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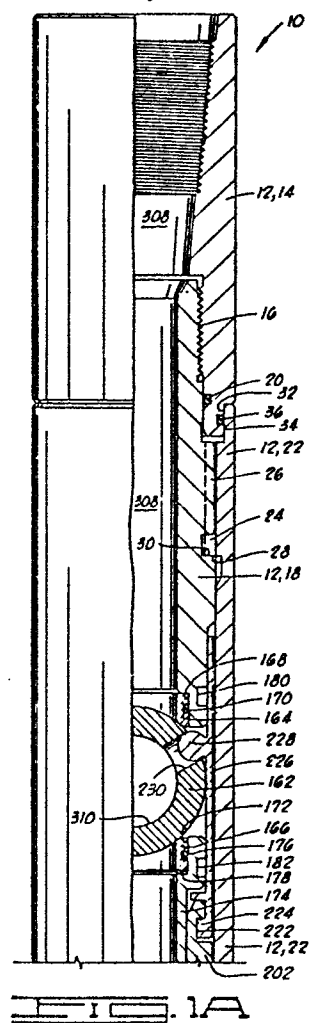
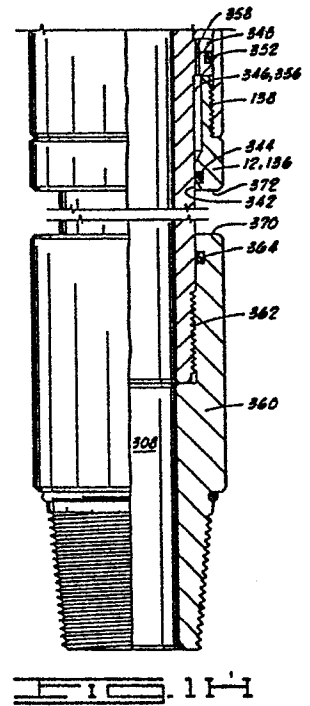
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54 **Downhole tool.**

57 A downhole tool apparatus has a housing (12) with a compression chamber (232) defined therein. A fill passage (234) is disposed through the housing for placing the compression chamber in open flow fluid communication with a well annulus (236) so that well fluid may flow into the compression chamber as the apparatus is lowered into a well. An isolation valve (242) selectively closes the fill passage to trap well fluid in the compression chamber. An operating element (162) is operated by an actuating piston (140) slidably disposed in the housing. A first side of the actuating piston is in fluid pressure communication with the compression chamber so that a volume of the compression chamber is decreased when the actuating piston moves between first and second position thereof relative to the housing. An injection passage (286) is preferably provided for injecting pressurized gas into the compression chamber at a location within the compression chamber such that the injected gas will directly contact at a gas-well fluid interface an upper surface of well fluid that flows into the compression chamber. The compression chamber is preferably primarily defined by an elongate diametrically irregular annular space, and the gas-well fluid interface moves upward past a number of irregular diameters of this diametrically irregular elongated space, as the apparatus is lowered into a well.



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DOWNHOLE TOOL

The present invention relates to a downhole tool, more particularly to a downhole tool utilizing a compression chamber within which is trapped a compressible fluid to provide a compressible fluid spring acting to restore an actuating piston to an original position thereof.

It is well known in the art that downhole tools such as testing valves, circulating valves and sampler, can be operated by varying the pressure of fluid in a well annulus and applying that pressure to a differential pressure piston within the tool.

The predominant method of creating the differential pressure across the differential pressure piston has been to isolate a volume of fluid within the tool at a fixed reference pressure. Such a fixed reference pressure has been provided in a number of different ways. Further, these prior art tools have often provided a volume of fluid, either liquid or gas, through which this reference pressure is transmitted. Sometimes this volume of fluid provides a compressible fluid spring which initially stores energy when the differential area piston compresses that fluid, and which then aids in returning the differential area piston to its initial position.

One manner of providing a fixed reference pressure is by providing an essentially empty sealed chamber on the low pressure side of the power piston, which chamber is

merely filled with air at the ambient pressure at which the tool was assembled. Such a device is shown, for example, in U.S. Patent No. 4,076,077. However, this type of device does not balance hydrostatic annulus
5 pressure across the power piston as the tool is run into the well, and nor does it provide a fluid spring to aid in return of the power piston.

Another approach has been to provide a chamber on the low pressure side of the piston, and fill that
10 chamber with a charge of inert gas such as nitrogen. Then, when the annulus pressure overcomes the gas pressure, the power piston is moved by that pressure differential, and the gas is compressed allowing the movement of the power piston. Such a device is shown, for example, in
15 U.S. Patent No. 3,664,415. However, this type of device does not balance hydrostatic annulus pressure across the power piston as the tool is run into the well.

Another approach has been to use a charge of inert gas as described above, in combination with means
20 for supplementing the gas pressure with the hydrostatic pressure of the fluid in the annulus contained between the well bore and the test string, as the test string is lowered into the well. Such a device is shown, for example, in U.S. Patent No. 3,856,085. When a tool of this type
25 has been lowered to the desired position in the well, the inert gas pressure is supplemented by the amount of the hydrostatic pressure in the well at that depth. Then, an isolation valve is closed which traps in the tool a volume of well annulus fluid at a pressure substantially
30 equal to the hydrostatic pressure in the well annulus at that depth. Once the isolation valve has closed, the reference pressure provided by the inert gas is no longer effected by further increases in well annulus pressure. Then, well annulus pressure may be increased to create
35 a pressure differential across the power piston to

actuate the tool. Thus, the U.S. 3856085 device utilizes the energy stored in compression of the nitrogen gas within a chamber to assist in returning the power piston to its upper position.

5 Also, rather than utilize a compressible inert gas such as nitrogen within such tools, it has been proposed to use a large volume of a somewhat compressible liquid such as silicone oil, as a compressible fluid spring on the low pressure side of the tool. Such a device
10 is seen, for example, in U.S. Patent No. 4,109,724.

Other devices utilizing a large volume of trapped silicone oil as a compressible fluid spring are shown in U.S. Patents Nos. 4,444,268 and 4,448,254. In each of these devices, the silicone oil pressure is
15 supplemented by well annulus pressure as the tool is lowered into the well.

One recent device which has not relied upon either a large volume of compressible liquid or a volume of compressible gas is shown in U.S. Patent No. 4,341,266.
20 This is a trapped reference pressure device which uses a system of floating pistons and a differential pressure valve to accomplish actuation of the tool. The reference pressure is trapped by a valve which shuts upon the initial pressurizing up of the well annulus after the packer
25 is set. This tool does balance hydrostatic pressure across its various differential pressure components as it is run into the well. The power piston of this device is returned to its original position by a mechanical coil compression spring without the aid of any compressed volume of fluid.

30 Another relatively recent development is shown in U.S. Patent No. 4,113,012. This device utilizes fluid flow restrictors to create a time delay in any communication of changes in well annulus pressure to the lower side of its power piston. During this time delay, the power
35 piston moves from a first position to a second position.

The particular tool disclosed utilizes a compressed nitrogen gas chamber in combination with a floating shoe which transmits the pressure from the compressed nitrogen gas to a relatively non-compressible liquid filled chamber.

5 This liquid-filled chamber is communicated with the well annulus through pressurizing and depressurizing passages, each of which includes one of the fluid flow restrictors plus a back pressure check valve. Hydrostatic pressure is balanced across the power piston as the tool is run
10 into the well, except for the relatively small differential created by the back pressure check valve in the pressurizing passage.

It is apparent from the numerous examples set forth above that it is well known in the prior art to
15 create a trapped reference pressure within a tool by communicating a chamber within the tool with the well annulus, and then isolating that chamber to trap the reference pressure within the tool. In combination with that concept, a number of these prior tools have also
20 utilized a volume of compressible inert gas or of a relatively compressible liquid such as silicone oil contained within the tool to act as a fluid spring to aid in returning the power piston to its initial position. This compressed gas or silicone oil generally is separated from the
25 trapped well fluid providing the reference pressure by a floating piston so that the trapped well fluid and the compressed gas or silicone oil are always at the same pressure.

Those prior art devices discussed above which
30 do utilize a compressible fluid spring to aid in returning the power piston to its original position rely upon the compressibility of the compressed inert gas or silicone oil, and not upon compressibility of the well fluid itself which may be trapped within the tool.

35 Those tool utilizing either inert gas or silicone

oil suffer from the inherent disadvantage that these materials are not always readily available, particularly at very remote well sites. Additionally, when using inert gas, there are inherent dangers due to the high pressures at which the inert gas must be initially placed within the tool while it is still above the ground and personnel are in the immediate vicinity of the tool; for example, when using nitrogen gas the initial pressures typically used have been in the range of 2000 to 8000 psi (13.8 to 55.1 MPa).

The present invention provides a tool which provides both a trapped reference pressure and a trapped fluid spring, without the use of either an initially highly pressurized inert gas or of silicone oil.

According to the present invention, there is provided a downhole tool apparatus comprising: a housing having a compression chamber defined therein and having fill passage means disposed through said housing for placing said compression chamber in open flow fluid communication with a well annulus exterior of said housing so that well fluid may flow into said compression chamber as said apparatus is lowered into a well, the compression chamber containing air or gas adapted to contact directly the well fluid at an air or gas/well fluid interface within the chamber; isolation valve means for selectively closing said fill passage means to trap said air or gas and well fluid in said compression chamber; an operating element disposed in said housing; an actuating piston means slidably disposed in said housing, said actuating piston means being operably associated with said operating element; wherein movement of said actuating piston means between first and second positions thereof decreases the volume of said compression chamber. Preferably, the volume of said compression chamber is sufficiently large that an equal volume of water containing no absorbed gases would

have a volume decrease when subjected to a pressure increase equivalent to a predetermined operating pressure of said actuating piston means, at least as great as the decrease in volume of said compression chamber when said
5 actuating piston means is moved between its first and second positions. The apparatus also preferably comprises means for injecting a gas (such as air for example) into the compression chamber as, or before, the apparatus is lowered into the well.

10 The downhole tool of the present invention has a compression chamber which contains both gas and well fluid which directly contact each other at a gas-well fluid interface within the chamber. In those embodiments of the invention wherein compressibility of the gas is
15 primarily relied upon to provide a compressible fluid spring, the gas need not initially be highly pressurized. Instead, only a relatively low pressure need initially be applied to the gas, and this relatively low pressure pressurized gas can be provided from a typical rig air
20 system of a drilling rig which provides compressed air at a pressure in the range of 100 to 140 psi (690 to 965 kPa).

 In one embodiment of the invention, an injection means is provided for admitting pressurized gas into
25 the compression chamber at a location within the compression chamber such that the injected gas will directly contact at a gas-well fluid interface an upper surface of any well fluid which flows into the compression chamber.

 A majority of the total volume of the compression
30 chamber is preferably contained in a diametrically irregular elongate annular space defined between a plurality of interconnected tubular outer housing sections and a plurality of interconnected tubular inner housing sections. As the well fluid flows through the fill passage into the
35 compression chamber, when the tool is being lowered into

place within a well, the gas-well fluid interface moves upward past a plurality of diametrically irregular surfaces defining the diametrically irregular elongate annular space of the compression chamber.

5 The downhole tool apparatus of the invention has a trapped reference pressure and a compressible fluid spring for aiding in the return of the actuating piston to its original position, yet does not require the use of an initially highly pressurized volume of inert gas or
10 of silicone oil. Instead, the apparatus relies on the compressibility of a relatively large volume of initially relatively lowly pressurized gas and/or a relatively large volume of well fluid obtained from the well annulus.

 In a preferred embodiment, the compression chamber
15 contains a volume of well fluid sufficient that, when the actuating piston means moves between first and second positions to operate the operating element, the volume of well fluid in the compression chamber changes by a first amount greater than one-half of the amount by which the
20 volume of said compression chamber decreases. Preferably, in use, the compression chamber contains a mass of air substantially equal to the mass of air required to fill it at atmospheric pressure and ambient temperature at a surface of said well, and the pressure of the mass of air
25 is at least as great as the hydrostatic pressure of well fluid in the well at a depth at which the apparatus is located within the well.

 In order that the invention may be more fully understood, two embodiments of tool of the invention will
30 now be described, by way of example only, with reference to the accompanying drawings, wherein:

 FIGURES 1A-1H comprise an elevation right side only sectioned view of a first embodiment of downhole apparatus of the present invention.

35 FIGURE 2 is a sectional view taken along line

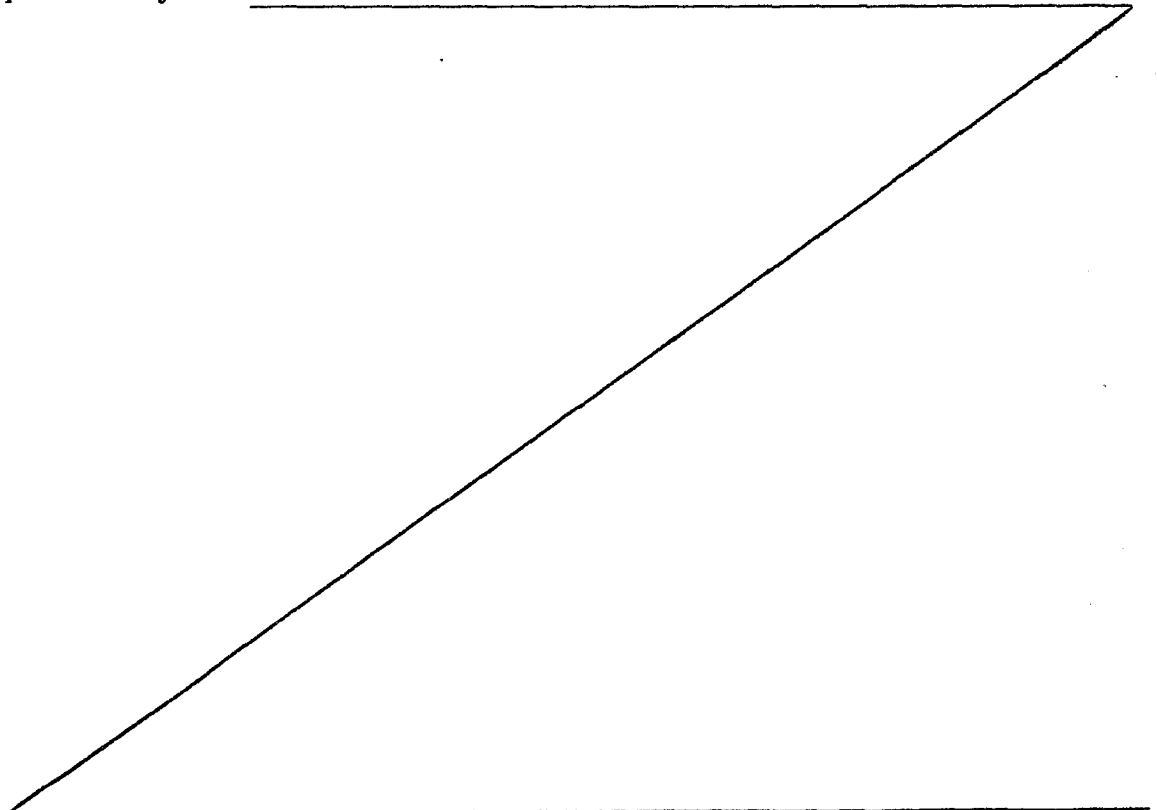
2-2 of Figure 1D which shows an injection valve means. Figure 2 has been rotated 90° clockwise to aid in fitting the same on the sheet of drawings.

FIGURE 3 is a schematic elevation view of the downhole tool apparatus of Figures 1A-1H showing the same partially lowered into place within a well.

FIGURES 4F-4I comprise an elevation right side only sectioned view of the lower portion of a second embodiment of downhole apparatus of the present invention. The upper portion of this alternative embodiment is similar to the structure shown in Figures 1A-1E.

Referring now to the drawings, and particularly to Figures 1A-1H, the downhole tool apparatus of the present invention is shown and generally designated by the numeral 10. The downhole tool apparatus 10 may also be referred to as a well tester valve apparatus 10.

The apparatus 10 includes a housing generally designated by the numeral 12. The housing 12 includes a plurality of _____



threadedly connected tubular members, the uppermost one of which is an upper adapter 14.

Threadedly connected at 16 to upper adapter 14 is an upper seat housing 18 of housing 12, with a seal being provided therebetween by resilient O-ring seal means 20.

Housing 12 includes a ball valve housing 22 which has a plurality of radially inward directed splines 24 which mesh with a plurality of radially outward directed splines 26 of upper seat housing 18.

Lower ends 28 of the splines 24 abut an upward facing annular shoulder 30 of upper seat housing 18 to hold the ball valve housing 22 longitudinally in place relative to the upper adapter 14 and the upper seat housing 18.

An upper inner cylindrical surface 32 of ball valve housing 22 is closely received about a cylindrical outer lower surface 34 of upper adapter 14 with a seal being provided therebetween by resilient O-ring seal means 36.

A first housing adapter 38 of housing 12 has its upper end threadedly connected at 40 to ball valve housing 22, with a seal being provided therebetween by resilient O-ring seal means 42. A lower end of first housing adapter 38 is threadedly connected at 44 to a spring chamber housing 46 of housing 12, with a seal being provided therebetween by resilient O-ring seal means 48.

A lower end of spring chamber housing 46 is threadedly connected at 48 to an air injection adapter 50 of housing 12 with a

seal being provided therebetween by resilient O-ring seal means 52.

5 A lower end of air injection adapter 50 is threadedly connected at 54 to an outer lubricant chamber housing 56 of housing 12 with a seal being provided therebetween by resilient O-ring seal means 58.

10 Also, air injection adapter 50 has an upper inner lubricant chamber housing 60 of housing 12 threadedly connected to an internal thread 62 thereof with a seal being provided therebetween by resilient O-ring seal means 64.

Outer lubricant chamber housing 56 has a lubricant chamber adapter 66 of housing 12 threadedly connected thereto at 68 with a seal being provided therebetween by resilient O-ring seal means 70.

15 Upper inner lubricant chamber housing 60 has a lower inner lubricant chamber housing 72 threadedly connected thereto at 74 with a seal provided therebetween by resilient O-ring seal means 76.

20 Lubricant chamber adapter 66 has a first outer compression housing section 78 threadedly connected thereto at 80 with a seal being provided therebetween by resilient O-ring seal means 82.

25 The first outer compression housing section 78 includes an upper outer compression housing adapter 84 threadedly connected at 86 to an outer compression housing 88 which has its lower end threadedly connected at 90 to a lower outer compression housing adapter 92 of housing means 12.

The upper outer compression housing adapter 84 and the outer compression housing 88 of first outer compression housing section 78 are further connected by a circumferential weld 94. Also, outer compression housing 88 and lower outer compression housing adapter 92 of first outer compression housing section 78 are further connected by a circumferential weld 96 therebetween.

Thus, the upper outer compression housing adapter 84, the outer compression housing 88, and the lower outer compression housing adapter 92 are all permanently connected together by welds 94 and 96 to form the first outer compression housing section 78.

Concentrically received within the first outer compression housing section 78 of housing 12 is a first inner compression housing section 98 of housing 12.

The first inner compression housing section 98 includes an upper inner compression housing adapter 100 and an inner compression housing 102 threadedly connected together at 104 and further connected by a circumferential weld 106 therebetween.

A plurality of notches 105 are disposed in a lower outer edge of upper inner compression housing adapter 100 to allow fluid flow, through notches 105, between upper inner compression housing adapter 100 and an upward facing shoulder 107 of upper outer compression housing adapter 84.

Upper inner compression housing adapter 100 has an upper inner bore 108 within which is closely received an external

cylindrical surface 110 of lower inner lubricant chamber housing 72 with a seal being provided therebetween by resilient O-ring seal means 112.

Housing 12 further includes a tube receiving adapter 114
5 threadedly connected to a lower end of lower outer compression housing adapter 92 at 116 with a seal being provided therebetween by resilient O-ring seal means 118. Tube receiving adapter 114 has an upper inner cylindrical bore 120 within which is closely received a cylindrical outer surface 122 of a lower end of inner
10 compression housing 102 with a seal being provided therebetween by resilient O-ring seal means 124.

In FIGS. 1E-1G, only a single outer compression housing section 78 and a single inner compression housing section 98 have been shown. Preferably, the outer and inner compression housing
15 sections 78 and 98 each have a length of approximately ten feet. Preferably, the apparatus 10 includes a plurality of interconnected outer compression housing sections such as 78, and a complimentary plurality of interconnected inner compression housing sections 98, as is schematically illustrated in FIG. 3.

20 The manner in which this is accomplished will be readily apparent in view of the fact that the outside diameter of outer surface 122 of the inner compression housing 102 is the same as the outside diameter of outer surface 110 of lower inner lubricant chamber housing 72. Thus, a plurality of inner compression
25 housing sections 98 may be interconnected merely by sliding the

lower end of each of the inner compression housings such as 102 into the bore such as 108 of each of the upper inner compression housing adapters such as 100.

Similarly, additional outer compression housing sections 78
5 may be threadedly connected together since the internal threads at threaded connection 80 of upper outer compression housing adapter 84 are complimentary with the external threads at threaded connection 116 of the lower outer compression housing adapter 92.

10 The housing 12 also includes a valve adapter 126 connected to the lower end of tube receiving adapter 114 at threaded connection 128 with a seal being provided therebetween by resilient O-ring seal means 130.

A bypass housing 132 of housing 12 is connected to the lower
15 end of valve adapter 126 at threaded connection 134.

A lower housing shoe 136 of housing 12 is connected to a lower end of bypass housing 132 at threaded connection 138.

An actuating piston means 140, which may also be referred to as a power piston means 140, is slidably disposed within a bore
20 142 of spring chamber housing 46 of housing 12 with a seal being provided therebetween by resilient O-ring piston seal 144.

In the embodiment illustrated in FIG. 1C, the actuating piston 140 is integrally formed on the upper end of a lower actuating mandrel 146.

25 The lower actuating mandrel 146 has a cylindrical outer surface 148 of a lower end thereof closely and slidably received

within a bore 150 of air injection adapter 50 of housing 12, with a pair of seals being provided therebetween by resilient O-ring seal means 152 and 154.

5 An inner annular groove 156 is disposed in bore 150 of air injection adapter 50 and is communicated with an exterior surface 158 of air injection adapter 50 by an oblique port shown in dashed lines and designated by the numeral 160. The port 160 and annular groove 156 prevent any hydraulic blockage of movement of lower actuating mandrel 146 within the bore 150.

10 An operating element 162, which may also be described as a spherical full opening ball valve 162, is disposed within the housing 12 between upper and lower annular seats 164 and 166, respectively.

Upper seat 164 is received within a lower inner bore 168 of upper seat housing 18 with a seal being provided therebetween by resilient O-ring seal means 170.

Lower seat 166 is received within an upper inner bore 172 of a lower seat holder 174 with a seal being provided therebetween by resilient O-ring seal means 176. A single Belleville spring 178 biases lower seat 166 upward relative to lower seat holder 174.

Lower seat holder 174 is held in place relative to upper seat holder 118 by a plurality of C-clamps (not shown) which span grooves 180 and 182 in upper seat holder 18 and lower seat holder 174, respectively.

The actuating piston means 140 and the ball valve 162 are operably related so that the ball valve 162 is moved from its first closed position illustrated in FIG. 1A, corresponding to the first uppermost position of actuating piston 140 relative to housing 12 seen in FIG. 1C, by movement of actuating piston 140 downward to a second lower position corresponding to a second open position of ball valve 162. This is accomplished as follows.

The actuating piston means 140 is connected to an upper actuating mandrel 184 at threaded connection 186. A seal is provided therebetween by O-ring 187.

Upper actuating mandrel 184 includes a plurality of radially outward directed splines 183 which are interlocked with a plurality of radially inward directed splines 185 of first housing adapter 38 so as to permit relative longitudinal motion therebetween while preventing relative rotational motion therebetween.

Upper actuating mandrel 184 has a retaining collar 188 threadedly connected to the upper end thereof at threaded connection 190. Retaining collar 188 is longitudinally located between a downward facing shoulder 200 of an actuating sleeve 202 and an upper end 204 of an actuating collar 206. The actuating collar 206 and actuating sleeve 202 are connected together at threaded connection 208.

An upper external cylindrical surface 210 of upper actuating mandrel 184 is closely received within an inner bore 212 of

actuating collar 206 with a seal being provided therebetween by resilient O-ring seal means 214.

5 An external cylindrical surface 216 of actuating collar 206 is closely and slidably received within a cylindrical inner surface 218 of ball valve housing 22 with a seal being provided therebetween by resilient O-ring sliding seal means 220.

The actuating sleeve 202 has a radially outer annular groove 222 disposed therein within which is received a radially inward directed flange 224 of a first actuating arm 226.

10 Actuating arm 226 has a lug 228 which is received within an eccentric radial bore 230 of spherical ball valve 162.

A second actuating arm (not shown) is similarly constructed and is peripherally spaced from the first actuating arm 226, so that longitudinal movement of actuating piston means 140 within
15 the housing 12 is transmitted through the lugs such as 228 to the spherical ball valve member 162 to rotate the spherical ball valve member 162 from its closed first position to an open second position as the actuating piston 140 moves downward within the housing 12 from its initial position shown in FIG. 1C.

20 The housing 12 has an elongated compression chamber 232 defined therein. Housing 12 also has a fill passage means 234 disposed therethrough for placing the compression chamber 232 in open flow fluid communication with a well annulus 236 (see FIG. 3) exterior of the housing 12 so that well fluid may flow into
25 the compression chamber 232 as the apparatus 10 is lowered into a well such as the one defined by casing 238 in FIG. 3.

By the term "open flow fluid communication" as used herein, it is meant that fluid can actually flow from one of the two designated points to the other. This is to be distinguished from the broader, more general terms such as "fluid communication" or
5 "fluid pressure communication" used elsewhere herein which only require that fluid pressure can be transmitted between the two designated points, but do not require that it be possible for fluid to actually flow between the two designated points.

An isolation valve means 240 is defined on a valve sleeve 242
10 and provides a means for selectively closing the fill passage means 234 and for thereby trapping well fluid within the compression chamber 232.

The compression chamber 232 includes a main compression chamber portion 244 defined between the outer compression housing
15 sections 78 and the inner compression housing sections 98.

When a plurality of outer compression housing sections 78 and a plurality of inner compression housing sections 98 are utilized as illustrated in FIG. 3, the main compression chamber portion 244 includes the annular space defined between all of the outer
20 compression housing sections 78 and all of the inner compression housing sections 98. It will be appreciated in viewing FIGS. 1E-1G that the main compression chamber portion 244 can be described as a diametrically irregular annular space 244 located between the plurality of interconnected tubular outer housing
25 sections 78 and the plurality of interconnected tubular inner housing sections 98.

Compression chamber 232 extends from the main compression chamber portion 244 upward through annular spaces 246, 248, and 250 to a lubricant chamber 252. Lubricant chamber 252 is communicated through a plurality of longitudinal passageways such as
5 254 disposed longitudinally through air injection adapter 50 with an annular spring chamber 256.

A coil compression spring 258 is disposed in spring chamber 256.

An annular floating piston 260 is disposed in lubricant
10 chamber 252 and has a plurality of radially inner seals 262 and 264 which seal against upper inner lubricant chamber housing 60, and a plurality of outer seals 266 and 268 which seal against outer lubricant chamber housing 56.

The purpose of annular floating piston 260 is to permit
15 spring chamber 256 to be filled with a non-corrosive fluid such as lubricating oil to protect the coil spring 258. The lubricating chamber 252 is filled with lubricating oil above annular piston 260, and will be filled with well fluid and/or possibly some entrapped air below the annular piston 260.

20 In the event that the coil compression spring 258 is eliminated or is constructed in such a manner that it can withstand the environment of the particular well fluid involved, the annular floating piston 260 may be eliminated.

The compression chamber 232 also extends downward from main
25 compression chamber 244 through a plurality of longitudinal

passageways 270 disposed through tube receiving adapter 114 to an annular cavity 272 defined between valve sleeve 242 and valve adapter 126.

5 The actuating piston means 140 is a differential area piston means 140 having its lower side 274 communicated with compression chamber 232. Lower side 274 may also be referred to as a first side or as a low pressure side of actuating piston means 140.

10 Housing 12 has a power passage means 276 disposed therethrough for communicating the well annulus 236 exterior of the housing 12 with an upper side 278 of actuating piston means 140. The upper side 278 of actuating piston means 140 may also be referred to as a second side or a high pressure side of actuating piston means 140.

15 The power passage means 278 includes a power port 280 and an annular cavity 282 defined between upper actuating mandrel 184 and spring chamber housing 46.

20 The actuating piston means 140 will move downward relative to housing 12 from its first position shown in FIG. 1C in response to increases in pressure in well annulus 236 relative to pressure within compression chamber 232 as is further described below.

The coil spring 258 previously mentioned can generally be described as a resilient mechanical spring biasing means 258 for biasing the actuating piston means 140 from its lower second position back toward its upper first position shown in FIG. 1C.

25 The coil compression spring 258 is disposed within spring chamber portion 256 of compression chamber 232 between the lower

side 274 of actuating piston means 140 and an upper end 284 of air injection adapter 50.

As is further explained below with regard to the manner of operation of the present invention, it is sometimes desirable to
5 inject a compressed gas into the compression chamber 232.

This is accomplished through an air injection passage means 286 disposed through the housing 12 and best seen in FIG. 2.

FIG. 2 is a section view taken along line 2-2 of FIG. 1D and it illustrates the manner in which the air injection passage
10 means 286 is communicated with longitudinal passageway 254 of compression chamber 232. FIG. 2 has been rotated 90° clockwise from the manner in which it would normally be shown in a section of FIG. 1D, in order to more easily fit the same on the sheet of drawings.

15 Air injection passage means 286 includes a threaded inlet 288, which is shown in FIG. 2 as being temporarily closed by a threaded plug 290.

As shown in FIG. 3, when it is desired to inject compressed air into the compression passage 232, an air line 292 from a
20 source of rig air 294 is connected to threaded inlet 288 so as to communicate the source of rig air 294 with the air injection passage means 286.

A needle valve 296 is disposed in a valve carrier body 298 which itself is threadedly engaged at 300 with a threaded bore
25 302 of air injection adapter 50.

In FIG. 2, the needle valve 296 is shown in a closed position where it sealingly engages a short portion 304 of air injection passage means 286.

To open the air injection passage means 286 in order to permit air to flow from the source of rig air 294 into the compression passage 232, the valve carrier body 298 is rotated through use of a wrench inserted in socket 306 thereof so as to back off the threaded connection 300 and move needle valve 296 out of engagement with the short portion 304 of air injection passage means 286. The manner of use of air injection passage means 286 is further described below in the section headed "Second Mode Of Operation Of The Embodiment Of FIGS. 1A-1H".

The housing 12 has a central flow passage 308 disposed therethrough. The ball valve member 162 is disposed within the flow passage 308 and, when in its closed position as illustrated in FIG. 1A, closes flow passage 308 to prevent fluid flow therethrough, and when in its second open position, has its ball valve bore 310 aligned with flow passage 308 to allow fluid flow through the flow passage 308.

The housing 12 has a bypass passage means 312 (see FIG. 1G) disposed therethrough for communicating the flow passage 308 of housing 12 with the well annulus 236 exterior of the housing 12.

The valve sleeve 242 includes a bypass valve portion 314 for selectively closing the bypass passage means 312.

The bypass passage means 312 includes a bypass port 316 disposed radially through bypass housing 132, an annular cavity

318 defined between valve sleeve 242 and bypass housing 132, and a bypass valve port 320 disposed through valve sleeve 242.

The valve sleeve 242 has an upper cylindrical outer surface 322 which is closely and slidably received within a lower inner bore 324 of tube receiving adapter 114 and a bore 326 of valve adapter 126.

An O-ring seal 328 is disposed in a complimentary groove of bore 324 and provides a sliding seal between valve mandrel 242 and bore 324 above the annular cavity 272.

10 An O-ring seal 330 is disposed within a complimentary groove in bore 326 of valve adapter 126 and provides a sliding seal between valve mandrel 242 and bore 326 below annular cavity 272.

A pair of O-ring seals 332 and 334 are disposed in complimentary grooves in a lower portion of bore 326 of valve adapter 126 just above a lower end thereof.

The valve mandrel 242 further includes a lower enlarged outer diameter cylindrical surface 336 which is closely slidably received within a reduced inner diameter bore 338 of bypass housing 132 with a sliding seal being provided therebetween by resilient O-ring seal means 340. Cylindrical outer surface 336 is also closely slidably received within a bore 342 of lower housing shoe 136 with a sliding seal being provided therebetween by resilient O-ring seal means 344.

20 A metering chamber 346 is defined between cylindrical outer surface 336 on the inside and bypass housing 132 and lower

housing shoe 136 on the outside, with its upper and lower extremities being defined by seals 340 and 344, respectively.

5 A metering piston 348 is disposed on valve mandrel 242 and is closely slidably received within an inner bore 350 of bypass housing 132 with a seal being provided therebetween by resilient O-ring piston seal 352.

The metering piston 348 divides metering chamber 346 into an upper portion 354 and a lower portion 356.

10 Metering chamber 346 is filled with a suitable fluid such as oil. Metering piston 348 has a fluid flow restricting orifice 358 disposed therethrough communicating upper and lower portions 354 and 356 of metering chamber 346.

15 The metering piston 348 with its orifice 358 therethrough impedes movement of valve mandrel 242 relative to housing 12. For valve mandrel 242 to move relative to housing 12, the metering fluid contained in metering chamber 346 must flow through orifice 358 between upper and lower metering chamber portions 354 and 356.

20 The metering piston 348 and associated structure are shown only in a schematic fashion in FIGS. 1G and 1H.

The lower end of valve mandrel 242 has a lower adapter 360 connected thereto at threaded connection 362 with a seal being provided therebetween resilient O-ring seal means 364.

25 As previously mentioned, the valve mandrel 242 has the bypass valve port 320 disposed therethrough. Valve mandrel 242 also has

an isolation port 366 disposed therethrough communicating the flow passage 308 with annular cavity 272 of compression chamber 232. The isolation port 232 may also be considered to be a portion of the fill passage means 234.

5 The valve mandrel 242 is shown in FIGS. 1G-1H in the position which it is in when apparatus 10 is run into a well. The isolation port 366 is located below seal 328 and provides open communication between the flow passage 308 and the compression chamber 232.

10 Also, the bypass valve port 320 is located below seal 334 and provides open communication between flow passage 308 and annular cavity 318 and bypass port 316.

After the apparatus 10 is lowered to its desired location within a well, weight is set down on the apparatus 10 to set a
15 packer means 368 (see FIG. 3) located therebelow, and to close the isolation valve means 240 and the bypass valve means 314.

When weight is set down on the apparatus 10 for a sufficient period of time determined by the construction of the metering piston 348 and its orifice 358, the housing 12 moves longitudi-
20 nally downward relative to valve mandrel 242 until an upper end 370 of lower adapter 360 abuts a lower end 372 of lower housing shoe 136. When the ends 370 and 372 abut, the valve mandrel 242 is in a position relative to housing 12 such that isolation port 366 is located above seal 328 and bypass valve port 320 is
25 located above seal 334.

Thus, the isolation valve means 240 and the bypass valve means 314 are operatively associated so that they are both open at the same time and are both closed at the same time.

5 The fill passage means 234 can be further described as including a first portion 376 which is coincident with a portion of flow passage 308.

Fill passage means 234 further includes a second portion 378 which is coincident with bypass passage means 312 and thus communicates flow passage 308 with the well annulus 236.

10 The isolation valve means 240 can be described as being located between the compression chamber 232 and first portion 376 of fill passage means 234, and the bypass valve means 314 may be described as being located between the first and second portions 376 and 378 of fill passage means 234.

15 The apparatus 10 is constructed so that the compression chamber 232 has a total volume sufficiently large that an equal volume of water containing no absorbed gases would have a volume decrease when subjected to a pressure increase equivalent to a predetermined operating pressure of the actuating piston means 140, at least as great as the decrease in volume of the
20 compression chamber 232 when the actuating piston means 140 is moved between its first and second positions relative to the housing 12.

The majority of this volume of the compression chamber 232 is
25 defined within the main compression chamber portion 244 located

between the outer compression housing sections 78 and the inner compression housing sections 98.

The actual total volume required for the compression chamber 230 will depend upon a number of parameters.

5 First, the volume change which the compression chamber 232 must undergo when it is compressed, is equal to the displacement of actuating piston means 140 as it moves from its first upper position illustrated in FIG. 1C to a lower second position corresponding to an open position of ball valve member 162. This
10 displacement is equal to the annular area of actuating piston 140 defined between seals 144 and 152, multiplied by the longitudinal displacement of actuating piston means 140 as it moves downward between its first and second positions.

 The required total volume for compression chamber 232 to
15 accommodate that displacement of actuating piston means 140 depends upon a number of parameters.

 First, it is dependent upon the particular well fluid which will be involved, and particularly it is dependent upon the compressibility factor of that well fluid. Generally, the well
20 fluid involved will be drilling mud, which normally has a compressibility factor at least as great as that of water which has no absorbed gases contained therein.

 Thus, the compression chamber 232 of the apparatus 10 of the present invention is designed based on the assumption that the
25 well fluid will have a compressibility factor equal to that of water with no absorbed gases under the appropriate conditions.

Those appropriate conditions which also must be taken into consideration include the operating temperature of the well at the particular depth involved, the hydrostatic pressure within the well annulus 236 at the depth at which it is desired to operate the apparatus 10, and the operating pressure which permissibly can be applied across the actuating piston means 140.

The operating temperature and the hydrostatic pressure are of course determined by the particular location in the well where it is necessary to utilize the apparatus 10, and thus they are fixed parameters in a given well situation.

The operating pressure which can permissibly be applied across actuating piston means 140 depends upon the maximum pressure increase which permissibly can be applied to the well annulus 236.

It is contemplated with the apparatus 10 of the present invention that it will normally be designed for an operating pressure in the range of about 1500 to 2000 psi (10.3 to 13.8 MPa). That is, the well annulus 236 will be pressurized to a pressure 1500 to 2000 psi (10.3 to 13.8 MPa) greater than the hydrostatic pressure normally present within the annulus.

This permissible operating pressure is based upon the strength of the well casing and the tubular members connected to the tubing string lowered into the well casing. It must be sufficiently low that it does not rupture the well casing or collapse the tubular members immersed within the well.

Thus, given the desired operating pressure of 1500 to 2000 psi
(10.3 to 13.8 MPa) which will be applied downwardly across
the actuating piston 140, and given the compressibility factor
for water with no absorbed gases at the operating temperature
and initial hydrostatic pressure of the well annulus 236, the
required volume of compression chamber 232 can be calculated.

As an example of the appropriate volume of compression
chamber 232 for a given displacement of actuating piston means
140, one particular design for the apparatus 10 involves an
actuating piston displacement of 14.68 cubic inches (0.23 dm³)
when the actuating piston moves between its first and second
positions. For that displacement, the compression chamber 232 must
have a volume of at least approximately 8,500 cubic inches (136 dm³).

With that particular ratio of displacement to compression
chamber volume, the apparatus 10 can operate in most commonly
encountered well environments.

In actual practice, the provision of a compression chamber
having a volume determined as described above, will provide a
significant margin of safety. This is because the chamber 232
will always contain some amount of air which is trapped therein
during assembly of the apparatus 10, and additionally, the
drilling mud normally used as well fluid will have a significant
amount of gas therein. Both of these sources of trapped gas pro-
vide additional compressibility of the total volume of fluid
which is contained within the compression chamber 232.

First Mode Of Operation Of The Embodiment Of FIGS. 1A-1H

A first mode of operation of the embodiment of apparatus 10 shown in FIGS. 1A-1H will now be described. In this first mode of operation, no pressurized gas is provided in the compression chamber 232, and during the operation of the apparatus 10, the apparatus 10 relies upon the compressibility of trapped well fluid within the compression chamber 232 as the primary source of fluid compressibility to provide a compressible fluid spring.

The apparatus 10 is assembled and initially oriented in the manner shown in FIGS. 1A-1H.

During the initial assembly of apparatus 10, the compression chamber 232 is communicated with the surrounding ambient air at the surface 388 of the well through the fill passage means 234, and thus the compression chamber 232 will initially be filled with air at the atmospheric pressure and ambient temperature prevailing at the surface location 388 of the well.

The upper adapter 14 of housing 12 is threadedly connected to a tubing string 380 as shown in FIG. 3.

The lower adapter 360 is threadedly connected to a packer housing 382 of the packer means 368.

A perforated tail pipe 384 is connected to the lower end of the packer housing 382.

The tubing string 380 with the apparatus 10 and other apparatus just described attached as shown in FIG. 3 is lowered into the well defined by well casing 238.

It is noted that FIG. 3 illustrates the apparatus 10 connected to the tubing string 380, after the apparatus 10 has been partially lowered into the well defined by well casing 238. Those portions of FIG. 3 near the upper end thereof relating to the rig air source 294 and rig air supply line 292 are not utilized in this first mode of operation of the apparatus 10, and thus they may be ignored for the purposes of the present description of this first mode of operation.

The apparatus 10 is lowered on the tubing string 380 into the well 238 to a desired location. Normally this location will be such that the packer means 368 is located immediately above a subsurface formation (not shown) intersecting the well 238, so that the packer 368 may be set to allow the subsurface formation to be tested through the tubing string 380 or to allow treatment fluids to be pumped down the tubing string 380 into the subsurface formation located immediately below the packer means 368.

As the apparatus 10 is lowered on the tubing string 380 into the well, open flow fluid communication is provided through fill passage means 234 between the compression chamber 232 and the well annulus 236. Thus, well fluid flows from the well annulus 236 through the fill passage means 234 into the compression chamber 232 as the apparatus 10 is lowered into the well.

During this lowering process, the pressure of the well fluid at a given elevation within the compression chamber 232 will be substantially equal to the hydrostatic pressure of well fluid in well annulus 236 at that same elevation.

The well fluid entering compression chamber 232 will contact the air already present in compression chamber 232 at an air-well fluid interface 386. As well-air fluid interface 386 moves upward through the compression chamber 232 as the apparatus 10 is lowered deeper into the well 238, the air initially present within the compression chamber 232 will be compressed into a relatively small volume located above air-well fluid interface 386.

As the air-well fluid interface 386 moves upward through the main compression chamber portion 244, which may also be referred to as a diametrically irregular annular space 244, the air-well fluid interface 386 moves past a number of upset diameters on both the inner surfaces of the outer compression housing sections 78 and the outer surfaces of the inner compression housing sections 98.

The pressure of the air trapped within compression chamber 232 above the air-well fluid interface 386 will be equal to the hydrostatic pressure of the well fluid within well annulus 236 at the elevation of the air-well fluid interface 386.

Since there is only a relatively small mass of air trapped within the compression chamber 232, which is equal to the mass of air required to fill the chamber 232 at atmospheric pressure and ambient temperature, that air will be compressed to a negligible, very small volume when the apparatus 10 is located at conventional depths on the order of many thousands or perhaps tens of

thousands of feet within a well, and thus the compression chamber 232 is usually substantially completely filled with well fluid by the time the apparatus 10 has been completely lowered to its desired final location within the well 238.

5 Once the apparatus 10 is at its desired location within the well 238, the tubing string 380 is manipulated to set the packer means 368 so that it seals against the casing 238 thus anchoring the tubing string 380 and the apparatus 10 relative to the casing 238. A preferred tool for use as the packer means 368 is a
10 packer made by Halliburton Services Division of Halliburton Company, the assignee of the present invention, and referred to as a "Retrievable Test-Treat-Squeeze Packer" as shown at pages 4014-4015 of Halliburton Services Sales and Service Catalog, No. 41. This particular packer is set by setting down weight on the
15 packer and applying right-hand torque to the tool string 380.

As part of this manipulation, or thereafter, weight is slacked off on tubing string 380 for a sufficient period of time to allow the housing 12 to move downward relative to valve mandrel 242 thus closing the bypass passage means 312 and also
20 closing the fill passage means 234 thus trapping well fluid within the compression chamber 232.

This trapped well fluid will initially be at a pressure equal to the hydrostatic pressure within the well annulus 236 at the same elevation at the time the fluid was trapped,

25 Then, fluid pressure is increased within the well annulus 232 by a pump (not shown) located at the surface and communicated

with the annulus 236 in a manner well known to those skilled in the art. This increased well annulus pressure is communicated through the power port 280 of power passage means 276 to the upper side 278 of actuating piston means 140, thus creating a
5 downward pressure differential across actuating piston means 140.

As previously mentioned, it is contemplated that the apparatus 10 will typically be designed for an operating pressure in the range of approximately 1500 to 2000 psi (10.3 to 13.8 MPa).

This downward pressure differential of approximately 1500 to
10 2000 psi causes the actuating piston means 140 to move downward relative to housing 12, thus moving the ball valve member 162 to its open position and allowing fluid to flow from the subsurface formation upward through the perforated tail pipe 384, the central bore (not shown) of packer housing 382, the flow passage
15 308 of apparatus 10, and the tubing bore (not shown) of tubing string 380.

As the actuating piston means 140 moves downward relative to housing 12, the well fluid trapped within compression chamber 232 is compressed and a volume thereof is decreased.

20 In those situations where the compression chamber 232 is substantially completely filled with well fluid, a first amount by which the volume of the trapped well fluid is decreased is approximately equal to a second amount by which the entire volume of the compression chamber 232 is decreased. This second amount
25 is of course equal to the displacement of the actuating piston means 140.

When utilizing the apparatus 10 in the manner just described wherein the only gas originally present in the compression chamber 232 is air at atmospheric pressure and temperature, and when using the apparatus at typical depths within a well, the
5 mass of air present within the compression chamber 232 is so small that it contributes very little to the volume compression required to accommodate the displacement of actuating piston 140. That is, the volume change which occurs within the compression chamber 232 must primarily be accounted for by a decrease in
10 volume in the well fluid contained within compression chamber 232, and only a relatively small amount of that volume decrease is accommodated by compression of the air which might be trapped in the compression chamber 232. This is because the amount of air is so small that it has already been compressed to a very
15 small volume as the apparatus 10 is lowered into the well.

Thus, it can generally be stated that when the apparatus 10 is operated in its normal manner just described above, and when the actuating piston means 140 moves between its first and second positions, the volume of the well fluid trapped within the
20 compression chamber 232 changes by an amount greater than one-half of the amount by which the total volume of the compression chamber changes. Thus, the apparatus 10, when used in the manner just described, relies primarily upon the compressibility of the well fluid to accommodate the displacement of the actuating
25 piston 140. A majority of the total amount of fluid pressure

energy which is stored within the various fluids trapped in the compression chamber is thus stored by compression of the trapped well fluid.

After the well testing procedure or well treatment procedure is completed, the increased well pressure previously applied to well annulus 236 is released so that the well annulus 236 returns to hydrostatic pressure. This eliminates the downward pressure differential across actuating piston means 140, and the trapped fluids, particularly the trapped well fluid, within compression chamber 232 expand thus forcing the actuating piston 140 back up to its initial position as shown in FIG. 1C corresponding to the closed position of ball valve means 140 as seen in FIG. 1A.

This upward movement of actuating piston means 140 is aided by the coil compression spring 258. Preferably, the coil compression spring 258 is sufficiently sized so that in most situations it could return the actuating piston means 140 to its first position thus closing the ball valve 162 even in the absence of the upward force from the compressed fluid within compression chamber 232.

Then, through appropriate manipulation of the tubing 380, the packer means 368 is released from its engagement with the well casing 238 and the apparatus 10 may be removed from the well or moved to another location within the well.

Also, during this manipulation of the tubing string 380, the housing 12 of apparatus 10 is returned to its upward position

relative to valve mandrel 242 so that the fill passage means 234 and bypass passage means 312 are both again opened.

Second Mode Of Operation Of The Embodiment Of FIGS. 1A-1H

5 There is also an alternative manner in which the apparatus 10 of FIGS. 1A-1H may be utilized when it is desired to lower the operating pressure which must be applied to the well annulus 236 to operate the apparatus 10. In this second mode of operation, the compression chamber 10 is initially injected with relatively
10 low pressure pressurized air. The mass of injected air is sufficient that when operated in this second mode, the apparatus 10 generally relies primarily on compression of the air rather than compression of the trapped well fluid to accommodate the displacement of actuating piston 140.

15 This is best understood with reference to FIGS. 2 and 3.

 The apparatus 10 as shown in FIG. 3 after it has been partially lowered into the well 238 to an intermediate position wherein a lower portion of the apparatus 10 is within the well 238 and is immersed in well fluid as seen in FIG. 3, and an upper
20 portion of the apparatus 10 still extends above a ground surface 388 at the well location.

 First, it should be noted that when utilizing the apparatus 10 in its second mode, wherein pressurized air is to be injected into the compression chamber 232, the annular floating piston 260
25 within the lubricating chamber 252 will generally be eliminated,

and thus there will be no lubricating fluid within the lubricating chamber 252 or the spring chamber 256.

Then, the rig air system 294, which is a source of compressed air normally at a pressure in the range of 100 to 140 psi, is
5 connected through the rig air line 292 to the inlet 288 of air injection passage means 286.

Then, the needle valve 296 is opened and a valve 390 in the rig air line 292 is opened so that the compressed air from the rig air source 294 is communicated with the longitudinal pas-
10 sageway 254 of compression chamber 232.

Remembering that the fill passage means 234 is open, it will be understood that prior to connection of the rig air source 294 to the compression chamber 232, well fluid will have risen into the compression chamber 232 so that the air-well fluid interface
15 386 is at some intermediate location as shown in FIG. 3.

Once the compressed air from the rig air source 294 at a pressure in the range of 100 to 140 psi is applied to the compression chamber 232, however, all of the well fluid then present within the compression chamber 232 will be forced downward
20 out through the fill passage means 234 and back to the well annulus 236 or at least into the flow passage 308.

Thus, the compressed air which remains within the compression passage 232 when the apparatus 10 is located as shown in FIG. 3 and the rig air source 294 has been communicated with the
25 compression passage 232 will be at an initial pressure substan-

tially equal to a hydrostatic pressure of well fluid within the well annulus 236 at the elevation of a lower end of the compression chamber 232, which in the embodiment illustrated in FIGS. 1A-1H is the elevation of isolation port 366. For an
5 apparatus 10 as shown in FIG. 3 having eight stacked outer compression housing sections 78, each ten feet (3.0m) long, the initial air pressure in chamber 232 will be approximately 50 psi (345 kPa).

After the compression chamber 232 has been filled with air, the needle valve 296 is closed and the rig air line 292 is
10 removed.

Then the apparatus 10 may be further lowered into the well to its final desired location.

Of course, as the apparatus 10 is further lowered into the well toward its final desired location, the gas-well fluid inter-
15 face 386 will again move upward through the diametrically irregular elongated space defined by main compression chamber portion 244 past a number of irregular upset diameters defined by the various portions of the inner and outer tubular housing sections 98 and 78.

20 Using this alternative manner of operation of the apparatus 10, wherein the compression chamber 232 is filled with compressed air from the rig air source 294, a sufficient mass of air is present within the compression chamber 232 such that in many operating situations the displacement of actuating piston means
25 140 will be accommodated primarily by compression of the air rather than compression of the trapped well fluid.

The injection of such a mass of compressed air into compression chamber 232 will lower the required operating pressure which must be applied to well annulus 236 to about 1000 psi (6.9 MPa) as compared to the 1500 to 2000 psi (10.3 to 13.8 MPa) previously mentioned for the normal manner of operation of apparatus 10 where no air is injected.

Detailed Description Of The Embodiment Of FIGS. 4F-4I

The lower portion of an alternative embodiment of the downhole tool apparatus of the present invention is illustrated in FIGS. 4F-4I, and is generally designated by the numeral 400. The upper portion of the downhole tool apparatus 400 is identical to the structure shown in FIGS. 1A-1E, except that the annular floating piston 260 shown in FIG. 1D is eliminated in order that the air injection passage means 286 may be utilized to provide a charge of compressed air to the compression chamber 232 from the rig air system 294 in a manner similar to that previously described with regard to FIG. 3.

The downhole tool apparatus 400 preferably has four outer tubular compression housing sections 78 and four inner tubular compression housing sections 98, each approximately ten feet (3m) long, so that the main compression chamber portion 244 of apparatus 400 will have a length of approximately forty feet (12m).

As seen near the bottom of FIG. 4F, the housing of the apparatus 400 as compared to that of the apparatus 10, is

modified below the lowermost outer compression housing section 78.

5 An upper outer check valve housing adapter 402 is threadedly connected at 404 to the lowermost outer tubular compression housing section 78 with a seal being provided therebetween by resilient O-ring seal means 406.

10 An outer check valve housing 408 has its upper end threadedly connected to the lower end of upper outer check valve housing adapter 402 at threaded connection 410 with a seal being provided therebetween by resilient O-ring seal means 412.

A lower end of outer check valve housing 408 is connected at threaded connection 414 to a modified tube receiving adapter 416 with a seal being provided therebetween by resilient O-ring seal means 418.

15 The lower end of modified tube receiving adapter 416 is connected at threaded connection 420 to the valve adapter 126 previously described with regard to the apparatus 10, with a seal being provided therebetween by resilient O-ring seal means 422.

20 Modified tube receiving adapter 416 of the apparatus 400 is similar in function to the tube receiving adapter 114 previously described with regard to FIG. 1G, and has the longitudinal passageways 270 of compression chamber 232 disposed therethrough.

25 The valve adapter 126 and other associated structure shown in FIGS. 4H-4I is identical to the similar structure shown in FIGS. 1G-1H, and is designated by the same numerals previously used in FIGS. 1G and 1H.

A lower end of a lowermost inner tubular compression housing section 98 has its external cylindrical surface 110, as seen in FIGS. 4F and 4G, closely received within a cylindrical internal surface 424 of an upper inner check valve housing adapter 426 with a seal being provided therebetween by resilient O-ring seal means 428.

An inner check valve housing 430 has its upper end threadedly connected to upper inner check valve housing adapter 426 at threaded connection 432.

10 A lower portion of an external cylindrical surface 434 of inner check valve housing 430 is closely and slidably received within an upper inner bore 436 of modified tube receiving adapter 416 with a seal being provided therebetween by resilient O-ring seal means 438.

15 Between the outer check valve housing 408 and the inner check valve housing 430, there is defined an annular check valve chamber portion 440 of compression chamber 232. The annular check valve chamber portion 440 is communicated at its lower end with the longitudinal passageways 270 disposed through modified tube receiving adapter 416, and is communicated at its upper end with annular spaces 442 and 444 of compression chamber 232.

The upper inner check valve housing adapter 426 is constructed in a fashion similar to that for the upper inner compression housing adapter 100 previously described, and includes a notch 446 in its lower end which engages an upward

facing shoulder 448 of upper outer check valve housing adapter 402. The notches 446 allow fluid to flow between the upper inner check valve housing adapter 446 and the upper outer check valve housing adapter 402.

5 Disposed in the annular check valve chamber portion 440 is a check valve means 450 for initially sealing pressurized gas within the compression chamber 232 at an initial pressure greater than a pressure in fill passage means 234, and for subsequently
10 permitting well fluid to flow through the fill passage means 234 into the compression chamber 232 to directly contact the pressurized gas within compression chamber 232 when the pressure in the fill passage means 234 exceeds an initial pressure of the pressurized gas within the compression chamber 232.

15 The check valve means 450 is an annular valve ring, or may also be referred to as an annular valve piston, slidably disposed in the annular check valve chamber portion 440 of compression chamber 232.

20 The check valve ring 450 includes a pair of annular inner O-ring seals 452 which provide a sliding seal between check valve ring 450 and inner check valve housing 430. Check valve ring 450 also includes a pair of outer O-rings 454 which provide a sliding seal between check valve ring 450 and outer check valve housing 408.

25 Check valve ring 450 is shown in FIG. 4G in its initial position in which it is located when the compression chamber 232 has

been filled with an initial charge of pressurized air, before any well fluid has entered the compression chamber 232 to directly contact that pressurized air. In this initial position, the check valve ring 450 abuts an upper end 456 of modified tube receiving adapter 416. The upper end 456 may be referred to as a stop means 456, engaging the annular check valve ring 450, for preventing movement of the annular check valve ring 450 toward the fill passage means 234.

So long as the pressure of the pressurized air within compression chamber 232 above the annular check valve ring 450 exceeds pressure within the fill passage 234 which is communicated with the very lowermost portion of compression chamber 232 below annular check valve ring 450, the annular check valve ring 450 will remain in the position illustrated in FIG. 4G abutting the stop means 456.

As the apparatus 400 is lowered into a well, the well annulus pressure which is communicated with the lower end of check valve ring 450 through the fill passage means 234 will soon exceed the initial pressure of the pressurized air within compression chamber 232 above check valve ring 450, and then the check valve ring 450 will be pushed upward within the annular check valve chamber portion 440 by this upward pressure differential.

Check valve ring 450 includes a plurality of longitudinally extending radially outer grooves 458 in its exterior surface below outer O-rings 454, which grooves 458 may be referred to as fluid fill bypass grooves 458.

As the annular check valve ring 450 moves upward within annular check valve chamber portion 440, the lowermost outer O-ring 454 will move upward past an internal upset diameter 460 of outer check valve housing 408 so that the fill fluid bypass grooves 458
5 will be communicated with an enlarged internal diameter 462 of outer check valve housing 408, thus allowing well fluid to flow from fill passage means 234 past the annular check valve ring 450 and into the compression chamber 232 after the annular check valve ring 450 is moved a predetermined distance away from the
10 stop means 456. This allows the well fluid to directly contact the pressurized gas previously injected into compression chamber 232, at a gas-well fluid interface such as 386 illustrated in FIG. 3.

A latch means 464 is provided for latching annular check
15 valve ring 450 in a final position wherein the fluid fill bypass grooves 458 are communicated with enlarged internal diameter 462 of outer check valve housing 408, which may be defined as an open position of the fill fluid bypass grooves 458.

The latch means 464 includes a plurality of upward extending
20 spring collet fingers 466 attached to annular check valve ring 450, and includes an annular latching collar 468 which is threadedly connected at 470 to the lower end of upper outer check valve housing adapter 402.

After the annular check valve ring 450 moves upward a
25 distance sufficient that the fill fluid bypass grooves 458

thereof are communicated with the enlarged internal diameter 462 of outer check valve housing 408, a plurality of radially outward extending lugs 472, one of which is located on each of the spring collet fingers 466, snap over a radially inwardly extending annular latching lug 474 of latch collar 468, to latch the check valve ring 450 in its open position.

Manner Of Operation Of The Embodiment Of FIGS. 4F-4I

The downhole tool apparatus 10 previously described with regard to FIGS. 1A-1H, as previously mentioned preferably has eight interconnected tubular outer compression housing sections 78, each having a length of approximately ten feet (3.0m), so that the total length of the apparatus 10 is on the order of ninety feet (27m) or more.

Although a large conventional oil and gas drilling rig can handle a tool having a ninety-foot (27m) length, some smaller drilling rigs simply are not capable of handling a tool of that length.

The apparatus 400 of FIGS. 4F-4I preferably has only four of the interconnected tubular outer housing sections 78, each having a length of approximately ten feet (3.0m), so that the overall length of the apparatus 400 is more on the order of approximately fifty feet (15m). This shorter apparatus can be handled by many smaller drilling rigs which could not handle a ninety-foot (27m) tool.

By reducing the overall length of the apparatus, however, the volume of the compression chamber 232 is so reduced that this

volume is generally not large enough that the tool can be operated based upon compressibility of well fluid trapped in the compression chamber 232.

Also, it would not generally be possible to provide a sufficient mass of air within the compression chamber 232 of apparatus 400 to operate the tool based on air compression in a manner like that described for the second mode of operation of the apparatus 10. This is because the length of the compression chamber has been cut approximately in half, thus similarly cutting the pressure which could be provided by merely injecting air to force the fluid out of the compression chamber in half.

With this reduced length of the apparatus 400, as compared to the apparatus 10, it is necessary to provide a means for allowing pressurized air to be injected into the compression chamber 232 at an initial pressure greater than the pressure which will be present in the fill passage means 234 which is communicated with the well annulus exterior of the tool. This is provided by the check valve means 450 previously discussed.

The check valve means 450 allows the compression chamber 232 to be initially pressurized to a pressure in excess of the pressure within fill passage 234. Subsequently, as the tool is further lowered into a well, the check valve means 250 will allow well fluid to move upward into the compression chamber 234 to directly contact this mass of initially pressurized air, when the pressure within fill passage 234 exceeds the initial pressure of the pressurized air within compression chamber 232.

A more detailed description of the manner in which the downhole apparatus 400 operates will now be given.

The operation of the apparatus 400 can best be understood with reference both to FIGS. 4F-4I and FIG. 3. When referring to
5 FIG. 3, however, it must be remembered that the apparatus 400 has only four interconnected tubular outer compression housing sections 78, rather than the eight interconnected sections illustrated in FIG. 3.

The apparatus 400 is first partially lowered into the well
10 238 to an intermediate position wherein a lower portion of the apparatus 400 is within the well 238 and is immersed in well fluid in a manner like that illustrated in FIG. 3 for the apparatus 10. An upper portion of the apparatus 400 will extend above the ground surface 388, again in a manner similar to that shown
15 in FIG. 3 for the apparatus 10.

The apparatus 400 will have its annular check valve ring 450 initially positioned as shown in FIG. 4G. The check valve ring 450 has sufficient frictional engagement with the inner and outer check valve housings 430 and 408 that the upward pressure differential caused by the initial lowering of the apparatus 400
20 within the well 238 will not move the check valve ring 450 up to a latched position.

With the apparatus 400 in an initial partially lowered position similar to that shown in FIG. 3 for the apparatus 10, the
25 rig air system 294 is connected through the rig air line 292 to the inlet 288 of air injection passage means 286.

Then the needle valve 296 is opened and the valve 390 in rig air line 292 is opened so that the compressed air from the rig air source 294 is communicated with the longitudinal passageway 254 of compression chamber 232.

5 Then that portion of the compression chamber 232 above check valve ring 450, which of course is the vast majority volume-wise of the compression chamber 232, will be filled with pressurized air from the rig air source 294. The initial pressure applied to compression chamber 232 will preferably be in the range of
10 approximately 100 to 140 psi.

 This initial pressure will exceed the pressure then present in fill passage means 234.

 Then, the rig air source 294 is disconnected from compression chamber 232, and the apparatus 400 may be further lowered into
15 the well to its final desired location.

 As the apparatus 400 is further lowered from its partially lowered position like that shown in FIG. 3, the well annulus pressure which is present in fill passage means 234 and communicated with the lower side of annular check valve ring 450
20 will soon exceed the initial pressure of pressurized air within compression chamber 232 above annular check valve ring 450, so that the upward pressure differential causes annular check valve ring 450 to move upward until the fill fluid bypass grooves 458 are latched in an open position by latch means 464, thus per-
25 mitting the well fluid to flow through the fill passage means 234

upward past annular check valve ring 450 into the main portion 244 of compression chamber 232, so that the well fluid directly contacts the initial mass of pressurized air at a gas-well fluid interface such as 386 seen in FIG. 3.

5 The initial charge of pressurized air injected into compression chamber 232 above annular check valve ring 450 has a mass sufficient that when the fill passage means 234 is closed by isolation valve means 240, and the actuating piston means 140 moves between first and second positions thereof relative to the
10 housing 12 in order to operate the ball valve 162, a volume of said initial mass of compressed air changes by a first amount greater than one-half of a second amount by which a total volume of the compression chamber 232 changes.

15 Thus, a majority of the fluid pressure energy stored in compression chamber 232 when the actuating piston means 140 moves from its first to its second position, is stored in compression of the pressurized air, rather than in compression of the well fluid.

20 Summary Of Operation

 Thus, three basic modes of operation of the present invention have been disclosed and described.

 The first mode of operation is provided by the use of the apparatus 10 of FIGS. 1A-1H, without the injection of any
25 pressurized air into the compression chamber 232 thereof.

The second mode is provided by utilizing the apparatus 10 of FIGS. 1A-1H, in combination with the procedure of injecting pressurized air into the compression chamber 232 after the apparatus has been partially lowered into a well, so that the
5 initial pressure of the pressurized air within compression chamber 232 is equal to the hydrostatic pressure of a column of well fluid having a height substantially equal to a length of the compression chamber.

The third mode is provided by the modified shortened apparatus shown in FIGS. 4F-4I which utilizes the check valve ring 450
10 to initially hold a mass of pressurized air within the tool.

In each of these three embodiments, a compression chamber provides a trapped reference pressure for the actuating piston 140, and also provides a compressible fluid spring for aid in
15 returning the actuating piston 140 to its initial position.

In each instance, this compression chamber is a relatively large compression chamber which is to some extent filled with well fluid as the apparatus is lowered into the well, with that well fluid directly contacting any gas present within the chamber
20 at a gas-well fluid interface. This gas-well fluid interface moves upwardly through the elongated compression chamber past a number of irregular diameters thereof as the tool is lowered into the well.

The apparatus of the present invention, in its various embodiments, provides a downhole tool having a trapped reference
25

pressure and providing a compressible fluid spring, without the use of either an initial highly pressurized volume of gas such as that currently used in the prior art in pressurized nitrogen tools, and without the use of a large volume of silicone oil.

5 Instead, either the relatively low pressure rig air, or the well fluid itself, both of which are present at any well drilling site, are utilized as the working fluids within the compression chamber.

10 This provides an apparatus which can be utilized at remote well locations where compressed nitrogen or silicone oil might not readily be available. Furthermore, it provides a very much simplified apparatus and safer manner of operation than many of the prior art devices.

15 Thus, it is seen that the apparatus and methods of the present invention readily achieve the ends and advantages mentioned as well as those inherent therein. While certain preferred embodiments of the invention have been illustrated for the purposes of the present disclosure, numerous changes in the arrangement and construction of parts and steps may be made by those
20 skilled in the art.

CLAIMS:

1. A downhole tool apparatus, comprising: a housing (12), having a compression chamber (232) defined therein and having fill passage means (234) disposed through said housing for placing said compression chamber in open flow
5 fluid communication with a well annulus (236) exterior of said housing so that well fluid may flow into said compression chamber as said apparatus is lowered into a well, the compression chamber containing air or gas adapted to contact directly the well fluid at an air or gas/well fluid
10 interface within the chamber; isolation valve means (248) for selectively closing said fill passage means (234) to trap said air or gas and well fluid in said compression chamber (232); an operating element (162) disposed in said housing; an actuating piston means (140) slidably disposed
15 in said housing, said actuating piston means being operably associated with said operating element; wherein movement of said actuating piston means between first and second positions thereof decreases the volume of said compression chamber.

20

2. An apparatus as claimed in claim 1, wherein the volume of said compression chamber (232) is sufficiently large that an equal volume of water containing no absorbed gases would have a volume decrease when subjected to a
25 pressure increase equivalent to a predetermined operating pressure of said actuating piston means (140), at least as great as the decrease in volume of said compression chamber when said actuating piston means is moved between its first and second positions.

30

3. An apparatus as claimed in either claim 1, or claim 2, further comprising resilient mechanical spring biasing means (258), preferably disposed in said compression

chamber, for biasing said actuating piston means from its second position towards its said first position.

4. An apparatus as claimed in claim 3, further
5 comprising: an annular floating piston (260) disposed in said compression chamber and dividing said compression chamber into first and second chamber portions; wherein said resilient mechanical spring biasing means (258) is disposed in said first chamber portion (256); wherein said
10 first chamber portion (256) is filled with a relatively non-corrosive fluid and is isolated from open flow fluid communication with said second chamber portion by said floating piston (260); and wherein said fill passage means (234) communicates said second chamber portion with
15 said well annulus exterior of said housing.

5. An apparatus as claimed in any one of the preceding claims, wherein: a majority of the volume of said compression chamber is a diametrically irregular annular
20 space defined by a plurality of interconnected tubular outer housing sections (46,50,56,66,78,92,114) and by a plurality of interconnected tubular inner housing sections (60,72,98,102).

25 6. An apparatus as claimed in any one of the preceding claims, further comprising: a bypass passage means (312) disposed through said housing for communicating a flow passage (308) of said housing with said well annulus exterior of said housing; and a bypass valve means (314)
30 for selectively closing said bypass passage means, said bypass valve means preferably being operatively associated with said isolation valve means (240) so that they are both open at the same time and are both closed at the same time.

7. An apparatus as claimed in claim 6, wherein:
a first portion of said fill passage means (234) is
coincident with a portion of said flow passage (308);
a second portion of said fill passage means is coincident
5 with said bypass passage means (312); said isolation valve
means (240) is located between said compression chamber
(232) and said first portion of said fill passage means;
and said bypass valve means (314) is located between said
first and second portions of said fill passage means.
- 10 8. An apparatus as claimed in either claim 6 or claim
7, wherein: said isolation valve means (240) and said
bypass valve means (314) are both sleeve-type valves operated
by a single common valve mandrel sleeve (242) slidably dis-
15 posed in said housing; and said isolation valve means
and said bypass valve means are both adapted to be operated
by vertical reciprocation of a tubing string attached to
said apparatus.
- 20 9. An apparatus as claimed in any of the preceding
claims, wherein: said apparatus is a tester valve and
said operating element is a flow valve (162) disposed in
a flow passage (308) of said housing to selectively open
and close said flow passage.
- 25 10. An apparatus as claimed in any of the preceding
claims, wherein: said actuating piston means (140) is
a differential area piston means having a first side thereof
communicated with said compression chamber; and said
30 housing has a power passage means (276) disposed there-
through for communicating said well annulus exterior of
said housing with a second side of said actuating piston
means, so that said actuating piston means moves between its
said first and second positions in response to changes in
35 well annulus pressure external of said housing.

11. An apparatus as claimed in any of claims 1 to 10, further comprising: means (286) for introducing a gas into said compression chamber as or before said apparatus is lowered into said well.

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12. An apparatus as claimed in claim 11, further comprising: check valve means (450) for sealing initially said injected air or gas within said compression chamber (232) at an initial pressure greater than a pressure in
10 said fill passage means (234), and subsequently for permitting well fluid to flow through said fill passage means into said compression chamber to contact directly said pressurized gas or air when the pressure in said fill passage means exceeds said initial pressure of said
15 injected gas or air.

13. An apparatus as claimed in claim 12, wherein: said compression chamber has an annular chamber portion (440) defined between an outer cylindrical member and
20 an inner cylindrical member of said housing; said check valve means (450) comprises an annular valve ring slidably disposed in said annular chamber portion and sealingly engaging said outer and inner cylindrical members of said housing; said housing comprises a stop means (456),
25 engaging said annular valve ring, for preventing movement of said valve ring toward said fill passage means (234); and said check valve means (450) further comprises fill fluid bypass means (458,460), operatively associated with said annular valve ring, for allowing well fluid to flow
30 from said fill passage means past said annular valve ring and into said compression chamber after said annular valve ring is moved a predetermined distance away from said stop means (456).

14. An apparatus as claimed in claim 13, wherein:
said fill fluid bypass means is an upset diameter (460)
on one of said outer and inner cylindrical members of
said housing.

5

15. An apparatus as claimed in either claim 13 or
claim 14, further comprising: latch means (464) for
latching said annular valve ring in a final position
wherein said fill fluid bypass means (458, 460) is open.

10

16. An apparatus as claimed in any one of claims 11 to
15, wherein, in use, said compression chamber (232) contains
a mass of injected air or gas sufficient that when said
fill passage means is closed by said isolation valve
15 means (240) and said actuating piston means (140) moves
between said first and second positions, the volume of
said mass of gas or air changes by a first amount greater
than one-half of the amount by which the volume of said
compression chamber changes.

20

17. A method of operating a downhole tool having
an operating element, an actuating piston operatively
associated with said operating element, and a compression
chamber defined within said tool and in fluid pressure
25 communication with said actuating piston; the method comprising
the steps of: injecting pressurized gas into said compression
chamber at a location in said compression chamber such
that said injected gas will directly contact at a gas-well
fluid interface well fluid that flows into said compression
30 chamber; lowering said tool connected to a tubing string,
to a desired location in a well while providing open flow
fluid communication between said compression chamber and
a well annulus defined between said tubing string and a well
bore of said well; allowing well fluid from said well
35 annulus to flow into said compression chamber so that the
well fluid at a given elevation in said compression chamber

is at a pressure substantially equal to the hydrostatic pressure of well fluid in said well annulus at said elevation; trapping said well fluid in said compression chamber of said tool; moving said actuating piston
5 between first and second positions to operate said operating element and to decrease the volume of the compression chamber, thereby compressing said pressurized gas and well fluid trapped therein, the volume of said
10 pressurized gas being decreased by a first amount greater than one-half of the volume by which said compression chamber is decreased, so that a majority of a total of fluid pressure energy stored in said compression chamber is stored by compression of said pressurized gas.

15 18. A method as claimed in claim 17, wherein the pressurized gas is pressurized air, which may be provided from a rig air system of a drilling rig.

19. A method as claimed in either claim 17 or claim
20 claim 18, wherein: said compression chamber is elongate and has a fill passage communicating the annulus with a lower end thereof, and has said actuating piston communicated with an upper end thereof, wherein the tool is first partially lowered into said well to an intermediate
25 position wherein a lower portion of said tool is within said well and is immersed in well fluid and an upper portion of said tool still extends above the surface; and injecting said pressurized gas into said compression chamber until substantially all well fluid is forced out
30 of said compression chamber, so that said pressurized gas in said compression chamber is then at a pressure substantially equal to a hydrostatic pressure of well fluid in said well at an elevation the same as the elevation of said lower end of said compression chamber.

20. A method as claimed in either claim 17 or claim 18, further comprising the steps, prior to lowering the tool, of sealing said pressurized gas within said compression chamber but subsequently permitting well
5 fluid to flow into said compression chamber to contact directly said pressurized gas and form said gas-well fluid interface when well annulus pressure exceeds an initial pressure of said pressurized gas.

10 21. A method as claimed in claim 20, wherein: said compression chamber is elongate and said sealing step is characterized in that the pressure initially is greater than the hydrostatic pressure of a column of well fluid having a height equal to a length of said
15 elongate compression chamber.

22. A method as claimed in any one of claims 17 to 21, wherein: said actuating piston is moved by communicating a side thereof remote from the compression chamber
20 with said well annulus; and changing a pressure within said well annulus so that said actuating piston is moved in response to a pressure differential between said well annulus and said compression chamber.

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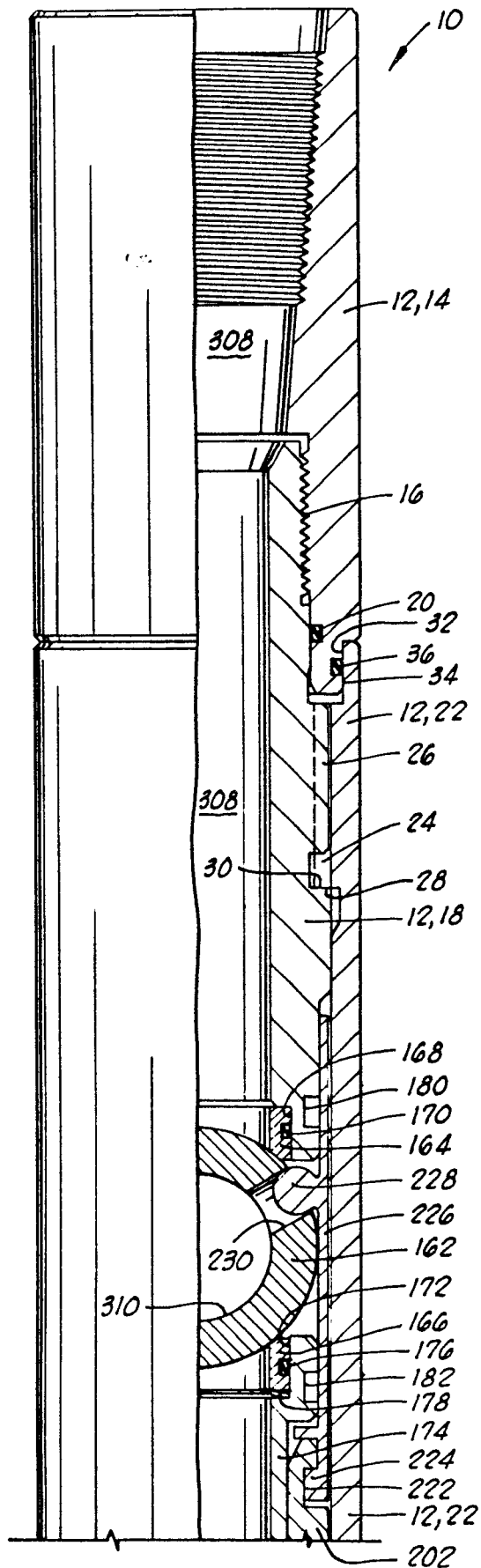


FIG. 1A

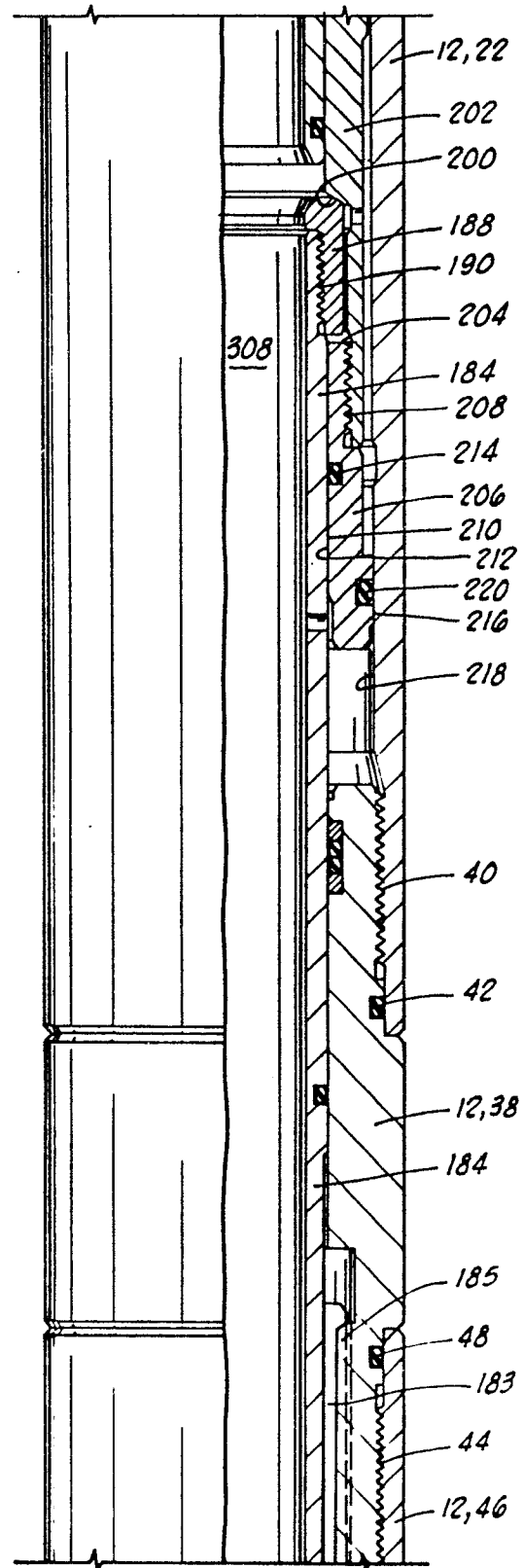


FIG. 1B

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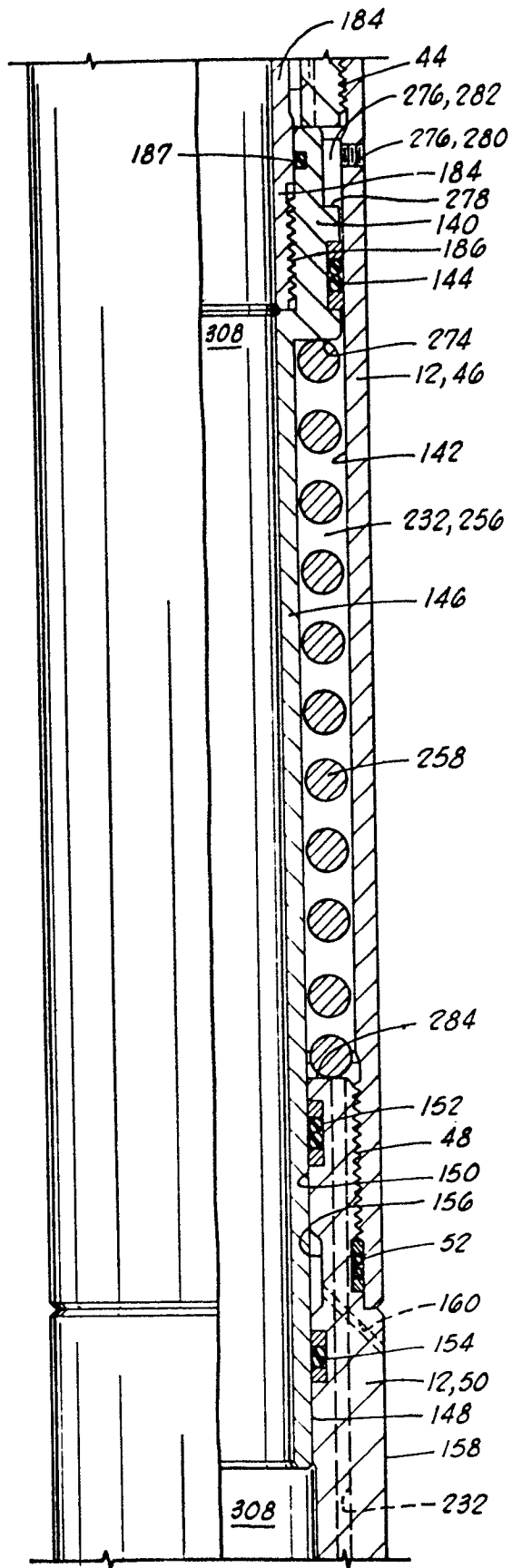


FIG. 1C

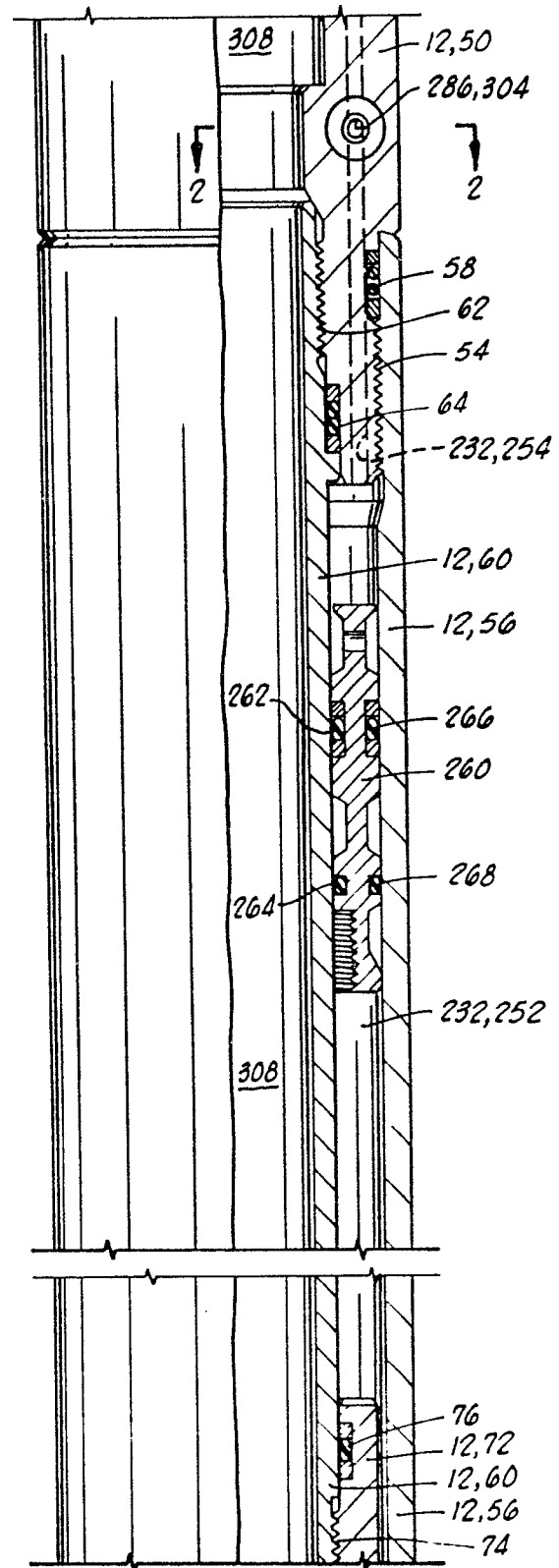


FIG. 1D

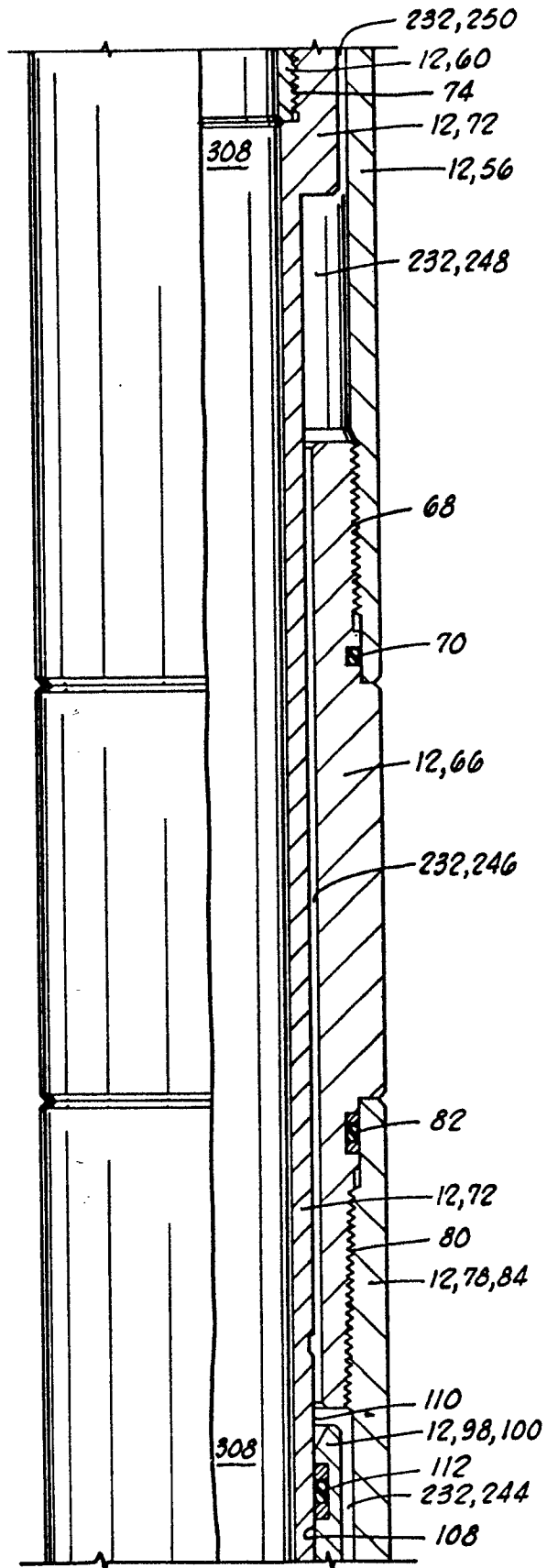


FIG. 1E

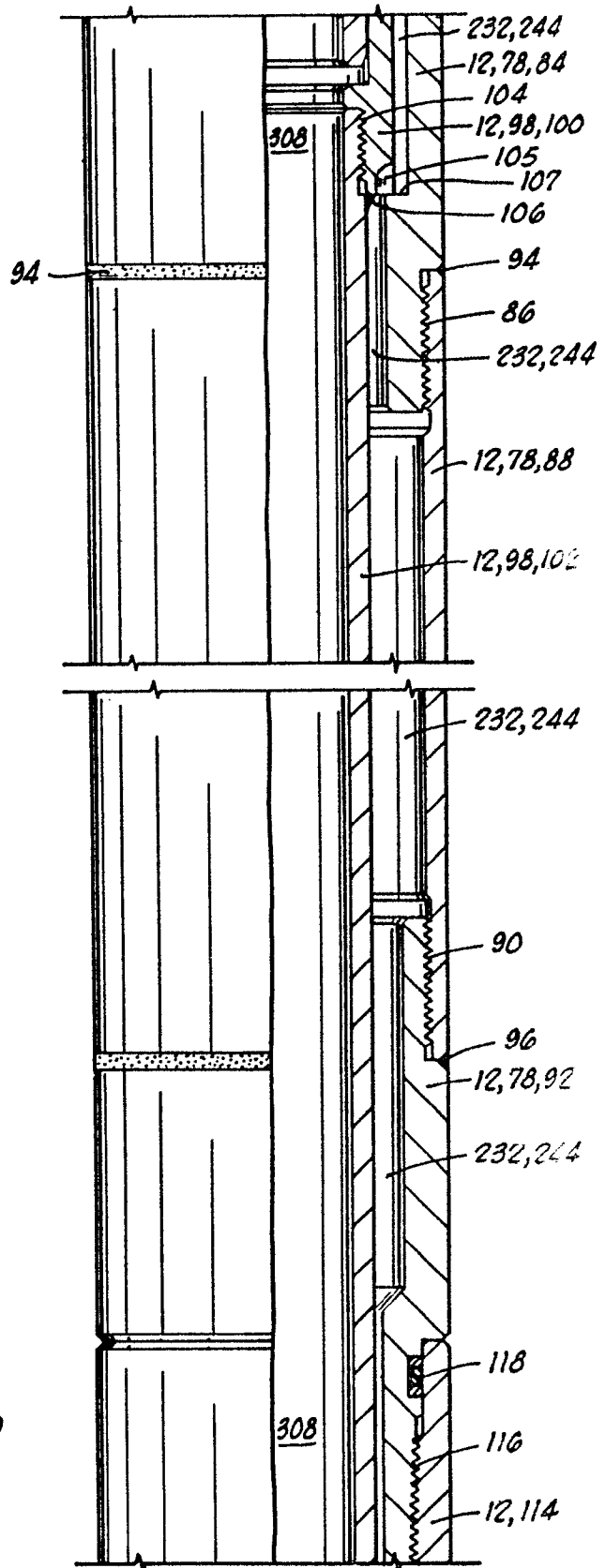


FIG. 1F

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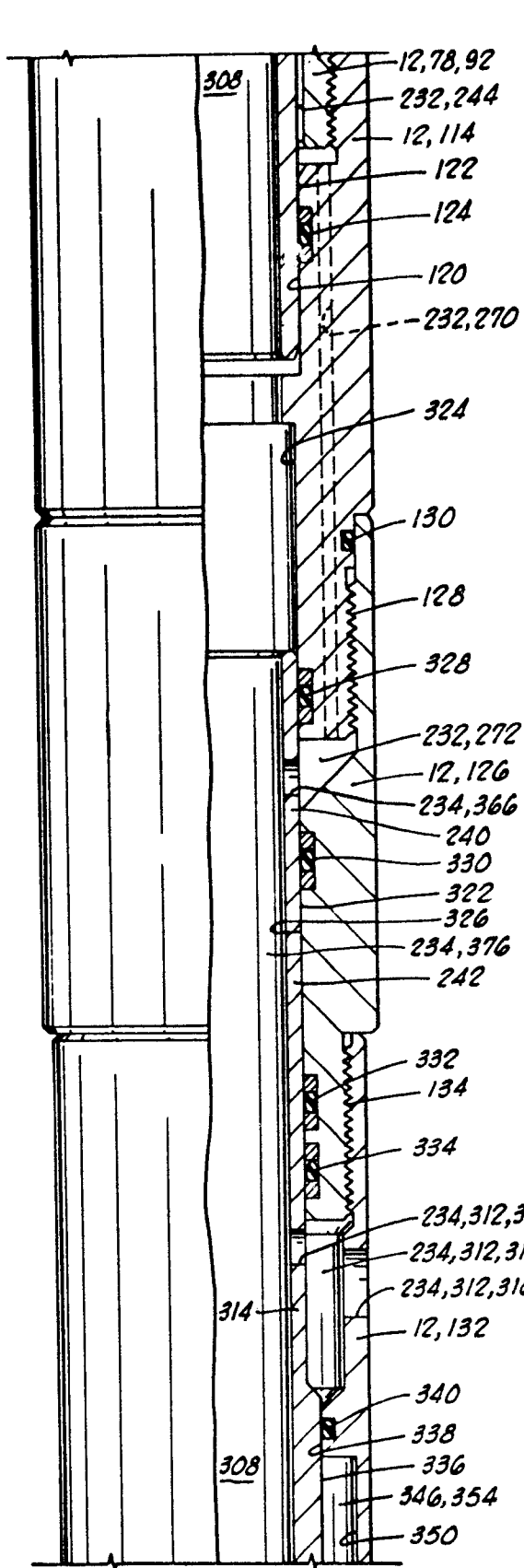


FIG. 1G

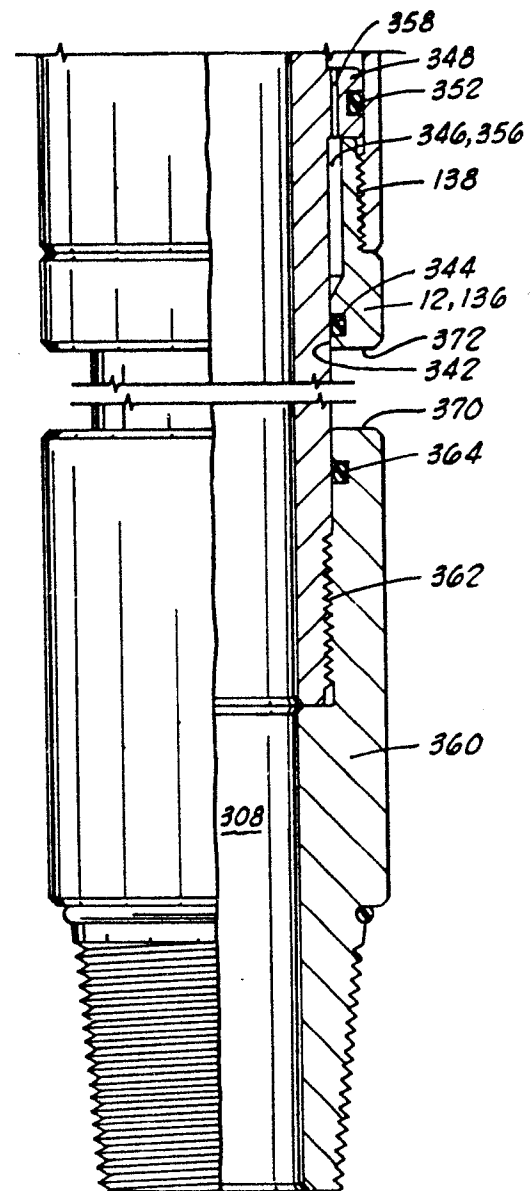


FIG. 1H

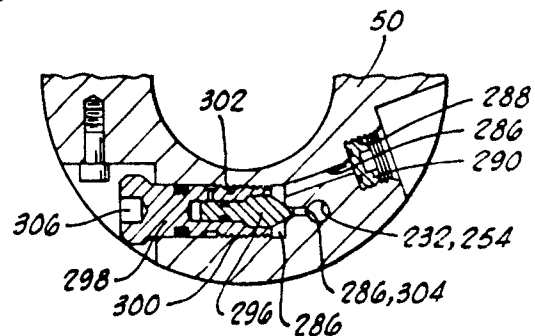


FIG. 2

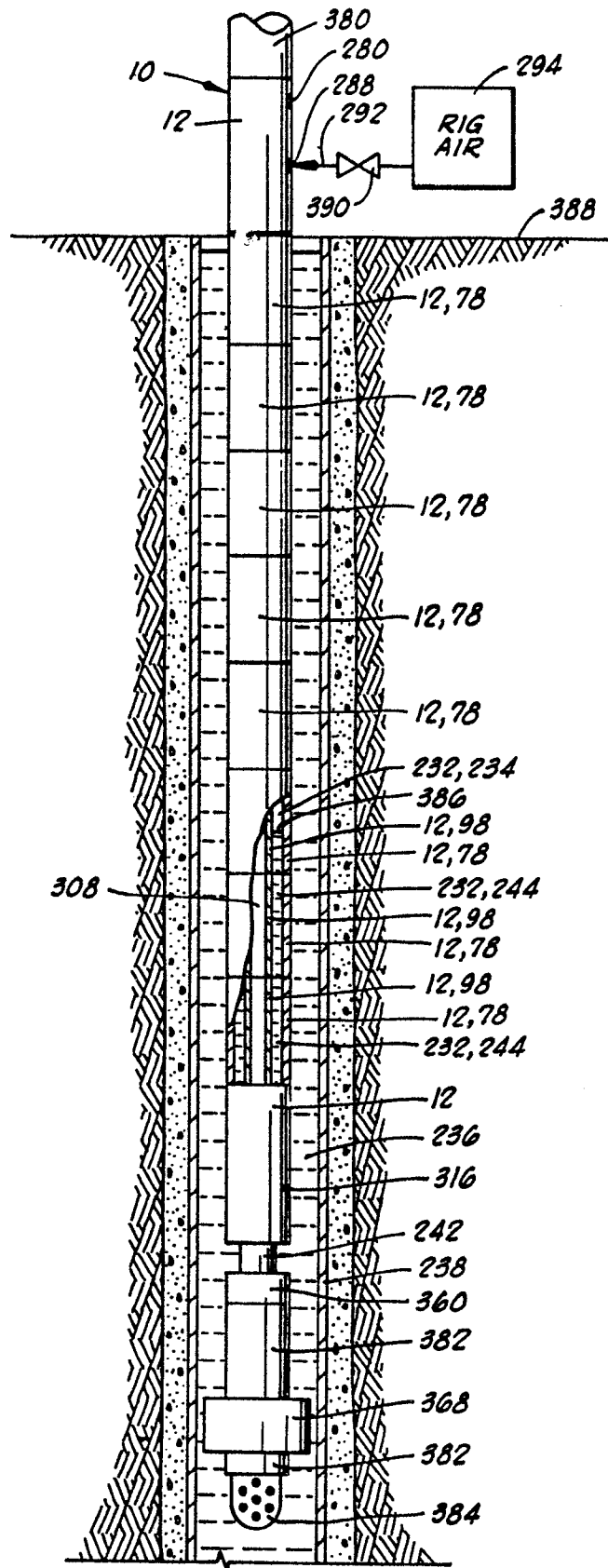


FIG. 3

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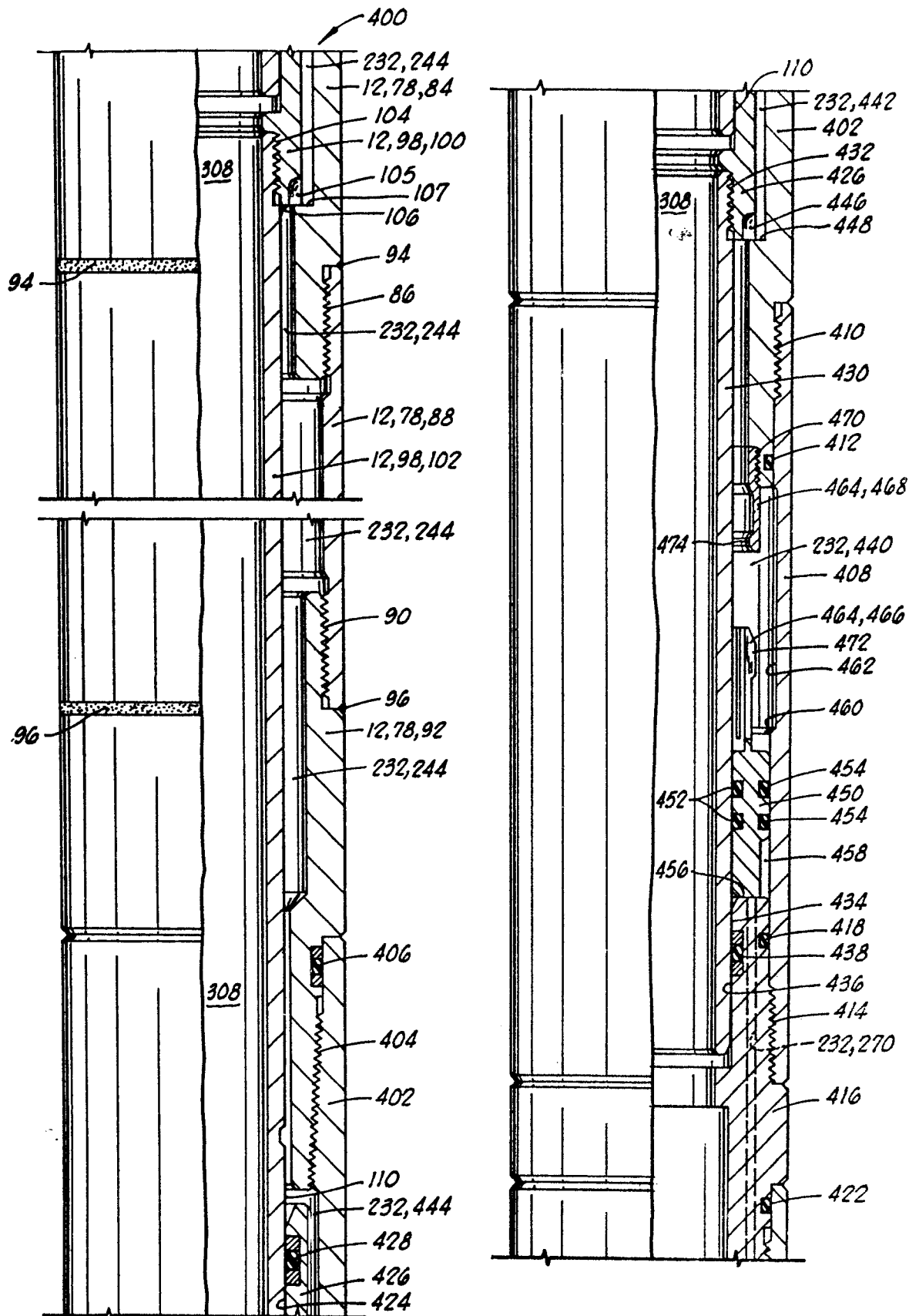


FIG. 4F

FIG. 4G

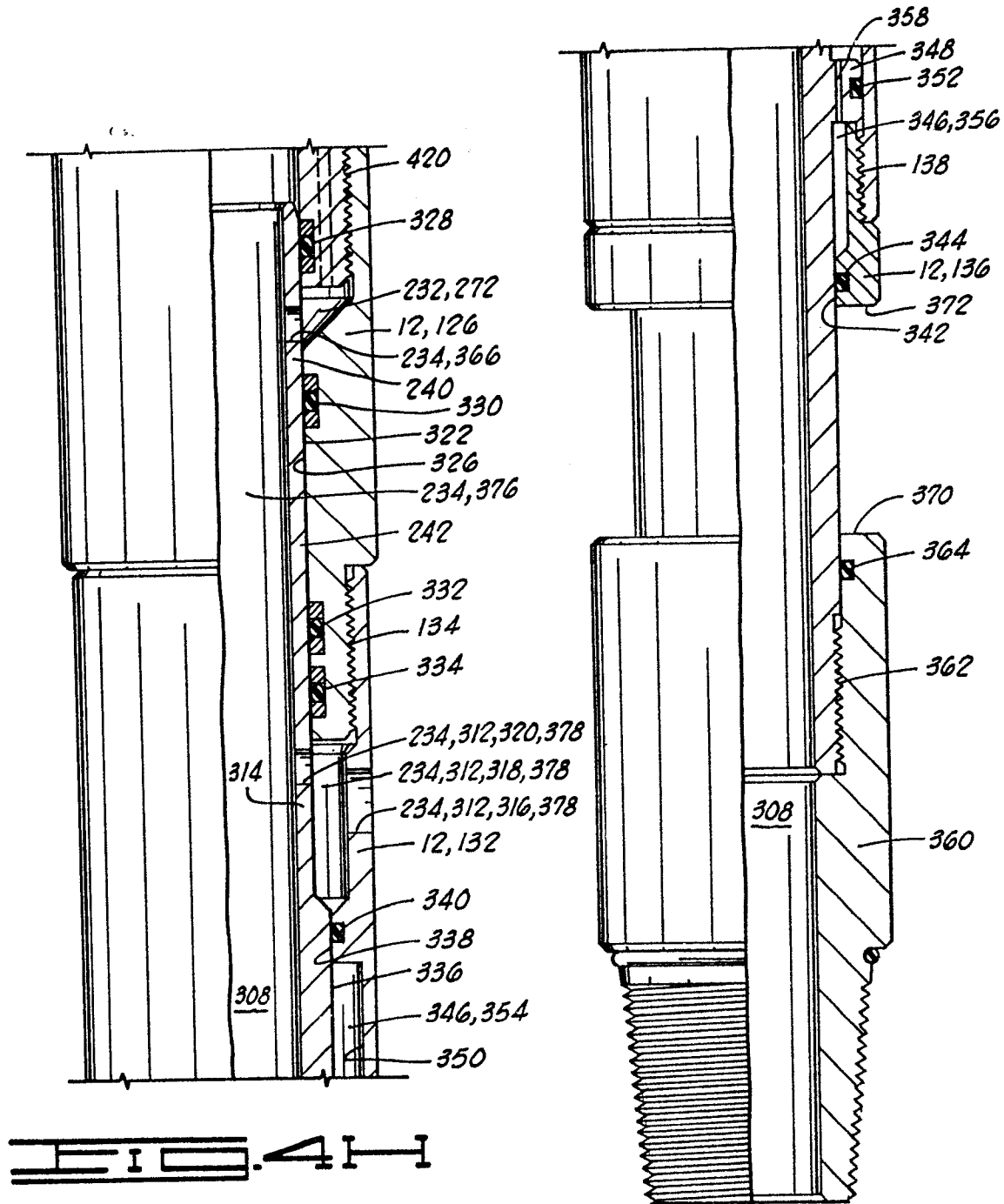


FIG. 4I