulleys needs to be increased. Increasing the size of the pulleys results in a larger and less usable bow.

FIGS. 1-3 illustrate a string guide system for a bow that includes power cables 20A, 20B ("20") attached to respective string guides 22A, 22B ("22") at first attachment points 24A, 24B ("24"). The second ends 26A, 26B ("26") of the power cables 20 are attached to the axles 28A, 28B ("28") of the opposite string guides 22. Draw string 30 engages down-range edges 46A, 46B of string guides 22 and is attached at draw string attachment points 44A, 44B ("44").

As the draw string 30 is moved from released configuration 32 of FIG. 1 to drawn configuration 34 of FIGS. 2 and 3, the string guides 22 counter-rotate toward each other about 270 degrees. The draw string 30 unwinds between the string guides 22 from opposing cam journals 48A, 48B ("48") in what is referred to as a reverse draw configuration. As the first attachment points 24 rotate in direction 36, the power cables 20 are wrapped around respective power cable take-up journal of the string guides 22, which in turn bends the limbs toward each other to store the energy needed for the bow to fire the arrow.

Further rotation of the string guides 22 in the direction 36 causes the power cables 20 to contact the power cable take-up journal, stopping rotation of the cam. The first attachment points 24 may also contact the power cables 20 at the locations 38A, 38B ("38"), preventing further rotation

2

in the direction 36. As a result, rotation of the string guides 22 is limited to about 270 degrees, reducing the length 40 of the power stroke.

BRIEF SUMMARY OF THE INVENTION

The present disclosure is directed to a crossbow with a string guide system that permits greater rotation of the string guides and a longer power stroke.

The present disclosure is directed to a crossbow including a riser located proximate a down-range side of a center rail and a trigger assembly located proximate an up-range side of the center rail. First and second flexible limbs are attached to the riser and extend toward the up-range side. First and second cams are mounted to the first and second bow limbs, respectively, with each of the cams having a string guide journal, an upper power cable take-up journal, and a lower power cable take-up journal. A draw string is received in the string guide journals and secured to the first and second cams in a reverse draw configuration with the draw string adjacent the down-range side of the string guides in a released configuration. The draw string translates from the down-range side toward the up-range side and unwinds from the string guide journals to a drawn configuration. Upper and lower power cables for each of the first and second cams have first ends secured to each of the first and second cams and second ends secured to the center rail without crossover. The upper and lower power cables are received in each of the respective upper and lower power cable take-up journals and displaced away from the respective draw string journals as the bow is drawn from the released configuration to the drawn configuration.

In one embodiment the upper and lower power cable take-up journals are helical in shape to translate the upper and lower power cable away from the first and second cams, respectively, as the bow is drawn from the released configuration to the drawn configuration. In another embodiment, the upper and lower power cable take-up journals have a width at least twice a width of the upper and lower power cables to permit the upper and lower power cables to be displaced away from the first and second cams and the draw string when the crossbow is drawn.

The first and second cams preferably rotate between about 270 degrees to about 330 degrees, and more preferably, between about 300 degrees and about 360 degrees, when the crossbow is drawn from the released configuration to the drawn configuration. In another embodiment, the first and second cams rotate more than about 360 degrees when the crossbow is drawn from the released configuration to the drawn configuration.

In another embodiment, the radial displacement of the first and second string guides as the draw string translates between the released configuration and the drawn configuration is between about 270 degrees to about 290 degrees, between about 290 degrees and about 310 degrees, or between about 310 degrees to about 360 degrees.

In one embodiment the second ends of the upper and lower power cables are connected to attachment points on respective sides of the center rail in a spaced apart configuration such that the upper and lower power cables are generally parallel to each other when the crossbow is in the drawn configuration. The first ends of the upper and lower power cables are attached to power cable attachments located on upper and lower surfaces of the first and second cams, respectively. The power cable attachments pass under

3

the respective upper and lower power cables as the crossbow moves between the released configuration and the drawn configuration.

In another embodiment, the first and second cams are mounted to the first and second bow limbs, respectively, with first and second axes of rotation offset from distal tips of the bow limbs. The first and second axes of rotation are preferably offset from the distal tips of the bow limbs by about 1.0 inch to about 3.0 inches.

BRIEF DESCRIPTION OF THE SEVERAL VIEWS OF THE DRAWING

FIG. 1 is a bottom view of a prior art string guide system for a bow in a released configuration.

FIG. 2 is a bottom view of the string guide system of FIG. 1 in a drawn configuration.

FIG. 3 is a perspective view of the string guide system of FIG. 1 in a drawn configuration.

FIG. 4 is a bottom view of a string guide system for a bow with a helical take-up journal in accordance with an embodiment of the present disclosure.

FIG. 5 is a bottom view of the string guide system of FIG. 4 in a drawn configuration.

FIG. 6 is a perspective view of the string guide system of FIG. 4 in a drawn configuration.

FIG. 7 is an enlarged view of the left string guide of the string guide system of FIG. 4.

FIG. 8 is an enlarged view of the right string guide of the string guide system of FIG. 4.

FIG. 9A is an enlarged view of a power cable take-up journal sized to receive two full wraps of the power cable in accordance with an embodiment of the present disclosure.

FIG. 9B is an enlarged view of a power cable take-up journal and draw string journal sized to receive two full wraps of the power cable and draw string in accordance with an embodiment of the present disclosure.

FIG. 9C is an enlarged view of an elongated power cable take-up journal in accordance with an embodiment of the present disclosure.

FIG. 10 is a schematic illustration of a bow with a string guide system in accordance with an embodiment of the present disclosure.

FIG. 11 is a schematic illustration of an alternate bow with a string guide system in accordance with an embodiment of the present disclosure.

FIG. 12 is a schematic illustration of an alternate dual-cam bow with a string guide system in accordance with an embodiment of the present disclosure.

FIGS. 13A and 13B are top and side views of a crossbow with helical power cable journals in accordance with an embodiment of the present disclosure.

FIG. 14A is an enlarged top view of the crossbow of FIG. 13A.

FIG. 14B is an enlarged bottom view of the crossbow of FIG. 13A.

FIG. 14C illustrates an arrow rest in accordance with an embodiment of the present disclosure.

FIGS. 14D and 14E illustrate the cocking handle for the crossbow of FIG. 13A.

FIGS. 14F and 14G illustrate the quiver for the crossbow of FIG. 13A.

FIG. 15 is a front view of the crossbow of FIG. 13A.

FIGS. 16A and 16B are top and bottom views of cams with helical power cable journals in accordance with an embodiment of the present disclosure.

4

FIGS. 17A and 17B are opposite side view of a trigger assembly in accordance with an embodiment of the present disclosure.

FIG. 17C is a side view of the trigger of FIG. 17A with a bolt engaged with the draw string in accordance with an embodiment of the present disclosure.

FIG. 17D is a perspective view of a low friction interface at a rear edge of a string catch in accordance with an embodiment of the present disclosure.

FIGS. 18A and 18B illustrate operation of the trigger mechanism in accordance with an embodiment of the present disclosure.

FIGS. 19 and 20 illustrate a cocking mechanism for a crossbow in accordance with an embodiment of the present disclosure.

FIGS. 21A and 21B illustrate a crossbow in a release configuration in accordance with an embodiment of the present disclosure.

FIGS. 22A and 22B illustrate the cams of the crossbow of FIGS. 21A and 21B in the release configuration.

FIGS. 23A and 23B illustrate the crossbow of FIGS. 21A and 21B in a drawn configuration in accordance with an embodiment of the present disclosure.

FIGS. 24A, 24B, and 24C illustrate the cams of the crossbow of FIGS. 23A and 23B in the drawn configuration.

FIGS. 25A and 25B illustrate an alternate trigger assembly in accordance with an embodiment of the present disclosure.

FIG. 25C is a front view of an alternate string carrier for the crossbow in accordance with an embodiment of the present disclosure.

FIGS. 26A and 26B illustrate an alternate cocking handle in accordance with an embodiment of the present disclosure.

FIGS. 27A-27D illustrate an alternate tunable arrow rest for a crossbow in accordance with an embodiment of the present disclosure.

DETAILED DESCRIPTION OF THE INVENTION

FIG. 4 illustrates a string guide system 90 for a bow with a reverse draw configuration 92 in accordance with an embodiment of the present disclosure. Power cables 102A, 102B ("102") are attached to respective string guides 104A, 104B ("104") at first attachment points 106A, 106B ("106"). Second ends 108A, 108B ("108") of the power cables 102 are attached to axles 110A, 110B ("110") of the opposite string guides 104. In the illustrated embodiment, the power cables 102 wrap around power cable take-ups 112A, 112B ("112") located on the respective cam assemblies 104 when in the released configuration 116 of FIG. 4.

In the reverse draw configuration 92 the draw string 114 is located adjacent down-range side 94 of the string guide system 70 when in the released configuration 116. In the released configuration 116 of FIG. 4, the distance between the axles 110 may be in the range of less than about 16 inches to less than about 10 inches. In the drawn configuration 118, the distance between the axles 110 may be in the range of about 6 inches to about 8 inches.

As illustrated in FIGS. 5 and 6, the draw string 114 translates from the down-range side 94 toward the up-range side 96 and unwinds between the first and second string guides 104 in a drawn configuration 118. In the illustrated embodiment, the string guides 104 counter-rotate toward each other in directions 120 more than 360 degrees as the draw string 114 unwinds between the string guides 104 from opposing cam journals 130A, 130B ("130").

5

The string guides **104** each include one or more grooves, channels or journals located between two flanges around at least a portion of its circumference that guides a flexible member, such as a rope, string, belt, chain, and the like. The string guides can be cams or pulleys with a variety of round and non-round shapes. The axis of rotation can be located concentrically or eccentrically relative to the string guides. The power cables and draw strings can be any elongated flexible member, such as woven and non-woven filaments of synthetic or natural materials, cables, belts, chains, and the like.

As the first attachment points **106** rotate in direction **120**, the power cables **102** are wrapped onto cams **126A**, **126B** (“**126**”) with helical journals **122A**, **122B** (“**122**”), preferably located at the respective axles **110**. The helical journals **122** take up excess slack in the power cables **102** resulting from the string guides **104** moving toward each other in direction **124** as the axles **110** move toward each other.

The helical journals **122** serve to displace the power cables **102** away from the string guides **104**, so the first attachment points **106** do not contact the power cables **102** while the bow is being drawn (see FIGS. **7** and **8**). As a result, rotation of the string guides **104** is limited only by the length of the draw string journals **130A**, **130B** (“**130**”). For example, the draw string journals **130** can also be helically in nature, wrapping around the axles **110** more than 360 degrees.

As a result, the power stroke **132** is extended. In the illustrated embodiment, the power stroke **132** can be increased by at least 25%, and preferably by 40% or more, without changing the diameter of the string guides **104**.

In some embodiments, the geometric profiles of the draw string journals **130** and the helical journals **122** contribute to let-off at full draw. A more detailed discussion of cams suitable for use in bows is provided in U.S. Pat. No. 7,305,979 (Yehle), which is hereby incorporated by reference.

FIGS. **7** and **8** are enlarged views of the string guides **104A**, **104B**, respectively, with the draw string **114** in the drawn configuration **118**. The helical journals **122** have a length corresponding generally to one full wrap of the power cables **102**. The axes of rotation **146A**, **146B** (“**146**”) of the first and second helical journals **122** preferably extend generally perpendicular to a plane of rotation of the first and second string guides **104**. The helical journals **122** displace the power cables **102** away from the draw string **114** as the bow is drawn from the released configuration **116** to the drawn configuration **118**. Height **140** of the helical journals **122** raises the power cables **102** above top surface **142** of the string guides **104**. The resulting gap **144** permits the first attachment points **106** and the power cable take-ups **112** to pass freely under the power cables **102**. The length of the helical journals **122** can be increased or decreased to optimize draw force versus draw distance for the bow and let-off. The axes of rotation **146** of the helical journals **122** are preferably co-linear with axes **110** of rotation for the string guides **104**.

FIG. **9A** illustrates an alternate string guide **200** in accordance with an embodiment of the present disclosure. Power cable take-ups **202** have helical journals **204** that permit the power cables **102** to wrap around about two full turns or about 720 degrees. The extended power cable take-up **202** increases the gap **206** between the power cables **102** and top surface **208** of the string guide **200** and provides excess capacity to accommodate more than 360 degrees of rotation of the string guides **200**.

6

FIG. **9B** illustrates an alternate string guide **250** in accordance with an embodiment of the present disclosure. The draw string journals **252** and the power cable journals **254** are both helical structures designed so that the draw string **114** and the power cables **102** can wrap two full turns around the string guide **250**.

FIG. **9C** illustrates an alternate string guide **270** with a smooth power cable take-up **272** in accordance with an embodiment of the present disclosure. The power cable take-up **272** has a surface **274** with a height **276** at least twice a diameter **278** of the power cable **102**. In another embodiment, the surface **274** has a height **276** at least three times the diameter **278** of the power cable **102**. Biasing force **280**, such as from a cable guard located on the bow shifts the power cables **102** along the surface **274** away from top surface **282** of the string guide **270** when in the drawn configuration **284**.

FIG. **10** is a schematic illustration of bow **150** with a string guide system **152** in accordance with an embodiment of the present disclosure. Bow limbs **154A**, **154B** (“**154**”) extend oppositely from handle **156**. String guides **158A**, **158B** (“**158**”) are rotatably mounted, typically eccentrically, on respective limbs **154A**, **154B** on respective axles **160A**, **160B** (“**160**”) in a reverse draw configuration **174**.

Draw string **162** is received in respective draw string journals (see e.g., FIGS. **7** and **8**) and secured at each end to the string guides **158** at locations **164A**, **164B**. When the bow is in the released configuration **176** illustrated in FIG. **10**, the draw string **162** is located adjacent the down-range side **178** of the bow **150**. When the bow **150** is drawn, the draw string **162** unwinds from the draw string journals toward the up-range side **180** of the bow **150**, thereby rotating the string guides **158** in direction **166**.

First power cable **168A** is secured to the first string guide **158A** at first attachment point **170A** and engages with a power cable take-up with a helical journal **172A** (see FIGS. **7** and **8**) as the bow **150** is drawn. As the string guide **158A** rotates in the direction **166**, the power cable **168A** is taken up by the cam **172A**. The other end of the first power cable **168A** is secured to the axle **160B**.

Second power cable **168B** is secured to the second string guide **158B** at first attachment point **170B** and engages with a power cable take-up with a helical journal **172B** (see FIGS. **7** and **8**) as the bow **150** is drawn. As the string guide **158B** rotates, the power cable **168B** is taken up by the cam **172B**. The other end of the second power cable **168B** is secured to the axle **160A**. The power cable take-ups **172** are arranged so that as the bow **150** is drawn, the bow limbs **154** are drawn toward one another.

FIG. **11** is a schematic illustration of a crossbow **300** with a reverse draw configuration **302** in accordance with an embodiment of the present disclosure. The crossbow **300** includes a center portion **304** with down-range side **306** and up-range side **308**. In the illustrated embodiment, the center portion **304** includes riser **310**. First and second flexible limbs **312A**, **312B** (“**312**”) are attached to the riser **310** and extend from opposite sides of the center portion **304**.

Draw string **314** extends between first and second string guides **316A**, **316B** (“**316**”). In the illustrated embodiment, the string guide **316A** is substantially as shown in FIGS. **4-8**, while the string guide **316B** is a conventional pulley.

The first string guide **316A** is mounted to the first bow limb **312A** and is rotatable around a first axis **318A**. The first string guide **316A** includes a first draw string journal **320A** and a first power cable take-up journal **322A**, both of which are oriented generally perpendicular to the first axis **318A**. (See e.g., FIG. **8**). The first power cable take-up journal

322A includes a width measured along the first axis **318A** that is at least twice a width of power cable **324**.

The second string guide **316B** is mounted to the second bow limb **312A** and rotatable around a second axis **318B**. The second string guide **316B** includes a second draw string journal **320B** oriented generally perpendicular to the second axis **318B**.

The draw string **314** is received in the first and second draw string journals **320A**, **320B** and is secured to the first string guide **316A** at first attachment point **324**. The draw string extends adjacent to the down-range side **306** to the second string guide **316B**, wraps around the second string guide **316B**, and is attached at the first axis **318A**.

Power cable **324** is attached to the string guide **316A** at attachment point **326**. See FIG. 4. Opposite end of the power cable **324** is attached to the axis **318B**. In the illustrated embodiment, power cable wraps **324** onto the first power cable take-up journal **322A** and translates along the first power cable take-up journal **322A** away from the first draw string journal **320A** as the bow **300** is drawn from the released configuration **328** to the drawn configuration (see FIGS. 5-8).

FIG. 12 is a schematic illustration of a dual-cam crossbow **350** with a reverse draw configuration **352** in accordance with an embodiment of the present disclosure. The crossbow **350** includes a center portion **354** with down-range side **356** and up-range side **358**. First and second flexible limbs **362A**, **362B** (“**362**”) are attached to riser **360** and extend from opposite sides of the center portion **354**. Draw string **364** extends between first and second string guides **366A**, **366B** (“**366**”). In the illustrated embodiment, the string guides **366** are substantially as shown in FIGS. 4-8.

The string guides **366** are mounted to the bow limb **362** and are rotatable around first and second axis **368A**, **368B** (“**368**”), respectively. The string guides **366** include first and second draw string journals **370A**, **370B** (“**370**”) and first and second power cable take-up journals **372A**, **372B** (“**372**”), both of which are oriented generally perpendicular to the axes **368**, respectively. (See e.g., FIG. 8). The power cable take-up journals **372** include widths measured along the axes **368** that is at least twice a width of power cables **374A**, **374B** (“**374**”).

The draw string **364** is received in the draw string journals **370** and is secured to the string guides **316** at first and second attachment points **375A**, **375B** (“**325**”).

Power cables **374** are attached to the string guides **316** at attachment points **376A**, **376B** (“**376**”). See FIG. 4. Opposite ends **380A**, **380B** (“**380**”) of the power cables **374** are attached to anchors **378A**, **378B** (“**378**”) on the center portion **354**. The power cables **374** preferably do not cross over the center support **354**.

In the illustrated embodiment, power cables wrap **374** onto the power cable take-up journal **372** and translates along the power cable take-up journals **372** away from the draw string journals **370** as the bow **350** is drawn from the released configuration **378** to the drawn configuration (see FIGS. 5-8).

The string guides disclosed herein can be used with a variety of bows and crossbows, including those disclosed in commonly assigned U.S. patent application Ser. No. 13/799,518, entitled Energy Storage Device for a Bow, filed Mar. 13, 2013 and Ser. No. 14/071,723, entitled DeCocking Mechanism for a Bow, filed Nov. 5, 2013, both of which are hereby incorporated by reference.

FIGS. 13A and 13B illustrate an alternate crossbow **400** in accordance with an embodiment of the present disclosure. The crossbow **400** includes a center rail **402** with a riser **404**

mounted at the distal end **406** and a stock **408** located at the proximal end **410**. The arrow **416** is suspended above the rail **402** before firing. In one embodiment, the central rail **402** and the riser **404** may be a unitary structure, such as, for example, a molded carbon fiber component. In the illustrated embodiment, the stock **408** includes a scope mount **412** with a tactical, picatinny, or weaver mounting rail. Scope **414** preferably includes a reticle with gradations corresponding to the ballistic drop of bolts **416** of particular weight. The riser **404** includes a pair of limbs **420A**, **420B** (“**420**”) extending rearward toward the proximal end **410**. In the illustrated embodiment, the limbs **420** have a generally concave shape directed toward the center rail **402**. The terms “bolt” and “arrow” are both used for the projectiles launch by crossbows and are used interchangeable herein.

FIGS. 14A and 14B are top and bottom views of the riser **404**. Limbs **420** are attached to the riser **404** near the distal end **406** by mounting brackets **422A**, **422B** (“**422**”). In the illustrated embodiment, distal ends **424A**, **424B** (“**424**”) of the limbs **420** extend past the mounting brackets **422** to create pocket **426** that contains arrowhead **428**. Bumpers **430** are preferably attached to the distal ends **424** of the limbs **420**. The tip of the arrowhead **428** is preferably completely contained within the pocket **426**.

Pivots **432A**, **432B** (“**432**”) attached to the riser **404** engage with the limbs **420** proximally from the mounting brackets **422**. The pivots **432** provide a flexure point for the limbs **420** when the crossbow **400** is in the drawn configuration.

Cams **440A**, **440B** (“**440**”) are attached to the limbs **420** by axle mounts **442A**, **442B** (“**442**”). In the illustrated embodiment, the axle mounts **442** are attached to the limbs **420** offset a distance **446** from the proximal ends **444A**, **444B** (“**444**”) of the limbs **420**. Due to their concave shape, greatest width **448** of the limbs **420** (in both the drawn configuration and the release configuration) preferably occurs at a location between the axle mounts **442** and the pivots **432**, not at the proximal ends **444**.

The offset **446** of the axle mounts **442** maximizes the speed of the limbs **420**, minimizes limb vibration, and maximizes energy transfer to the bolts **416**. In particular, the offset **446** is similar to hitting a baseball with a baseball bat at a location offset from the tip of the bat, commonly referred to as the “sweet spot”. The size of the offset **446** is determined empirically for each type of limb. In the illustrated embodiment, the offset **446** is about 1.5 to about 4 inches, and more preferably about 2 to about 3 inches.

Tunable arrow rest **490** is positioned just behind the pocket **426**. A pair of supports **492** are secured near opposite sides of the bolt **416** by fasteners **494**. The supports **492** preferably slide in the plane of the limbs **420**. As best illustrated in FIG. 14C, the separation **496** between the supports **492** can be adjusted to raise or lower front end of the bolt **416** relative to the draw string **501**. In particular, by increasing the separation **496** between the supports **492** the curved profile of the front end of the bolt **416** is lowered relative to the string carrier **480** (see FIG. 17A). Alternatively, by decreasing the separation **496** the curved profile of the bolt **416** is raised.

FIG. 14B illustrates the bottom of the riser **404**. Rail **450** on the riser **404** is used as the attachment point for accessories, such as quiver **452** for holding bolts **416** and cocking handle **454** that engages with pins **570** to rotate the driver shaft **564** (see FIG. 18A).

FIG. 14D illustrates the cocking handle **454** in greater detail. Distal end **700** is configured to engage with drive shaft **564** and pins **570** illustrated in FIG. 18A. Center recess

702 receives the drive shaft 564 and the undercuts 704 engage with the pins 570 when the system is under tension. Consequently, when cocking or uncocking the crossbow 400 the tension in the system locks the pins 570 into the undercuts 704. When tension in the system is removed, the cocking handle 454 can be rotated a few degrees and disengaged from the drive shaft 564.

The distal end 700 includes stem 706 that extends into hollow handle 708. Pins 710 permit the stem 706 to rotate a few degrees around pin 712 in either direction within the hollow handle 708. As best illustrated in FIG. 14E, torque assembly 714 is located in hollow handle 708 that resists rotation of the stem 706 until a pre-set torque is reached. Once that torque threshold is exceeded, the stem 706 breaks free of block 716 and rotates within the hollow handle 708, generating an audible noise and snapping sensation that signal to the user that the crossbow 400 is fully cocked.

FIGS. 14F and 14G illustrate a mounting system 730 for the quiver 452 and the cocking handle 454. Quiver spine 732 includes a pair of mounting posts 734 spaced to engage with openings 736 in the mounting bracket 738. Magazine catch 740 (see FIG. 140) slides within mounting bracket 738. Spring 742 biases the magazine catch 740 in direction 744. Openings 746 in the magazine catch 740 engage with undercuts 748 on the mounting posts 734 under pressure from the spring 742. To remove the quiver 452 the user presses the handle 750 in direction 752 until the openings 746 in the magazine catch 740 are aligned with the openings 736 in the mounting bracket 738. Once aligned, the mounting posts 734 can be removed from the mounting bracket 738.

FIG. 15 is a front view of the crossbow 400 with the draw string or the power cables removed to better illustrate the cams 440 having upper and lower helical journals 460A, 460B above and below draw string journal 464. As illustrated in FIG. 21A, separate power cables 610A, 610B are operatively engaged with each of the helical journals 460A, 460B, and minimizing torque on the cams 440. The draw string journal 464 defines plane 466 that passes through the bolt 416. The helical journals 460A, 460B move the power cables 610A, 610B in directions 468A, 468B, respectively, away from the plane 466 as the bow 400 is drawn.

FIGS. 16A and 16B are upper and lower perspective views of the cams 440 with the power cables and draw string removed. Recess 470 contains draw string mount 472 located generally in the plane 466 of the draw string journal 464. Power cable attachment 462A and pivot post 463A correspond to helical journal 460A. As best illustrated in FIG. 16B, power cable attachment 462B and pivot post 463B corresponds to the helical journal 460B. The pivot posts 463 serve to take-up a portion of the power cables 610 and redirect the power cables 610 onto the helical journals 460.

FIGS. 17A through 17D illustrate string carrier 480 for the crossbow 400 in accordance with an embodiment of the present disclosure. As best illustrated in FIG. 21A, the string carrier 480 slides along axis 482 of the center rail 402 to the location 483 (see FIG. 21A) to capture the draw string 501. After the string carrier 480 captures the draw string 501, the cocking mechanism 484 (see FIGS. 18A and 18B) is used to return the string carrier 480 back to the position illustrated in FIGS. 17A and 17B at the proximal end 410 of the crossbow 400 and into engagement with trigger 558.

The string carrier 480 includes fingers 500 on catch 502 that engage the draw string 501. The catch 502 is illustrated in a closed position 504. After firing the crossbow the catch 502 is retained in open position (see FIG. 18B), such as for example, by spring 510. In the illustrated embodiment, the

catch biasing force is applied to the catch 502 by spring 510 to rotate in direction 506 around pin 508 and retains the catch 502 in the open position 505. Absent an external force, the catch 502 automatically move to open position 505 (see FIG. 18B) and releases the draw string 501.

In the closed position 504 illustrated in FIGS. 17A, 17B, 18A, recess 512 on sear 514 engages low friction device 513 at rear edge of the catch 502 at interface 533 to retain the catch 502 in the closed position 504. The sear 514 is biased in direction 516 by a sear biasing force applied by spring 511 to engage with and retain the catch 502 in the closed position 504.

FIG. 17D illustrates the string carrier 480 with the sear 514 removed for clarity. In the illustrated embodiment, the low friction device 513 is a roller pin 523 mounted in rear portion of the catch 520. In one embodiment, the roller pin 523 has a diameter corresponding generally to the diameter of the recess 512. The roller pin 523 is preferably supported by ball bearings 525 to reduce friction between the catch 502 and the recess 512 when firing the crossbow 400. A force necessary to overcome the friction at the interface 533 to release the catch 502 is preferably less than about 1 pound, substantially reducing the trigger pull weight. In an alternate embodiment, the positions of the roller pin 523 and the ball bearings 525 can be reversed so that the sear 514 engages directly on the ball bearings 525.

In one embodiment, a force necessary to overcome the friction at the interface 533 to release the catch 502 is preferably less than the biasing force applied to the sear 514 by the spring 511. This feature causes the sear 514 to return fully to the cocked position 524 in the event the trigger 558 is partially depressed, but then released before the catch 502 releases the draw string 501.

In another embodiment, a force necessary to overcome the friction at the interface 533 to release the catch 502 is preferably less than about 3.2%, and more preferably less than about 1.6% of the draw force to retain the draw string 501 to the drawn configuration. The draw force can optionally be measured as the force on the flexible tension member 585 when the string carrier 480 is in the drawn position (See FIG. 18A).

Turning back to FIGS. 17A and 17B, when in safe position 509 shoulder 520 on safety 522 retains the sear 514 in a cocked position 524 and the catch 502 in the closed position 504. Safety button 530 is used to move the safety 522 in direction 532 from the safe position 509 illustrated in FIGS. 17A and 17B to free position 553 (see FIG. 18B) with the shoulder 520 disengaged from the sear 514.

A dry fire lockout biasing force is applied by spring 540 to bias dry fire lockout 542 toward the catch 502. Distal end 544 of the dry fire lockout 542 engages the sear 514 in a lockout position 541 to prevent the sear 514 from releasing the catch 502. Even if the safety 522 is disengaged from the sear 514, the distal end 544 of the dry fire lockout 542 retains the sear 514 in the cocked position 524 to prevent the catch 502 from releasing the draw string 501.

FIG. 17C illustrates the string carrier 480 with the catch 502 removed for clarity. Nock 417 of the bolt 416 is engaged with the dry fire lockout 542 and rotated it in the direction 546. Distal end 544 of the dry fire lockout 542 is now in disengaged position 547 relative to the sear 514. Once the safety 522 is removed from the safe position 509 using the safety button 530, the crossbow 400 can be fired. In the illustrated embodiment, the nock 417 is a clip-on version that flexes to form a snap-fit engagement with the draw string 501. Only when a bolt 416 is fully engaged with the

11

draw string 501 will the dry fire lockout 542 be in the disengaged position 547 that permits the sear 514 to release the catch 502.

FIGS. 18A and 18B illustrate the relationship between the string carrier 480, the cocking mechanism 484, and the trigger assembly 550 that form string control assembly 551. The trigger assembly 550 is mounted in the stock 408, separate from the string carrier 480. Only when the string carrier 480 is fully retracted into the stock 408 is the trigger pawl 552 positioned adjacent to the sear 514. When the user is ready to fire the crossbow 400, the safety button 530 is moved in direction 532 to a free position 553 where the extension 515 is disengaged from the shoulder 520. When the trigger 558 is depressed the sear 514 rotating in direction 517 to a de-cocked position 557 and the catch 502 moves to the open position 505 to release the draw string 501.

As best illustrate in FIG. 18B, after firing the crossbow the sear 514 is in a de-cocked position 557 and the safety 522 is in the free position 553. The catch 502 retains the sear 514 in the de-cocked position 557 even though the spring 511 biases it toward the cocked position 524. In the de-cocked position 557 the sear 514 retains the dry fire lockout 542 in the disengaged position 547 even though the spring 540 biases it toward the lockout position 541. The extension 515 on the sear 514 is located in recess 521 on the safety 522.

To cock the crossbow 400 again the string carrier 480 is moved forward to location 483 (see FIG. 21A) into engagement with the draw string 501. Lower edge 503 of the catch 502 engages the draw string 501 and overcomes the force of spring 510 to automatically push the catch 502 to the closed position 504 (See FIG. 18A). Spring 511 automatically rotates the sear 514 back into the cocked position 524 so recess 512 formed interface 533 with the catch 502. Rotation of the sear 514 causes the extension 515 to slide along the surface of the recess 521 until it engages with the shoulder 520 on the safety 522 in the safe position 509. With the sear 514 back in the cocked position 524 (See FIG. 18A), the spring 540 biases dry fire lockout 542 to the lockout position 541 so the distal end 544 engages the sear 514 to prevent the catch 502 from releasing the draw string 501 (See FIG. 18A) until an arrow is inserted into the string carrier 480. Consequently, when the string carrier 480 is pushed into engagement with the draw string 501, the draw string 501 pushes the catch 502 from the open position 505 to the closed position 504 to automatically (i) couple the sear 514 with the catch 502 at the interface 533 to retain the catch 502 in the closed position 504, (ii) move the safety 522 to the safe position 509 coupled with the sear 514 to retain the sear 514 in the cocked position 524, and (iii) move the dry fire lockout 542 to the lockout position 541 to block the sear 514 from moving to the de-cocked position 557.

The cocking mechanism 484 includes a spool 560 with a flexible tension member, such as for example, a belt, a tape or webbing material 585, attached to pin 587 on the string carrier 480. As best illustrated in FIGS. 19 and 20, the cocking mechanism 484 includes drive shaft 564 with a pair of drive gears 566 meshed with gear teeth 568 on opposite sides of the spool 560. Consequently, the spool 560 is subject to equalize torque applied to the spool 560 during the cocking operation. Cocking handle 454 releasably attaches to either of exposed ends of pin 570 of the driver shaft 564.

A pair of pawls 572A, 572B ("572") include teeth 574 (see FIG. 20) that are biased into engage with the gear teeth 568. The pawls 572 are preferably offset $\frac{1}{2}$ the gear tooth 568 spacing so that when the teeth 574 of one pawl 572 are disengaged from the gear teeth 568, the teeth 574 on the other pawl 572 are positioned to engage the gear teeth 568.

12

Consequently, during winding of the spool 560, the teeth 574 on one of the pawls 572 are always positioned to engage with the gear teeth 568 on the spool. If the user inadvertently released the cocking handle 454 when the crossbow 400 is under tension, one of the pawls 572 is always in position to arrest rotation of the spool 560.

In operation, the user presses the release 576 to disengage the pawls 572 from the spool 560 and proceeds to rotate the cocking handle 454 to move the string carrier 480 in either direction 482 along the rail 402 to cock or de-cocking the crossbow 400. Alternatively, the crossbow 400 can be cocked without depressing the release 576, but the pawls 572 will make a clicking sound as they advance over the gear teeth 568.

FIGS. 21A and 21B illustrate the crossbow 400 in the released configuration 600. Draw string 501 is located adjacent down-range side 602 of the cams 440 in a reverse draw configuration 604. In the illustrated embodiment of the released configuration 600 the draw string 501 is adjacent stops 606 attached to power cable bracket 608.

Upper power cables 610A are attached to the power cable bracket 608 at upper attachment points 612A and to power cable attachments 462A on the cams 440 (see also FIG. 22A). Lower power cables 610B are attached to the power cable bracket 608 at lower attachment points 612B and to the power cable attachments 462B on the cams 440 (see also FIG. 22B).

In the illustrated embodiment, the attachment points 612A, 612B for the respective power cables 610 are located on opposite sides of the center rail 402. Consequently, the power cables 610 do not cross over the center rail 402. As used herein, "without crossover" refers to a cabling system in which power cables do not pass through a vertical plane bisecting the center rail 402.

As best illustrated in FIG. 21B, the upper and lower attachment points 612A, 612B on the power cable bracket 608 maintains gap 614 between the upper and lower power cables 610A, 610B greater than the gap at the axes of the cams 440. Consequently, the power cables 610A, 610B angle toward each other near the cams 440.

FIGS. 22A and 22B are upper and lower perspective views of the cams 440 with the cables 510, 610A, and 610B in the released configuration 600. The cams 440 are preferably symmetrical so only one of the cams 440 is illustrated. Upper power cables 610A are attached to power cable attachments 462A, wrap around the upper pivots 463A and then return toward the bow 400 to attach to the power cable bracket 608 (see FIG. 21A). The draw cable 501 is attached to the draw string mount 472 and then wraps almost completely around the cam 440 in the draw string journal 464 to the down range side 602.

FIGS. 23A and 23B illustrate the crossbow 400 in the drawn configuration 620. Draw string 501 extends from the down-range side 602 of the cams 440 in a reverse draw configuration 604. As best illustrated in FIG. 23B, the power cables 610A, 610B move away from the cams 440 as they wrap onto the upper and lower helical journals 460A, 460B. In the drawn configuration 620 the power cables 610A, 610B are generally parallel (compare the angled relationship in the released configuration 600 illustrated in FIG. 21B). The resulting gap 622 permits the power cable attachments 462 and pivot 463 to pass under the power cables 610 without contacting them (see also, FIGS. 24A and 24B) as the crossbow 400 moves between the released configuration 600 and the drawn configuration 620. As best illustrated in FIG. 24C, gaps 623 between surfaces 625 of the cams 440

13

and the power cables **610** is greater than height **627** of the power cable attachments **462** and the pivots **463**.

FIGS. **24A** and **24B** are upper and lower perspective views of the cams **440** with the cables **510**, **610A**, and **610B** in the drawn configuration **620**. The upper power cables **610A** wraps around the upper pivots **463A** and then onto the upper helical journal **460A**, before returning to the power cable bracket **608** (see FIG. **23A**). Similarly, the lower power cables **610B** wraps around the lower pivots **463B** and then onto the lower journal **460B**, before returning to the power cable bracket **608** (see FIG. **23A**). The draw cable **501** is attached to the draw string mount **472** unwraps almost completely from the draw string journal **464** of the cam **440** to the down range side **602**.

In the illustrated embodiment, the draw string journal **464** rotates between about 270 degrees and about 330 degrees, and more preferably from about 300 degrees to about 360 degrees, when the crossbow **400** is drawn from the released configuration **600** to the drawn configuration **620**. In another embodiment, the draw string journal **464** rotates more than 360 degrees (see FIG. **9A**).

FIGS. **25A** and **25B** illustrate an alternate string carrier **480A** for the crossbow **400** in accordance with an embodiment of the present disclosure. The string carrier **480A** is similar to the assembly illustrated in FIGS. **17A-17C**, so the same reference numbers are used where applicable.

FIG. **25A** illustrates the catch **502** is illustrated in a closed position **504**. The catch **502** is biased by spring **510** to rotate in direction **506** and retained in open position **505** (see FIG. **18B**). Absent an external force, the catch **502** automatically releases the draw string **501** (See FIG. **17A**). In the closed position **504** illustrated in FIG. **25A**, recess **512** on sear **514** engages with low friction device **513** on the catch **502** to retain the catch **502** in the closed position **504**. The sear **514** is biased by spring **519** to retain the catch **502** in the closed position **504**. The safety **522** operates as discussed in connection with FIGS. **17A-17C**.

Spring **540A** biases dry fire lockout **542A** toward the catch **502**. Distal end **544A** of the dry fire lockout **542A** engages the sear **514** in a lockout position **541** to prevent the sear **514** from releasing the catch **502**. Even if the safety **522** is disengaged from the sear **514**, the distal end **544A** of the dry fire lockout **542A** locks the sear **514** in the closed position **504** to prevent the catch **502** from releasing the draw string **501**.

As illustrated in FIG. **25B**, when the bolt **416** is positioned on the string carrier **480A** the rear portions or arms on the clip-onnock **417** extends past the draw string **501** (so a portion of the nock **417** is behind the draw string **501**) and engages with the portion **543A** on the dry fire lockout **542A**, causing the dry fire lockout **542A** to rotate in direction **546A** so that the distal end **544A** is disengaged from the sear **514**. In the illustrated embodiment, the portion **543A** is a protrusion or finger on the dry fire lockout **542A**. Only when a bolt **416** is fully engaged with the draw string **501** will the dry fire lockout **542A** permit the sear **514** to release the catch **502**.

In the illustrated embodiment, the portion **543A** on the dry fire lockout **542A** is positioned behind the draw string location **501A**. As used herein, the phrase "behind the draw string" refers to a region between a draw string and a proximal end of a crossbow. Conventional flat or half-moon nocks do not extend far enough rearward to reach the portion **543A** of the dry fire lockout **542A**, reducing the chance that non-approved arrows can be launched by the crossbow **400**.

FIGS. **25A** and **25B** illustrate elongated arrow capture recess **650** that retains rear portion **419** of the arrow **416** and the clip-onnock **417** engaged with the string carrier **480A** in

14

accordance with an embodiment of the present disclosure. The elongated arrow capture recess **650** extends along a direction of travel of an arrow launched from the crossbow **400**. The arrow capture recess **650** is offset above the rail **402** as is the rest **490** (see FIG. **14C**) so the arrow **416** is suspended above the rail **402** (see FIG. **13B**).

Upper roller **652** is located near the entrance of the arrow capture recess **650**. The upper roller **652** is configured to rotate in the direction of travel of the arrow **416** as it is launched. That is, the axis of rotation of the upper roller **652** is perpendicular to a longitudinal axis of the arrow **416**. The upper roller **652** is displaced within the slot in a direction generally perpendicular to the arrow **416**, while spring **654** biases the upper roller **652** in direction **656** against the arrow **416**. As best illustrated in FIG. **25C**, the arrow capture recess **650** extends rearward past the fingers **500** on catch **502**. The string carrier **480A** includes lower angled surfaces **658A**, **658B** ("658") and upper angled surfaces **660A**, **660B** ("660") configured to engage the arrow **416** around the perimeter of the rear portion.

In the illustrated embodiment, the clip-onnock **417** must be fully engaged with the draw string **510A** near the rear of the arrow capture recess **650** to disengage the dry fire lock out **542A**. In this configuration (see FIG. **25B**), the rear portion **419** of the arrow **416** is fully engaged with the arrow capture recess **650**, surrounded by the rigid structure of the string carrier **480A**.

In one embodiment, the lower angled surfaces **658** do not support the arrow **416** in the arrow capture recess **650** unless the clip-onnock **417** is used. In particular, the upper angled surfaces **660** prevent the nock **417** from rising upward when the crossbow **400** is fired, but the arrow **417** tends to slide downward off the lower angled surfaces **658** unless the clip-onnock **417** is fully engaged with the draw string **510A**.

By contrast, prior art crossbows typically include a leaf spring or other biasing structure to retain the arrow against the rail. These devices tend to break and are subject to tampering, which can compromise accuracy.

FIG. **26A** illustrates an alternate the cocking handle **720** with an integral clutch to prevent excessive torque on the cocking mechanism **484** and tension on the flexible tension member **585** in accordance with an embodiment of the present disclosure. As discussed in connection with FIG. **14D**, distal end **700** is configured to engage with drive shaft **564** and pins **570**. Center recess **702** receives the drive shaft **564** and the undercuts **704** engage with the pins **570** when the system is under tension. Consequently, when cocking or uncocking the crossbow **400** the tension in the system locks the pins **570** into the undercuts **704**. When tension in the system is removed, the cocking handle **454** can be rotated a few degrees and disengaged from the drive shaft **564**.

FIG. **26B** is an exploded view of the cocking handle **720** of FIG. **26A**. Distal end **700** contains a torque control mechanism **722**. Head **724** that engages with the drive shaft **564** is contained between a pair of opposing friction washers **726** and a pair of opposing notched washers **728**. Pins **730** couple the notched washers **728**. One or more spring washers **732**, such as for example Belleville washers, conical spring washers, and the like, maintain a compressive load on the head **724** to control the torque applied to the drive shaft **564**. In an alternate embodiment, the torque control mechanism **722** is located in the stock **408** between the drive shaft **564** and the spool **560**.

FIGS. **27A-27C** illustrates an alternate tunable arrow rest **750** in accordance with an embodiment of the present disclosure. The tunable arrow rest **750** includes housing **760** that is positioned just behind the pocket **426**. A pair of spring

15

loaded support rollers **752** are rotatably secured in slots **754** by pins **756**. The support rollers **752** rotate freely around the pins **756**. When compressed, the support rollers **752** can be independently displaced in directions **758**. Springs **764** (see FIG. 27B) bias the pins **756** and the support rollers **752** to the tops of the slots.

As best seen in FIG. 27B with the housing **760** removed, arrow rest **750** is mounted to distal end **776** of the center rail **402** by fasteners **762**. Each of the support rollers **752** is biased to the tops of the slots **754** by the springs **764**. Rotating member **766** is provided at the interface between the support rollers **752** and the springs **764** to reduce friction and permit the support rollers **752** to turn freely.

As best seen in FIGS. 27C and 27D the housing **760** includes enlarged openings **768** with diameters larger than the diameters of the fasteners **762**. Consequently, the position of the arrow rest **750** can be adjusted (i.e., tuned) in at three degrees of freedom—the Y-direction **770**, the Z-direction **772**, and roll **774** relative to the center rail **402**. FIG. 27D illustrates an arrow **412** with arrowhead **428** positioned on the support rollers **752** and the various degrees of freedom **770**, **772**, **774** available for tuning the arrow rest **750**.

Where a range of values is provided, it is understood that each intervening value, to the tenth of the unit of the lower limit unless the context clearly dictates otherwise, between the upper and lower limit of that range and any other stated or intervening value in that stated range is encompassed within this disclosure. The upper and lower limits of these smaller ranges which may independently be included in the smaller ranges is also encompassed within the disclosure, subject to any specifically excluded limit in the stated range. Where the stated range includes one or both of the limits, ranges excluding either both of those included limits are also included in the disclosure.

Unless defined otherwise, all technical and scientific terms used herein have the same meaning as commonly understood by one of ordinary skill in the art to which this disclosure belongs. Although any methods and materials similar or equivalent to those described herein can also be used in the practice or testing of the various methods and materials are now described. All patents and publications mentioned herein, including those cited in the Background of the application, are hereby incorporated by reference to disclose and described the methods and/or materials in connection with which the publications are cited.

The publications discussed herein are provided solely for their disclosure prior to the filing date of the present application. Nothing herein is to be construed as an admission that the present disclosure is not entitled to antedate such publication by virtue of prior invention. Further, the dates of publication provided may be different from the actual publication dates which may need to be independently confirmed.

Other embodiments are possible. Although the description above contains much specificity, these should not be construed as limiting the scope of the disclosure, but as merely providing illustrations of some of the presently preferred embodiments. It is also contemplated that various combinations or sub-combinations of the specific features and aspects of the embodiments may be made and still fall within the scope of this disclosure. It should be understood that various features and aspects of the disclosed embodiments can be combined with or substituted for one another in order to form varying modes disclosed. Thus, it is intended that the

16

scope of at least some of the present disclosure should not be limited by the particular disclosed embodiments described above.

Thus the scope of this disclosure should be determined by the appended claims and their legal equivalents. Therefore, it will be appreciated that the scope of the present disclosure fully encompasses other embodiments which may become obvious to those skilled in the art, and that the scope of the present disclosure is accordingly to be limited by nothing other than the appended claims, in which reference to an element in the singular is not intended to mean “one and only one” unless explicitly so stated, but rather “one or more.” All structural, chemical, and functional equivalents to the elements of the above-described preferred embodiment that are known to those of ordinary skill in the art are expressly incorporated herein by reference and are intended to be encompassed by the present claims. Moreover, it is not necessary for a device or method to address each and every problem sought to be solved by the present disclosure, for it to be encompassed by the present claims. Furthermore, no element, component, or method step in the present disclosure is intended to be dedicated to the public regardless of whether the element, component, or method step is explicitly recited in the claims.

What is claimed is:

1. A crossbow comprising:

first and second flexible limbs attached to a center rail; a first cam mounted to the first bow limb and rotatable around a first axis, the first cam comprising a first draw string journal having a first plane of rotation perpendicular to the first axis, a first upper power cable take-up journal extending in a direction perpendicular to the first plane of rotation of the first draw string journal, and a first lower power cable take-up journal extending in an opposite direction perpendicular to the first plane of rotation;

a second cam mounted to the second bow limb and rotatable around a second axis, the second cam comprising a second draw string journal having a second plane of rotation perpendicular to the second axis, a second upper power cable take-up journal extending in a direction perpendicular to the second plane of rotation of the second draw string journal, and a second lower power cable take-up journal extending in an opposite direction perpendicular to the second plane of rotation;

a draw string received in the string guide journals and secured to the first and second cams, wherein the draw string unwinds from the string guide journals as it translates from a released configuration to a drawn configuration; and

upper and lower power cables received in the upper and lower power cable take-up journals on each of the first and second cams;

wherein as the crossbow is drawn from the released configuration to the drawn configuration the upper and lower power cables wrap onto the respective upper and lower power cable take-up journals and are displaced along the first and second axes away from the first and second planes of rotation of the first and second draw string journals.

2. The crossbow of claim 1 wherein the upper and lower power cable take-up journals comprise helical journals that translates the upper and lower power cable away from the first and second cams along the first and second axes, respectively, as the crossbow is drawn from the released configuration to the drawn configuration.

17

3. The crossbow of claim 1 wherein the upper and lower power cable take-up journals comprise a width at least twice a width of the upper and lower power cables.

4. The crossbow of claim 1 wherein the first and second cams rotate between about 270 degrees to about 330 degrees when the crossbow is drawn from the released configuration to the drawn configuration.

5. The crossbow of claim 1 wherein the first and second cams rotate between about 300 degrees to about 360 degrees when the crossbow is drawn from the released configuration to the drawn configuration.

6. The crossbow of claim 1 wherein the first and second cams rotate more than about 360 degrees when the crossbow is drawn from the released configuration to the drawn configuration.

7. The crossbow of claim 1 wherein first ends of the upper and lower power cables are attached to power cable attachments extending above upper and lower surfaces of the first and second cams, respectively, wherein the power cable attachments pass under the respective upper and lower power cables as the crossbow moves between the released configuration and the drawn configuration.

8. The crossbow of claim 1 wherein second ends of the upper and lower power cables are connected to attachment points on respective sides of the center rail in a spaced apart configuration such that the upper and lower power cables are generally parallel to each other when the crossbow is in the drawn configuration.

9. The crossbow of claim 1 wherein the first and second cams are mounted to the first and second bow limbs, respectively, with first and second axes of rotation offset from distal tips of the bow limbs by about 1.0 inch to about 3.0 inches.

10. The crossbow of claim 1 wherein the upper and lower power cables do not cross over the center rail.

11. A crossbow comprising:

first and second flexible limbs attached to, and extending away from, a center rail;

a first cam mounted to the first flexible limb and rotatable around a first axis, the first cam comprising a first draw string journal having a first plane of rotation perpendicular to the first axis and a first helical power cable take-up journal extending in a direction perpendicular to the first plane of rotation of the first draw string journal;

a second cam mounted to the second flexible limb and rotatable around a second axis, the second cam comprising a second draw string journal having a second plane of rotation perpendicular to the second axis and a second helical power cable take-up journal extending in a direction perpendicular to the second plane of rotation of the second draw string journal;

a draw string received in the first and second draw string journals and secured to the first and second cams, wherein the draw string unwinds from the first and second draw string journals as it translates between a released configuration to a drawn configuration;

power cables received in the first and second helical power cable take-up journals; and

wherein the radial displacement of the first and second cams as the draw string translates between the released configuration and the drawn configuration is greater than 270 degrees.

12. The crossbow of claim 11 wherein as the crossbow is drawn from the released configuration to the drawn con-

18

figuration the power cables wrap onto the respective first and second helical power cable take-up journals and are displaced along the first and second axes away from the respective first and second planes of rotation of the first and second cams.

13. The crossbow of claim 11 wherein first ends of the power cables are attached to power cable attachments located on surfaces of the first and second cams, respectively, and the power cable attachments pass under the respective power cables as the crossbow moves between the released configuration and the drawn configuration.

14. The crossbow of claim 11 wherein the power cables do not cross over the center rail.

15. The crossbow of claim 11 wherein the first and second helical power cable take-up journals comprise upper and lower helical power cable take-up journals on each of the first and second cams extending in a direction perpendicular to the respective plane of rotation of the first and second cams.

16. The crossbow of claim 11 wherein the first and second cams rotate between about 270 degrees to about 360 degrees when the crossbow is drawn from the released configuration to the drawn configuration.

17. The crossbow of claim 11 wherein the first and second cams rotate more than about 360 degrees when the crossbow is drawn from the released configuration to the drawn configuration.

18. The crossbow of claim 11 wherein the first and second cams are mounted to the first and second bow limbs, respectively, with first and second axes of rotation offset from distal tips of the bow limbs by about 1.0 inch to about 3.0 inches.

19. A crossbow comprising:

first and second flexible limbs attached to, and extending away from, a center rail;

a first cam mounted to the first flexible limb and rotatable around a first axis, the first cam comprising a first draw string journal having a first plane of rotation perpendicular to the first axis and a first power cable take-up journal extending in a direction perpendicular to the first plane of rotation of the first draw string journal;

a second cam mounted to the second flexible limb and rotatable around a second axis, the second cam comprising a second draw string journal having a second plane of rotation perpendicular to the second axis and a second power cable take-up journal extending in a direction perpendicular to the second plane of rotation of the second draw string journal;

a draw string received in the first and second draw string journals and secured to the first and second cams, wherein the draw string unwinds from the first and second draw string journals as it translates between a released configuration to a drawn configuration; and

power cables received in the first and second power cable take-up journals, wherein the first and second power cable take-up journals comprise a width at least twice a width of the power cables;

wherein as the crossbow is drawn from the released configuration to the drawn configuration the power cables wrap onto the respective first and second power cable take-up journals in a helical configuration and are displaced along the first and second axes away from the first and second planes of rotation of the first and second draw string journals.

* * * * *

UNITED STATES PATENT AND TRADEMARK OFFICE
CERTIFICATE OF CORRECTION

PATENT NO. : 9,494,379 B2
APPLICATION NO. : 15/098537
DATED : November 15, 2016
INVENTOR(S) : Craig Yehle

Page 1 of 1

It is certified that error appears in the above-identified patent and that said Letters Patent is hereby corrected as shown below:

Column 16, Line 29, delete the word “bow” and insert the word --flexible--.

Column 16, Line 38, delete the word “bow” and insert the word --flexible--.

Column 16, Line 46, delete the phrase “string guide” and insert the phrase --draw string--.

Column 16, Line 49, delete the phrase “string guide” and insert the phrase --draw string--.

Column 17, Line 30, delete the word “bow” and insert the word --flexible--.

Column 17, Line 32, delete the word “bow” and insert the word --flexible--.

Column 18, Line 29, delete the word “bow” and insert the word --flexible--.

Column 18, Line 31, delete the word “bow” and insert the word --flexible--.

Signed and Sealed this
First Day of January, 2019



Andrei Iancu
Director of the United States Patent and Trademark Office